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Ventilation In Houses

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2

1

Build Tight - Ventilate Right

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Build Tight - Ventilate Right
How Tight?
What's Right?

3

**Air Barrier Metrics** 

Material 0.02 l/(s-m2) @ 75 Pa Assembly 0.20 l/(s-m2) @ 75 Pa Enclosure 2.00 l/(s-m2) @ 75 Pa 0.25 cfm/ft2 @ 50 Pa

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5

**Best** 

As Tight as Possible - with - Balanced Ventilation

**Energy Recovery** 

Distribution and Mixing

Source Control - Spot exhaust ventilation

Filtration

Material selection

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Getting rid of big holes 3 ach@50
Getting rid of smaller holes 1.5 ach@50
Getting German 0.6 ach@50

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6

Joseph Lstiburek 6

Worst

Leaky - with – Nothing

Spot Ventilation in Bathroom/Kitchen

Exhaust Ventilation – with – No Distribution and No Mixing

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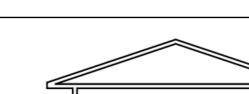
7

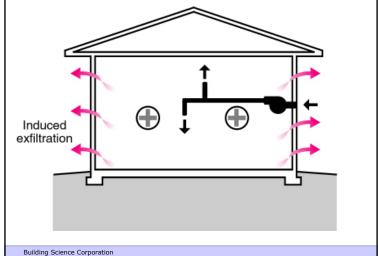
Three Types of Controlled Ventilation Systems

**Exhaust Ventilation** Supply Ventilation **Balanced Ventilation** 

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9



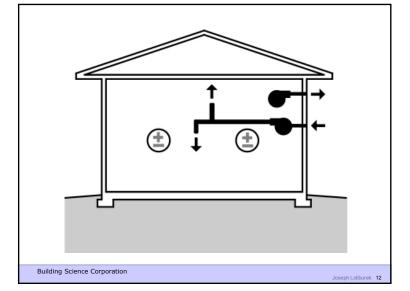


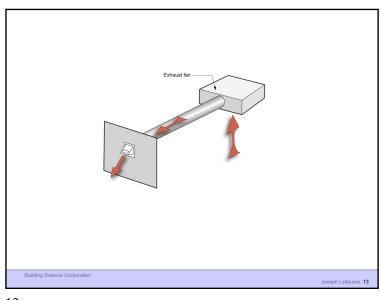
Building Science Corporation

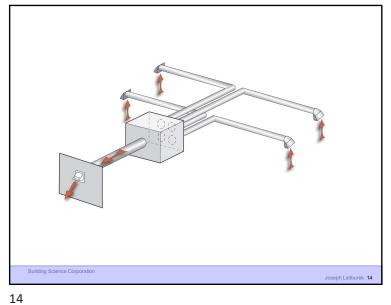
10

Induced

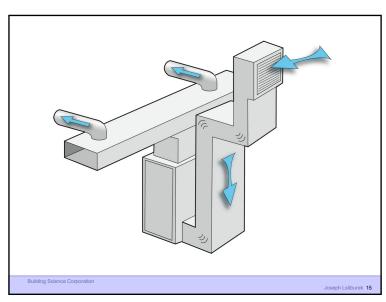
infiltration

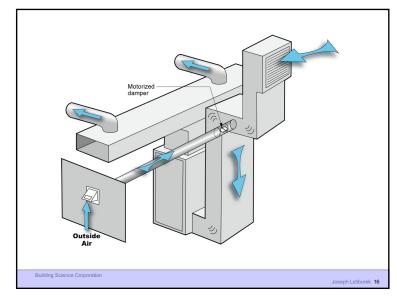




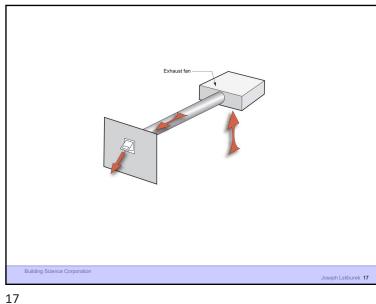


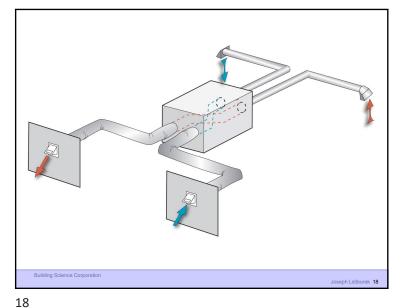
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15 16





Ventilation Rates Are Based on Odor Control Building Science Corporation

Ventilation Rates Are Based on Odor Control Health Science Basis for Ventilation Rates is **Extremely Limited** Building Science Corporation

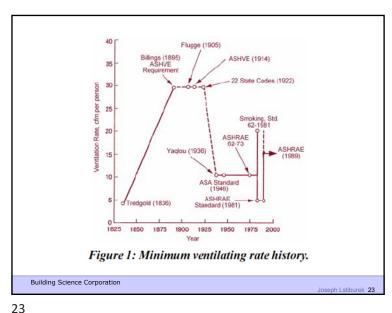
Joseph Lstiburek 22

Ventilation Rates Are Based on Odor Control Health Science Basis for Ventilation Rates is **Extremely Limited** Almost Nothing Cited Applies to Housing

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Joseph Lstiburek 21

21

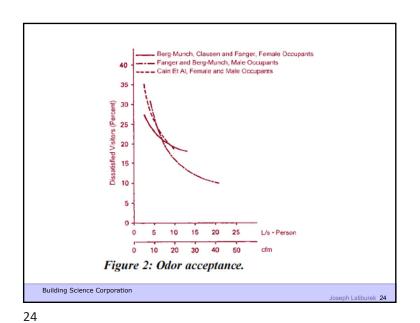


Ventilation Rates Are Based on Odor Control Health Science Basis for Ventilation Rates is **Extremely Limited** Almost Nothing Cited Applies to Housing

