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Building Science

Airflow In Buildings I

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Stuff That Is Not Particularly Useful But Studied
and Researched to Death

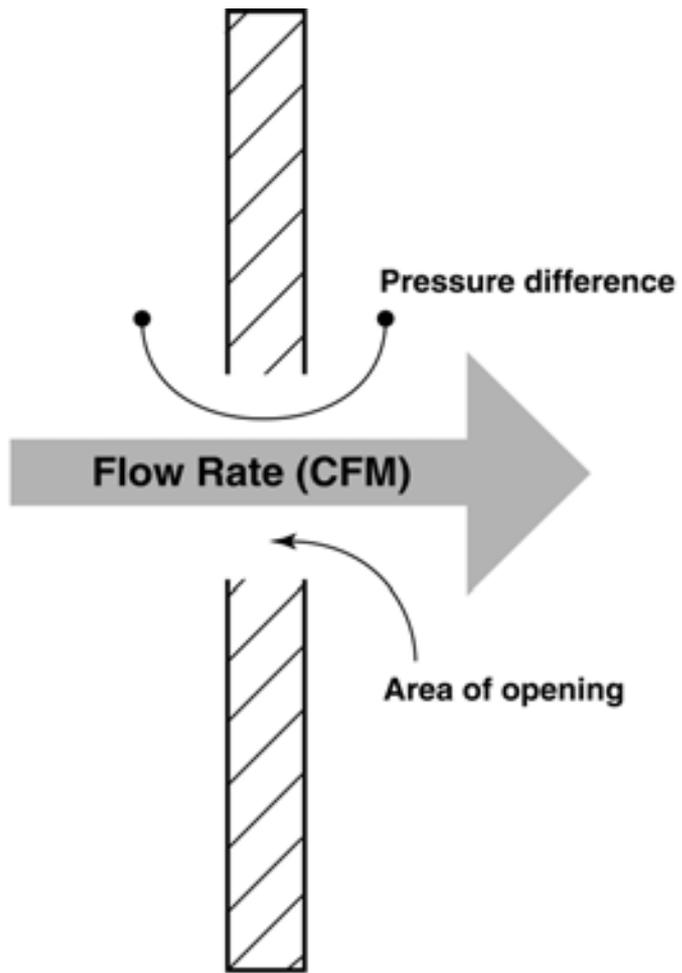
Stuff That Is Very Useful but Ignored by the
Research Community

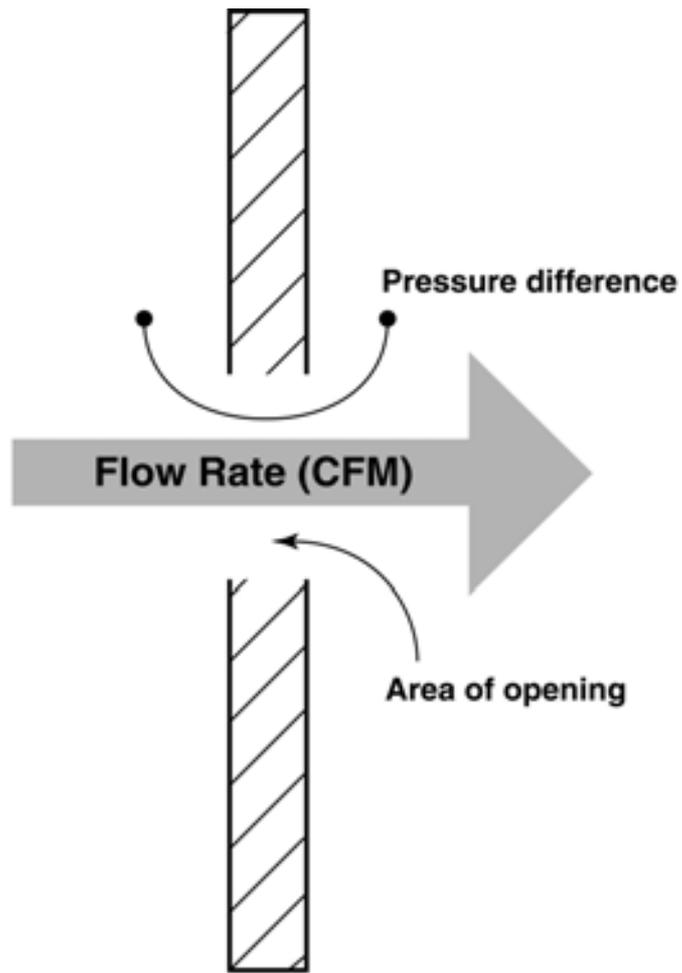
Stuff That Is Not Particularly Useful But Studied
and Researched to Death