The Applicable Studies Focus on Dampness

22

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# House

2,000 ft<sup>2</sup> 3 bedrooms 8 ft. ceiling Volume: 16,000 ft<sup>3</sup>

.35 ach 93 cfm .30 ach 80 cfm .25 ach 67 cfm .20 ach 53 cfm .15 ach 40 cfm

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25

### Office

#### **Occupant Density**

15/1000 ft<sup>2</sup> (67 ft<sup>2</sup>/person) 62 - 89

15 cfm/person

5/1000 ft<sup>2</sup> (200 ft<sup>2</sup>/person) 62.1 - 2007

17 cfm/person

### **Correctional Facility Cell**

**Occupant Density** 

20/1000 ft<sup>2</sup> (48 ft<sup>2</sup>/person) 62.1 - 2007

10 cfm/person

#### House

2,000 ft<sup>2</sup> 3 bedrooms 8 ft. ceiling

Volume: 16,000 ft<sup>3</sup>

		\	/entilation Rates	
.35 ach	93 cfm	62 - 73	5 cfm/person	20 cfm
.30 ach	80 cfm		10 cfm/person	40 cfm
.25 ach	67 cfm	62 - 89	15 cfm/person	60 cfm
.20 ach	53 cfm		.35 ach	90 cfm
.15 ach	40 cfm	62.2 - 2010	7.5 cfm/person	50 cfm
			+ 0.01	
		62.2 - 2013	7.5 cfm/person	90 cfm
			+ 0.03	

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### C.P. Yaglou

Harvard School of Public Health

1936 1955

150 ft³ → 20 cfm/person

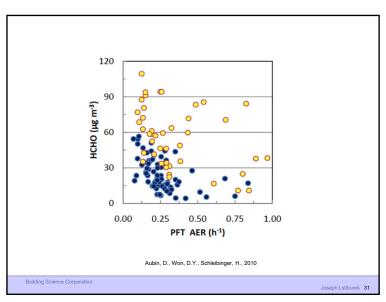
300 ft<sup>3</sup> → 12 cfm/person

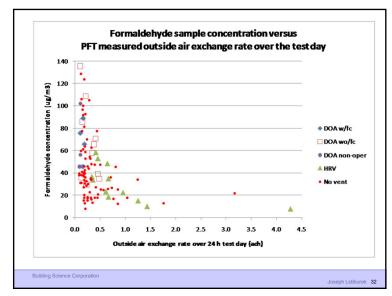
## C.P. Yaglou Harvard School of Public Health 1936 1955 150 ft³ → 20 cfm/person 18.75 ft<sup>2</sup> 106 occupants 300 ft³ → 12 cfm/person 37.5 ft<sup>2</sup> 53 occupants Experiment 470 ft<sup>3</sup> → 59 ft<sup>2</sup> 200 ft<sup>3</sup> -> 25 ft<sup>2</sup> 100 ft<sup>3</sup> → 12 ft<sup>2</sup>

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29





31

ASHRAE Standard 62.2 calls for 7.5 cfm per person plus 0.03 cfm per square foot of conditioned area

Occupancy is deemed to be the number of bedrooms plus one

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33

IRC 2015 and 2018 calls for 7.5 cfm per person plus 0.01 cfm per square foot of conditioned area

Occupancy is deemed to be the number of bedrooms plus one

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Joseph Lstiburek 35

ASHRAE Standard 62.2 calls for 7.5 cfm per person plus 0.03 cfm per square foot of conditioned area

Occupancy is deemed to be the number of bedrooms plus one

Outcome is often bad – part load humidity problems, dryness problems, energy problems

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3 Bedroom House – 2,500 ft2 30 cfm plus 75 cfm 105 cfm

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Joseph Lstiburek 3

3 Bedroom House – 2,500 ft2 30 cfm plus 25 cfm 55 cfm

37

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The Cult of The Blower Door

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3 Bedroom House – 2,500 ft2
30 cfm plus 25 cfm
55 cfm
With Balanced and Distributed 30 percent credit
38.5 cfm

38

Joseph Lstiburek 37

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Joseph Lstiburek 38

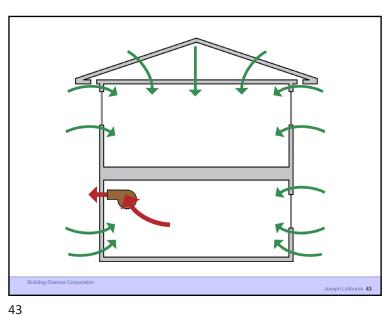


39

Blower Door Can't Get You The True ACH On A Short Term Basis – Hour, Day, Week

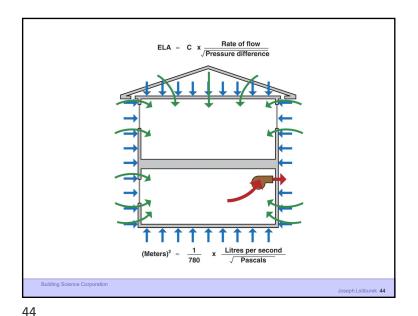
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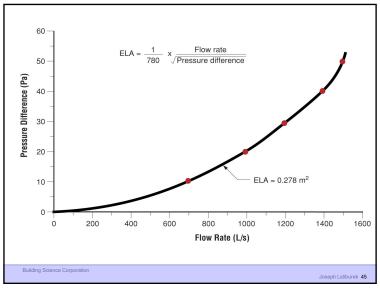
41



Don't Know Where The Holes Are Don't Know The Type of Holes Don't Know The Pressure Across The Holes

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Flow Rate (CFM)

Area of opening

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Joseph Lstiburek – Airflow 46