“this is called Physics”

Stuff That Is Very Useful but Ignored by the
Research Community

“this is called Engineering”





“this is a lie”

Flow Through Orifices

Turbulent Flow - “inertial effects”

Flow Through Porous Media

Laminar Flow - “viscosity effects”

Flow Through Orifices

Turbulent Flow - “inertial effects”

Flow Through Porous Media

Laminar Flow - “viscosity effects”

“true but not useful”

$$Q = A \cdot C_D \left[\frac{2}{\rho} (\Delta P) \right]^{\frac{1}{2}}$$

Bernoulli

$$Q = C_K \frac{\rho}{\mu} (\Delta P)$$

Darcy

$$Q = A \cdot C_D \left[\frac{2}{\rho} (\Delta P) \right]^{\frac{1}{2}} \quad \text{Bernoulli}$$

$$Q = C_K \frac{\rho}{\mu} (\Delta P) \quad \text{Darcy}$$

$$Q = A \cdot C (\Delta P)^{\frac{1}{2}}$$

$$Q = C (\Delta P)$$

$$Q = A \cdot C_D \left[\frac{2}{\rho} (\Delta P) \right]^{\frac{1}{2}}$$

Bernoulli

$$Q = C_K \frac{\rho}{\mu} (\Delta P)$$

Darcy

$$Q = A \cdot C (\Delta P)^{\frac{1}{2}}$$

$$Q = C (\Delta P)$$

$$Q = A \cdot C (\Delta P)^n$$

Kronval “an engineer”