45

Flow Through Orifices
Turbulent Flow - "inertial effects"

Flow Through Porous Media
Laminar Flow - "viscosity effects"

Flow Through Orifices
 Turbulent Flow - "inertial effects"

Flow Through Porous Media
 Laminar Flow - "viscosity effects"

"true but not useful"

 $Q = A \cdot C_D \left[ \frac{2}{\rho} (\Delta P) \right]^{\frac{1}{2}}$   $Q = C_K \frac{\rho}{\mu} (\Delta P)$ Bernoulli

Darcy

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Joseph Lstiburek - Airflow 49

49

 $Q = A \cdot C_D \left[ \frac{2}{\rho} (\Delta P) \right]^{\frac{1}{2}}$   $Q = C_K \frac{\rho}{\mu} (\Delta P)$ Bernoulli

Darcy

 $Q = A \cdot C(\Delta P)^{\frac{1}{2}}$ 

 $Q = C(\Delta P)$ 

 $Q = A \cdot C(\Delta P)^n$ Kronval "an engineer"

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Joseph Lstiburek – Airflow 51

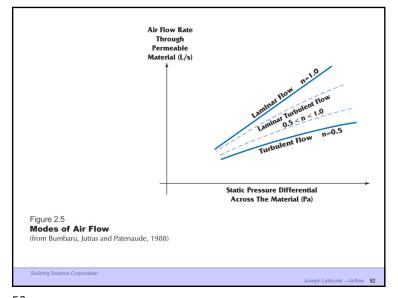
 $Q = A \cdot C_D \left[ \frac{2}{\rho} (\Delta P) \right]^{\frac{1}{2}}$ Bernoulli  $Q = C_K \frac{\rho}{\mu} (\Delta P)$ Darcy  $Q = A \cdot C(\Delta P)^{\frac{1}{2}}$ 

 $Q = C(\Delta P)$ 

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Joseph Lstiburek - Airflow 50

50



51

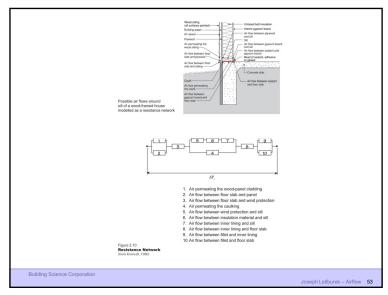
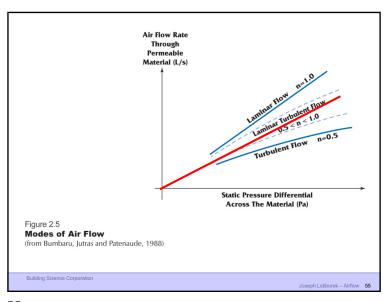


Figure 2.11
Three Dimensional Multi-Layer Multi-Cell Analogue

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Joseph Lstiburek – Airflow 54

53

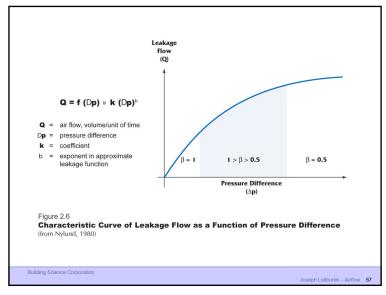


Air Flow Rate
Through
Permeable
Material (L/s)

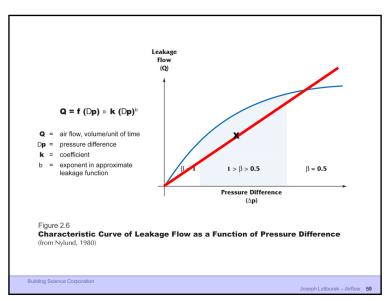
Static Pressure Differential
Across The Material (Pa)

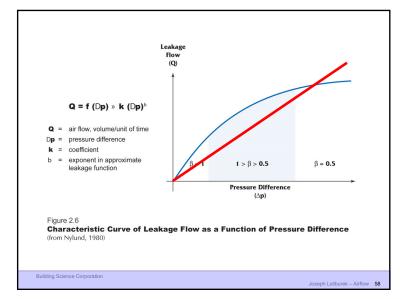
Figure 2.5
Modes of Air Flow
(from Bumbaru, Jutras and Patenaude, 1988)

55 56

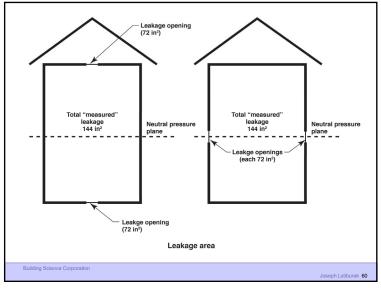


57





58



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Dilution Is Not The Solution To Indoor Pollution Source Control

61

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Target Range

Winter Summer

Uncomfortably Comfort Range\*

Uncomfortably Wet

25 50 70 100

75

Relative Humidity (RH) %

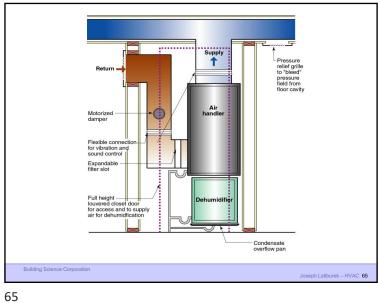
Dilution For People
Source Control For The Building

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62

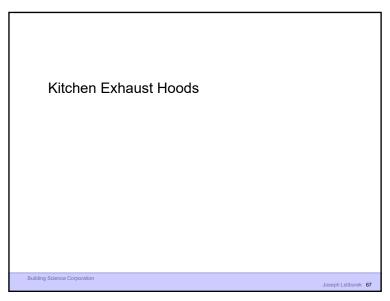
Recommended Range of Relative Humidity
Above 25 percent during winter
Below 70 percent during summer