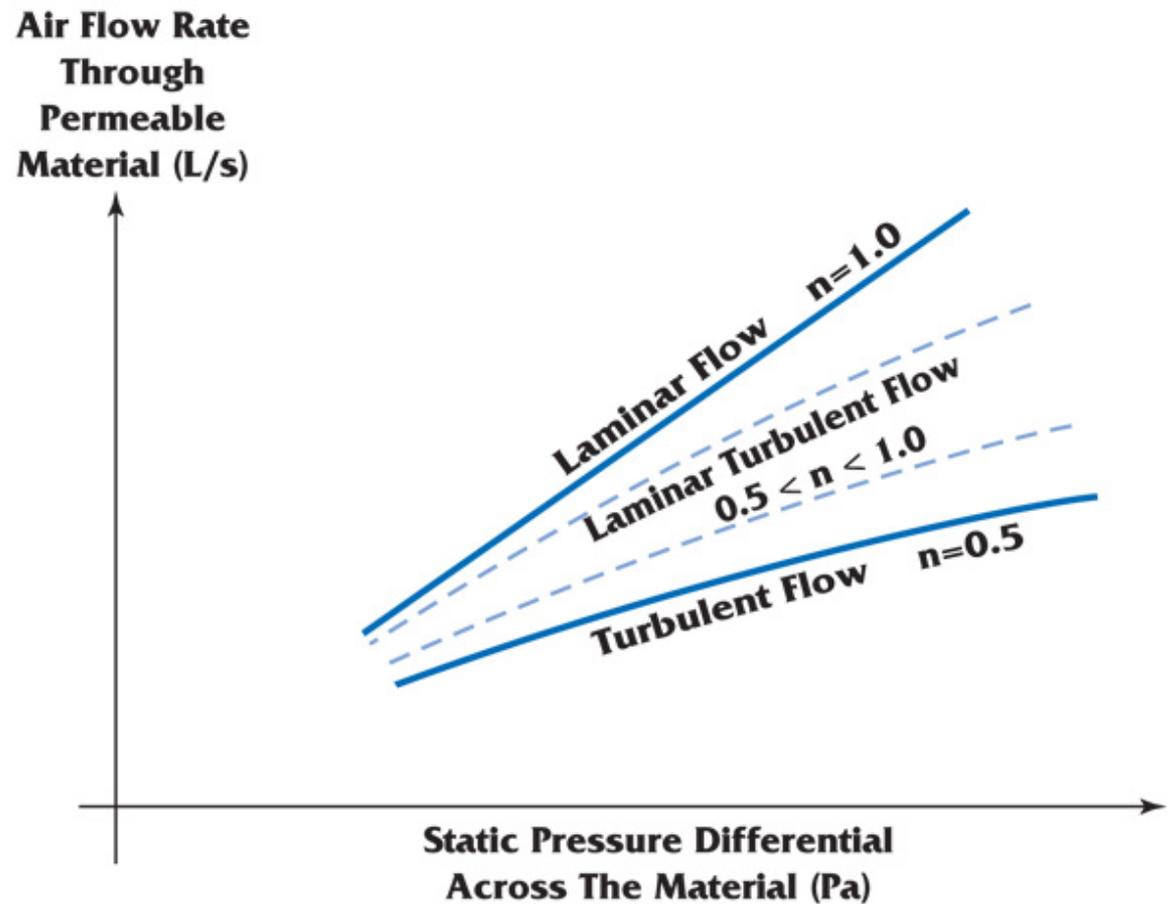


Figure 2.5

Modes of Air Flow

(from Bumbaru, Jutras and Patenaude, 1988)