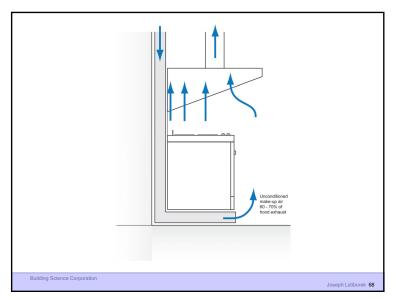
63



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66

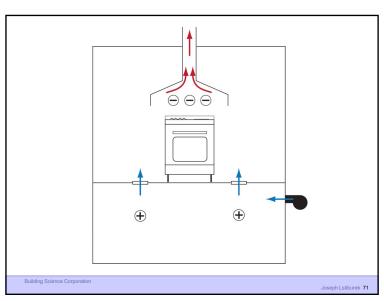




67 68

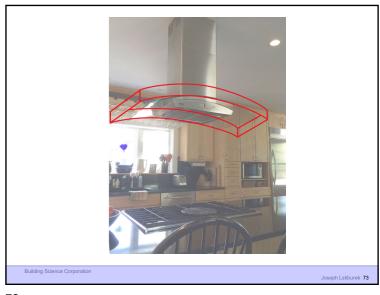


Building Science Corporation Joseph Lstiburek 70 70



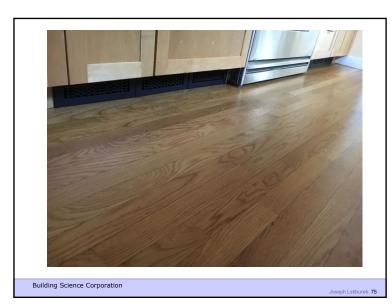


71 72





73





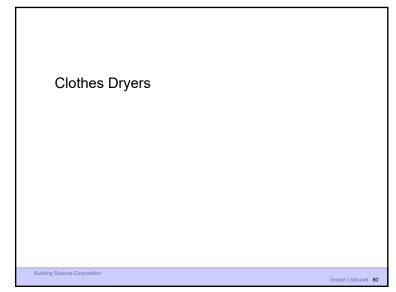
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77





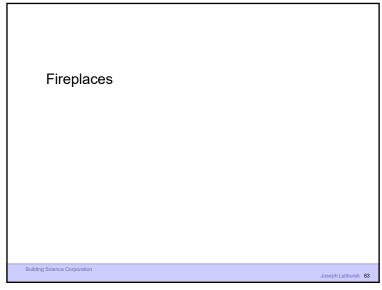
79 80



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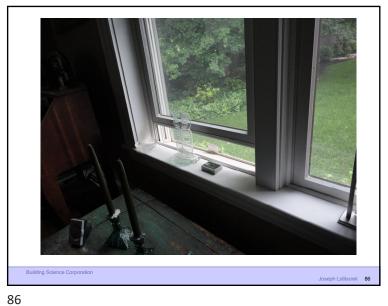
81 82



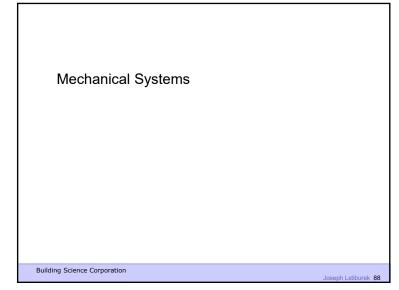


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Mechanical Systems
Cooling System To Make It Cold

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Mechanical Systems
Cooling System To Make It Cold
Dehumidification System To Make It Dry
Heating System To Make It Warm

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Mechanical Systems
Cooling System To Make It Cold
Dehumidification System To Make It Dry

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Incanh Letihurak

90

Mechanical Systems
Cooling System To Make It Cold
Dehumidification System To Make It Dry
Heating System To Make It Warm
Energy Recovery System To Keep It Cold
and Dry and Warm and Comfortable

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Mechanical Systems
Cooling System To Make It Cold
Dehumidification System To Make It Dry
Heating System To Make It Warm
Energy Recovery System To Keep It Cold
and Dry and Warm and Comfortable
Distribution System To Make It Uniform

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oseph Lstiburek 93

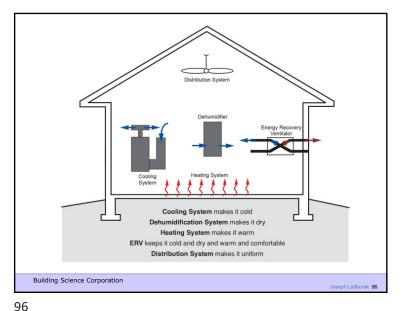
93

The Best Approach is Not to Combine Them.....

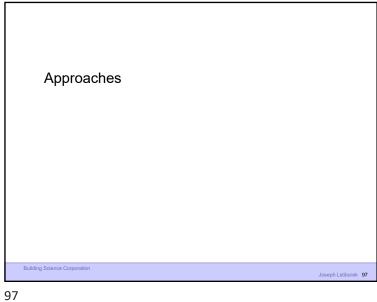
Mechanical Systems
Cooling System To Make It Cold
Dehumidification System To Make It Dry
Heating System To Make It Warm
Energy Recovery System To Keep It Cold
and Dry and Warm and Comfortable
Distribution System To Make It Uniform
Range Hoods Are A Special Kind of Hell

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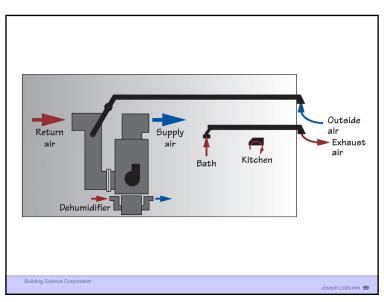
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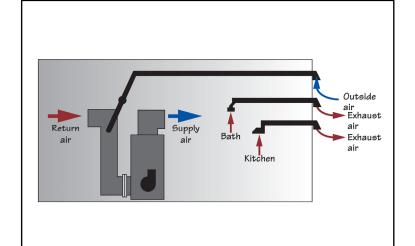


Outside Return Exhaust Kitchen Bath Building Science Corporation Joseph Lstiburek 98

98

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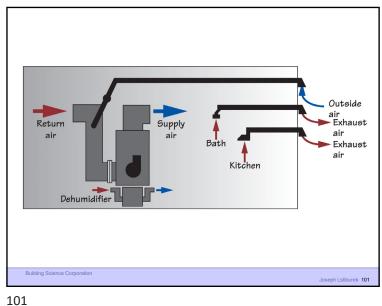


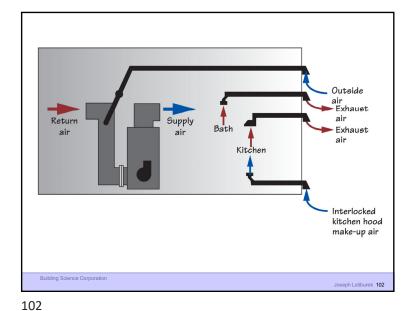


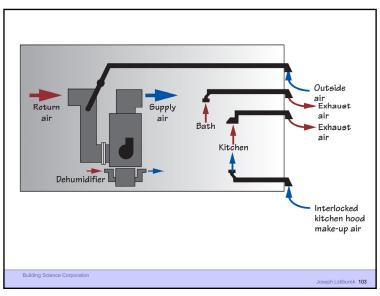
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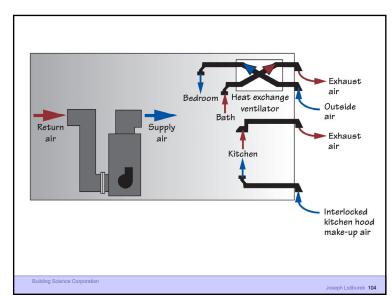
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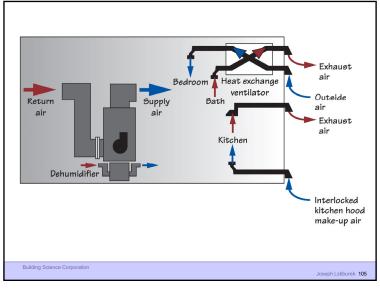
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Exhaust air

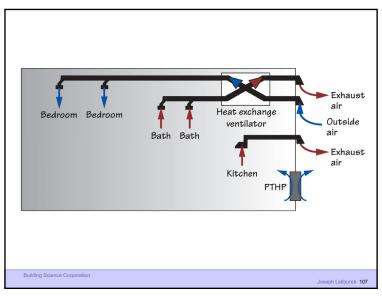
Outside air

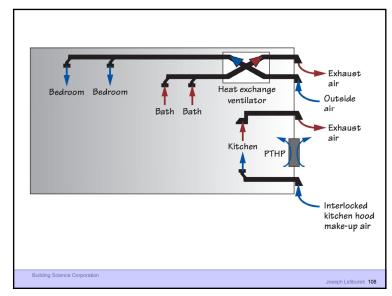
Exhaust air

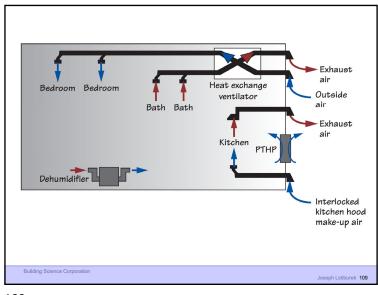
Exhaust air

Exhaust air

Adoseph Lstiburek. 106

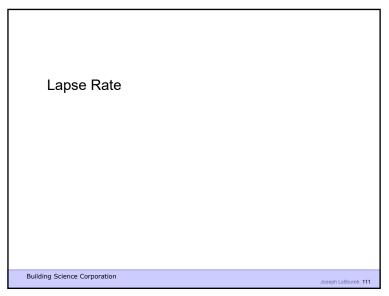


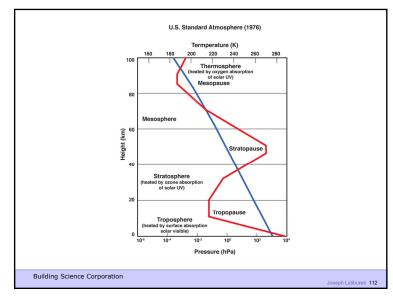




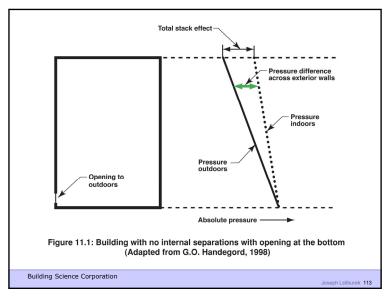


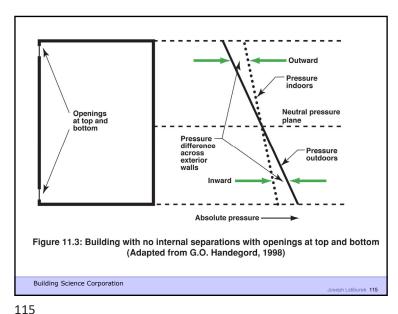
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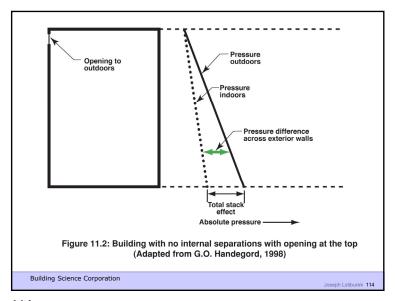