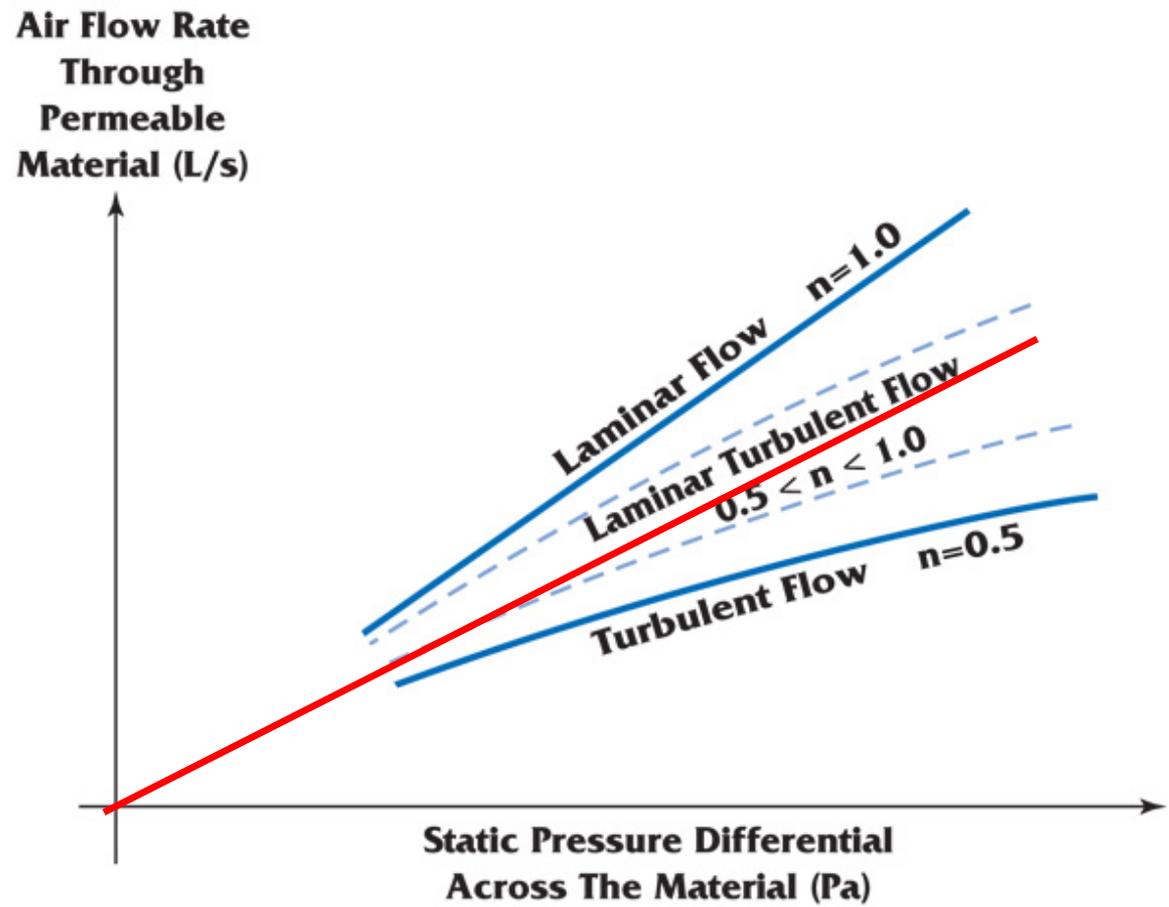


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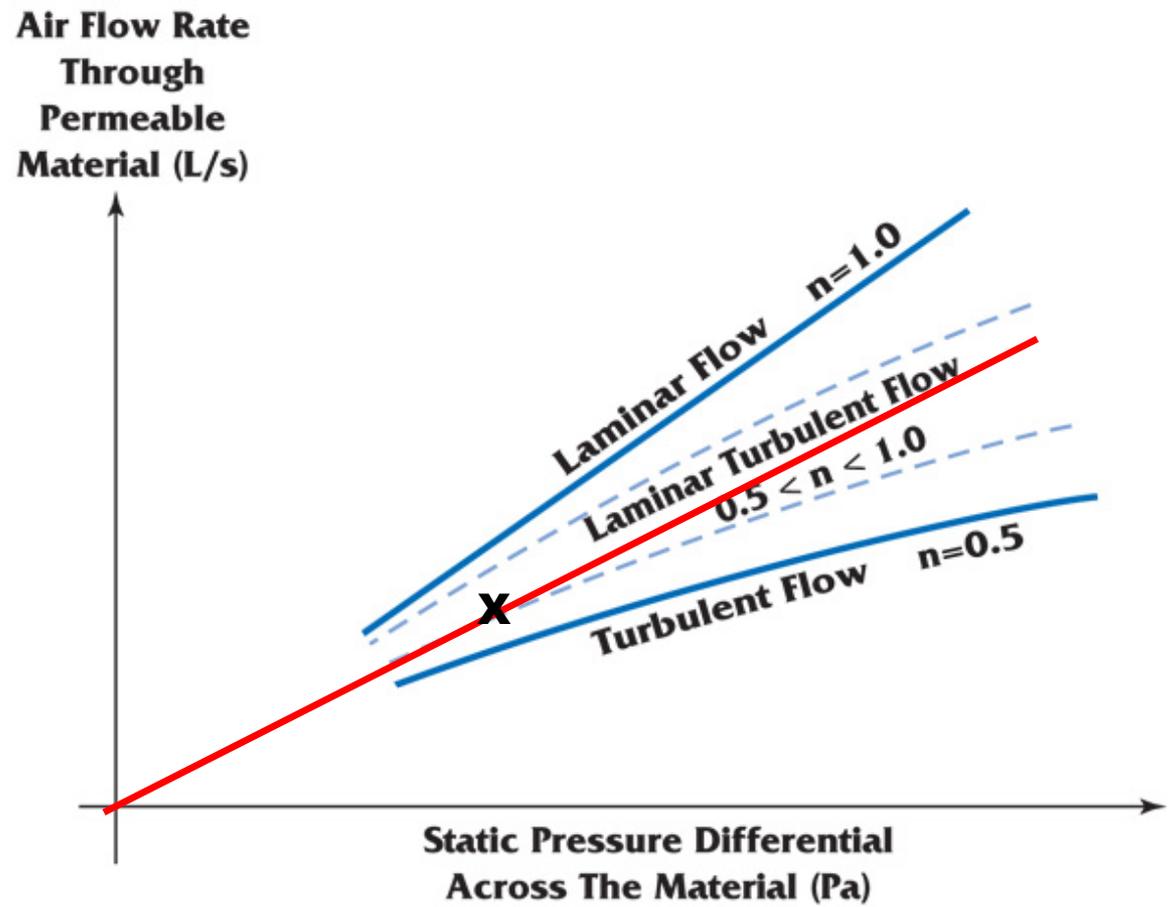