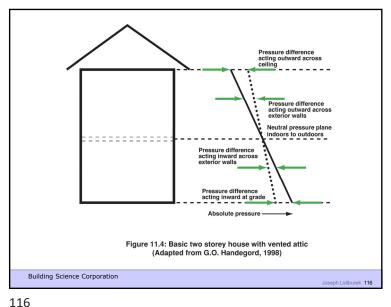


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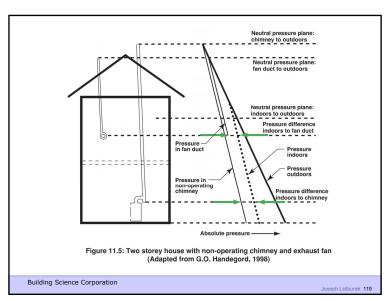




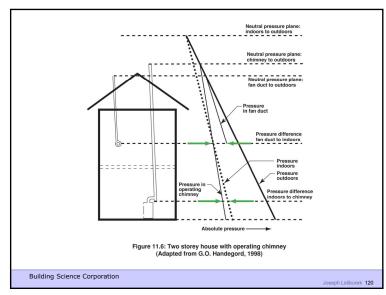


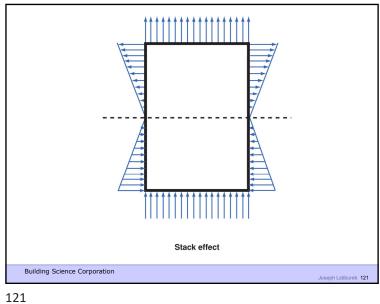






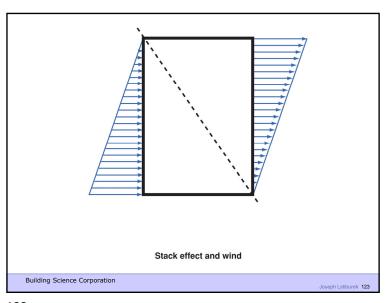


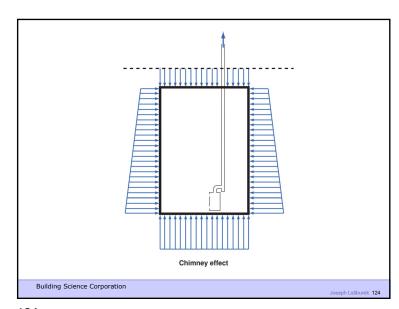




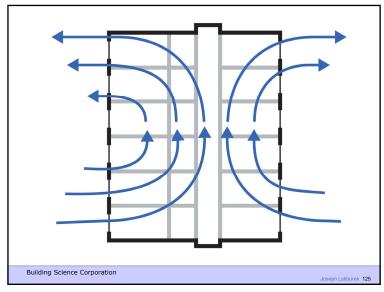
Wind Building Science Corporation Joseph Lstiburek 122

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123 124



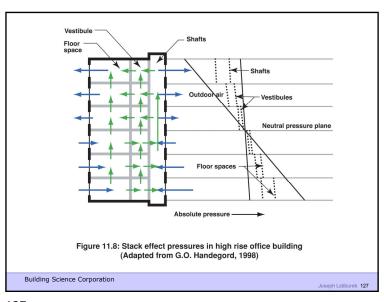
Stack Effect Flow Out (Exfiltration)

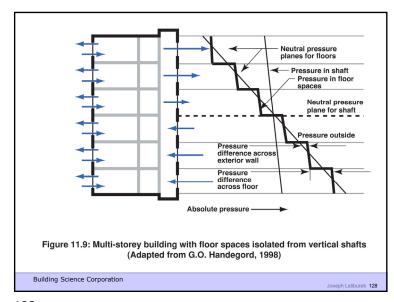
Pinside drops with height slower than Poutside with height faster than Pinside

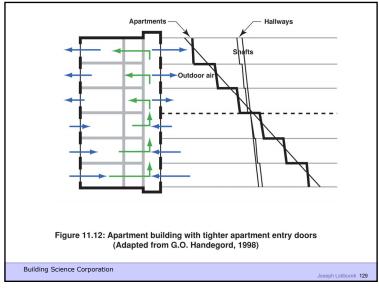
Neutral Pressure Plane

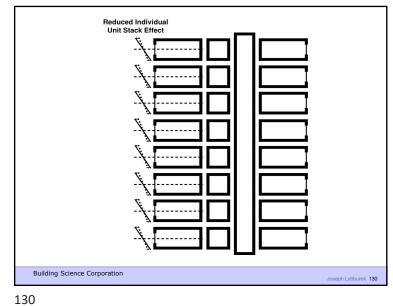
Stack Effect Flow In (Infiltration)

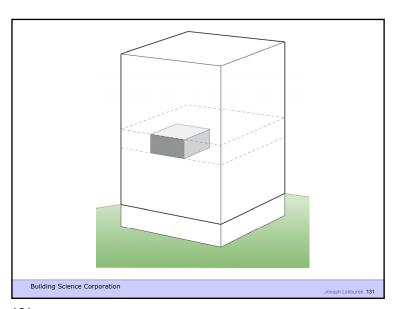
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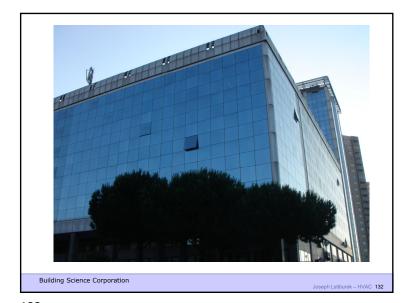