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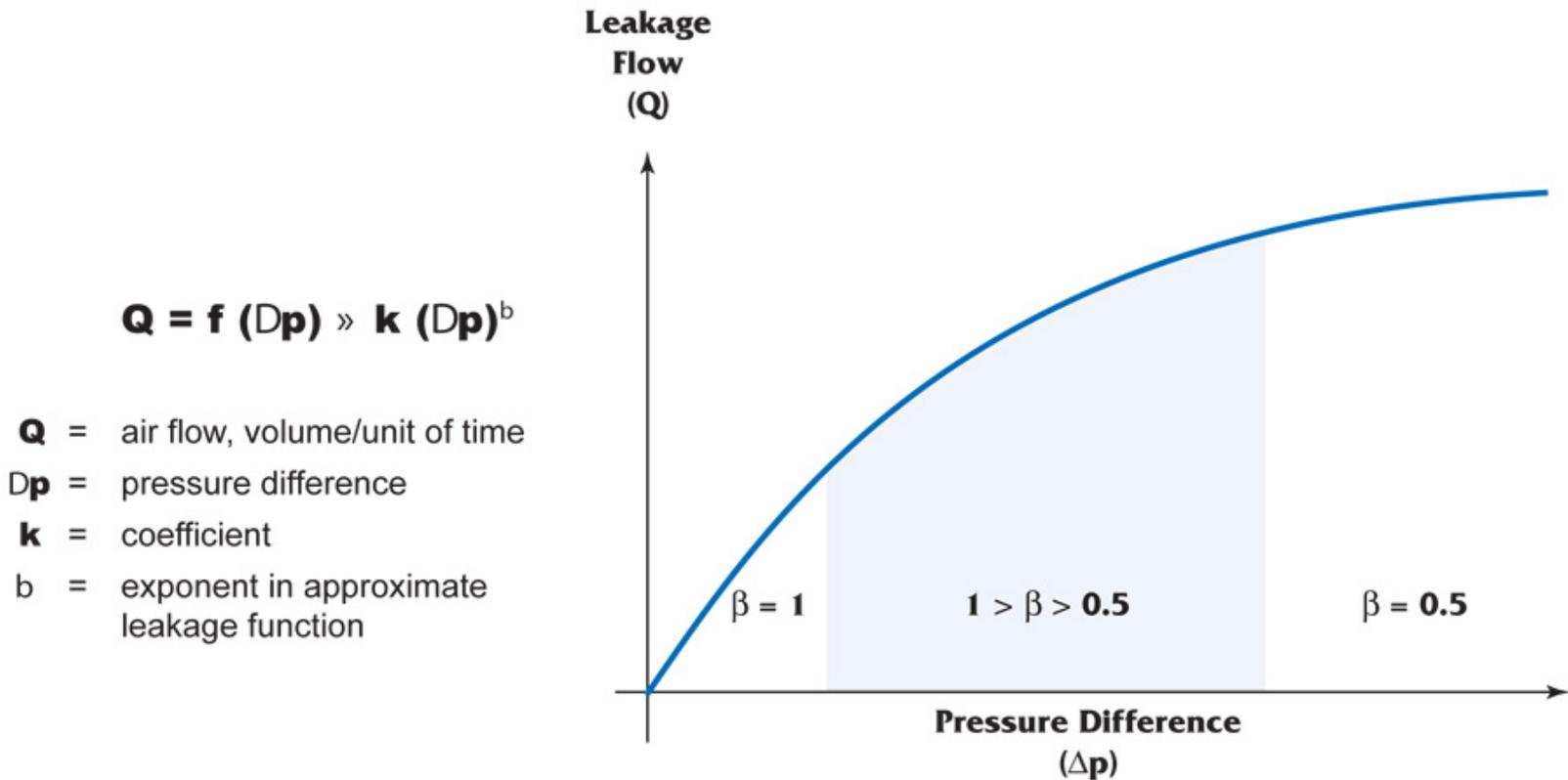


Figure 2.6

Characteristic Curve of Leakage Flow as a Function of Pressure Difference

(from Nylund, 1980)

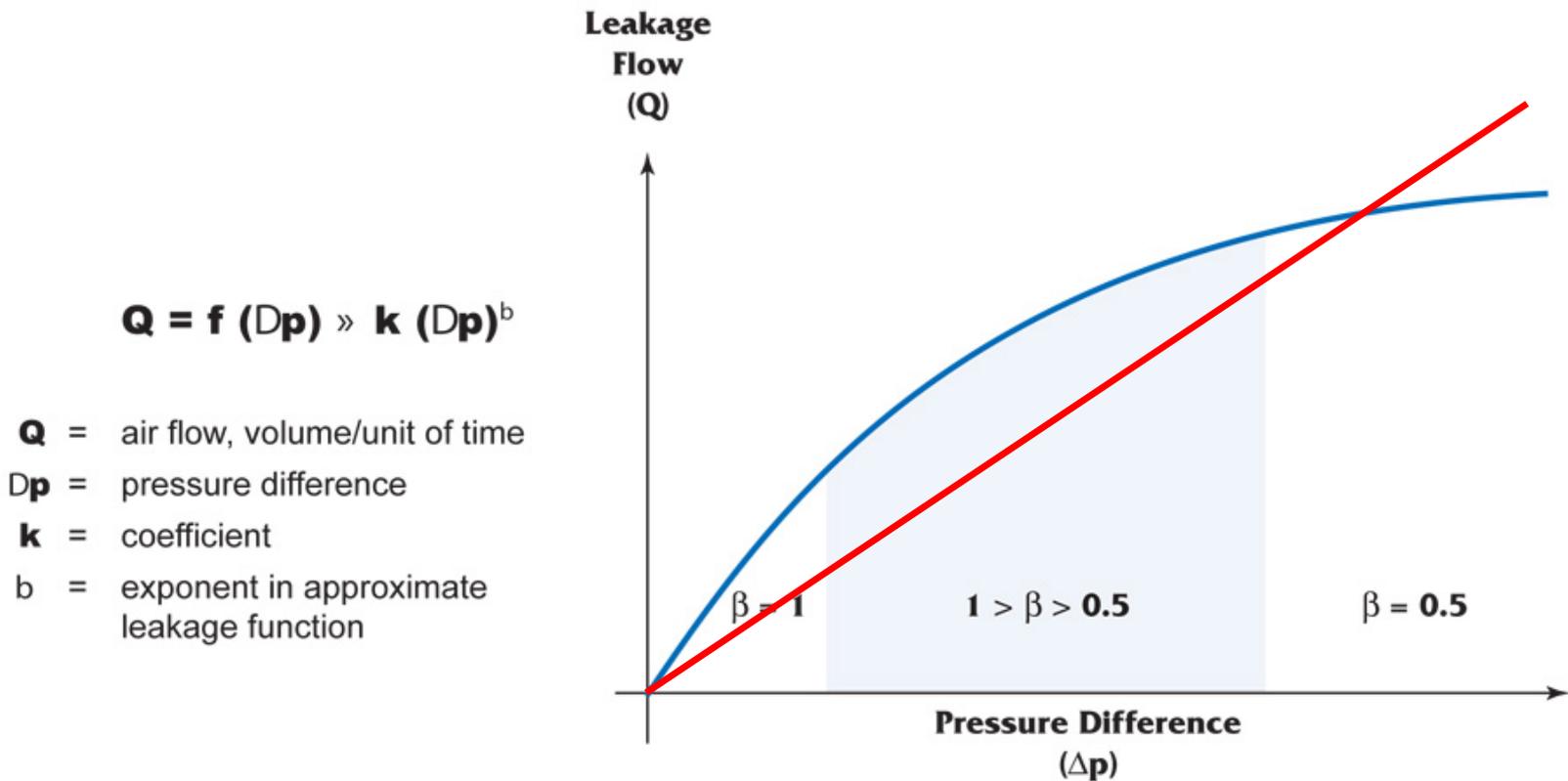


Figure 2.6
Characteristic Curve of Leakage Flow as a Function of Pressure Difference
 (from Nylund, 1980)