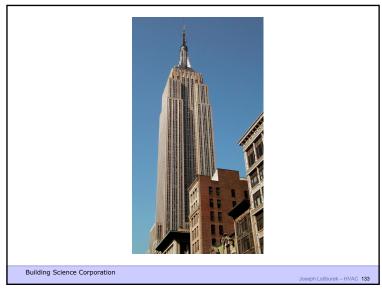


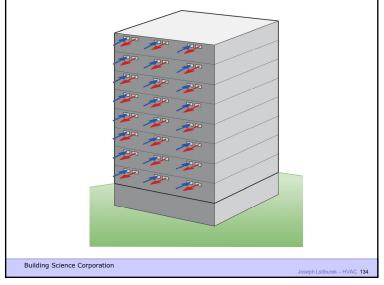




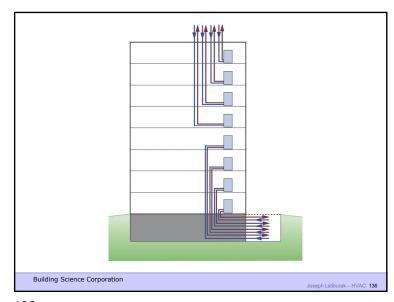


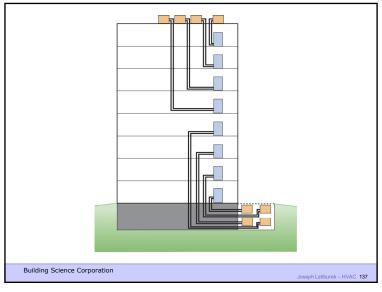


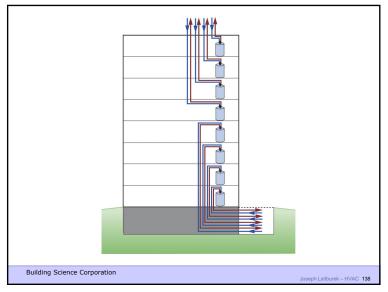


















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Building Science Corporation

Joseph Lstburek, 146

145 146





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Joseph Latiburek – HVAC 150

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149





151 152





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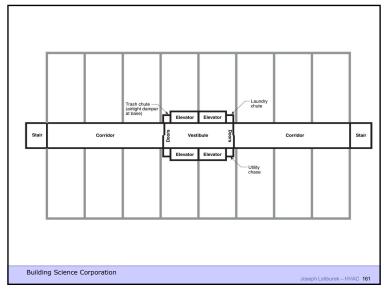
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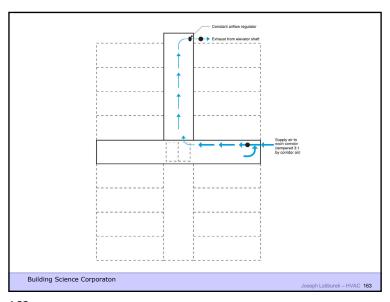


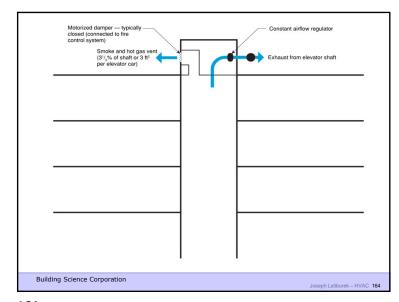














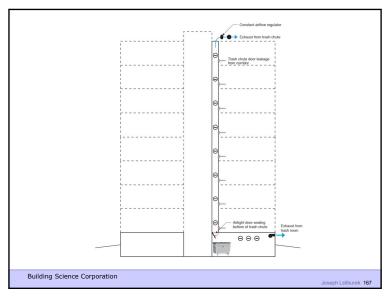
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Constant airriow regulator

E-chaust from elevator shaft

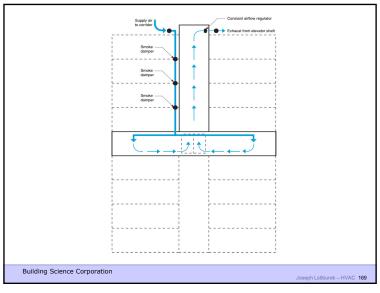
Joseph Lstiburek – HVAC 166

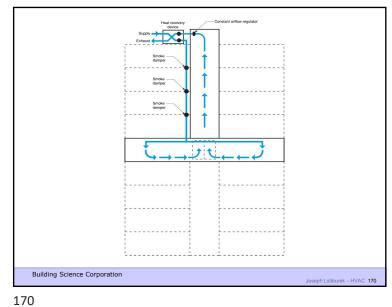
165 166



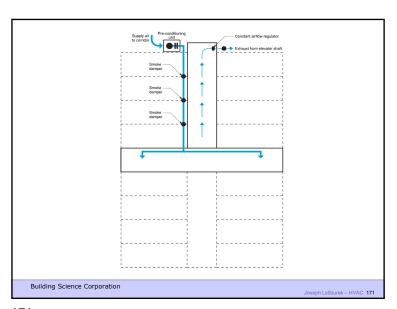


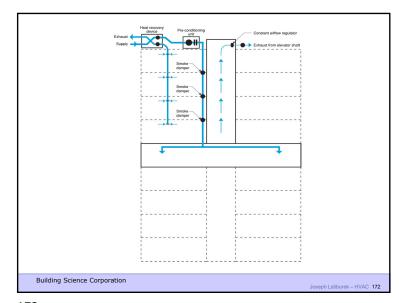
167 168





169



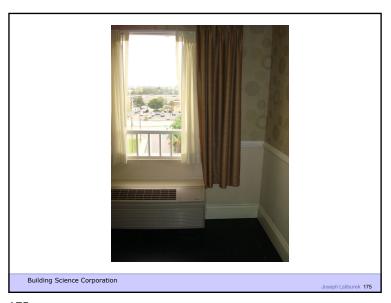


171 172





173 174





175 176



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Joseph Lstiburek 178

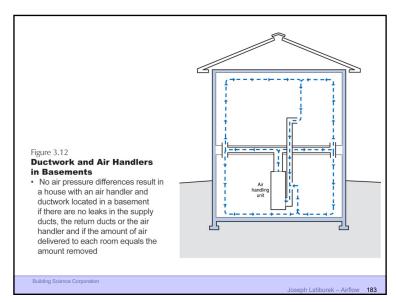
177 178



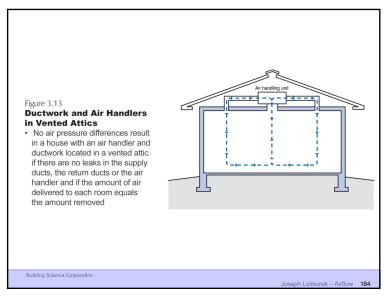


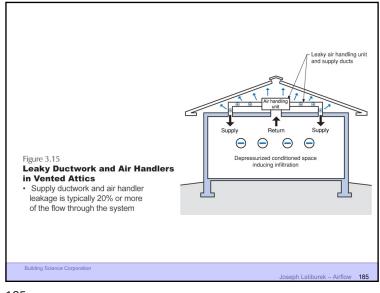
179

















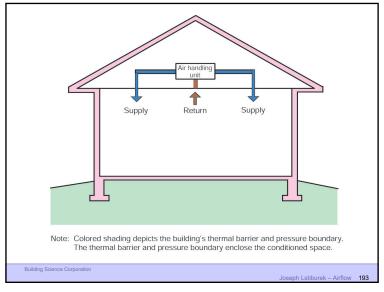
189



Duct Leakage Should Be Less Than 5% of Rated Flow As Tested By Pressurization To 25 Pascals

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Joseph Lstiburek – Airflow 192



194

193

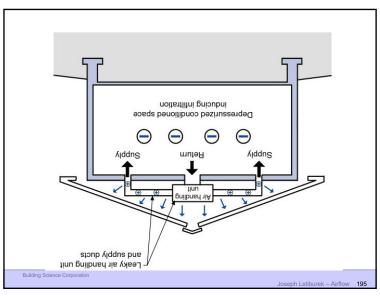


Figure 3.16

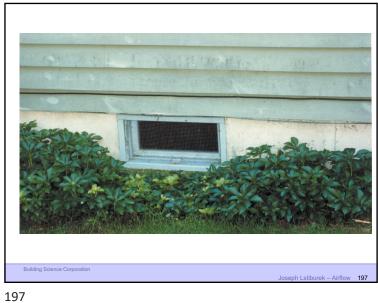
Leaky Supply Ductwork in Vented Crawl Space

• Air pressurization pattern with mechanical system ducts in the crawl space

Building Science Corporation

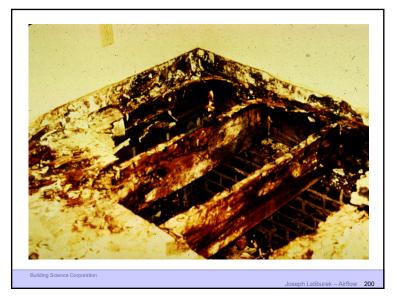
Joseph Lstiburek – Airflow 196

195



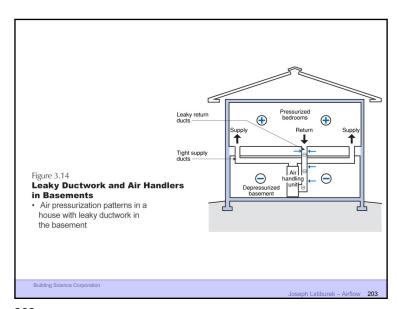








201 202





203 204

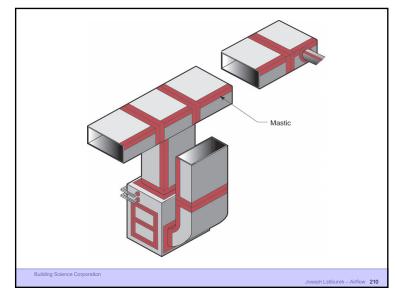


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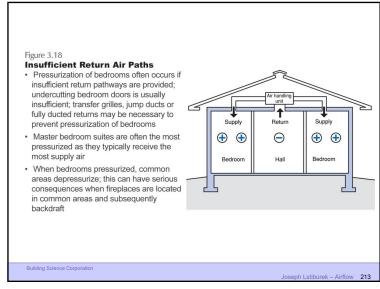


209 210





211 212



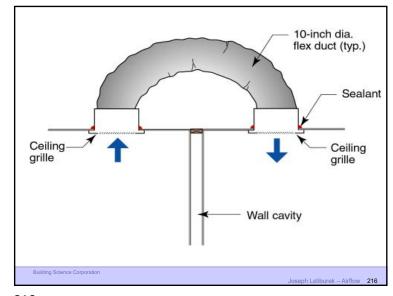
Grille located high in wall on bedroom side to avoid blockage by furniture

Cavity is sealed tight, drywall glued to studs and plates on both sides

Grille located low in wall on hallway side

213





215 216





217 218





219 220



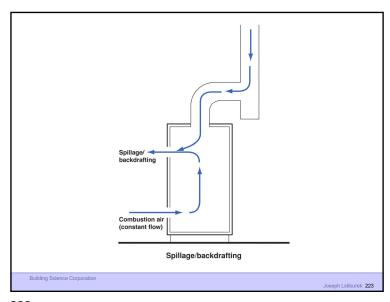
Dilution air (variable flow)

Combustion air (constant flow)

Draft hood

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221 222





223 224