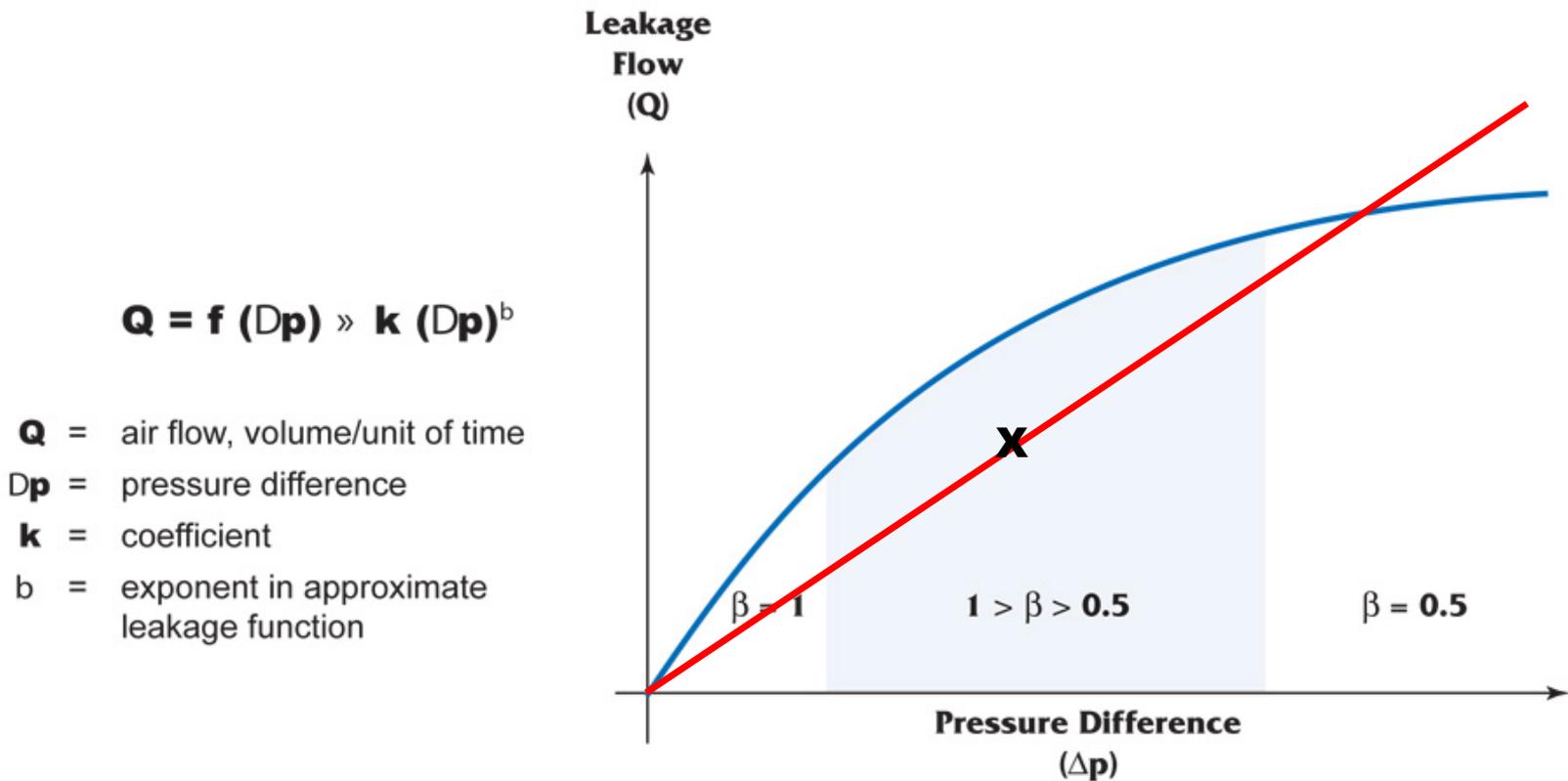
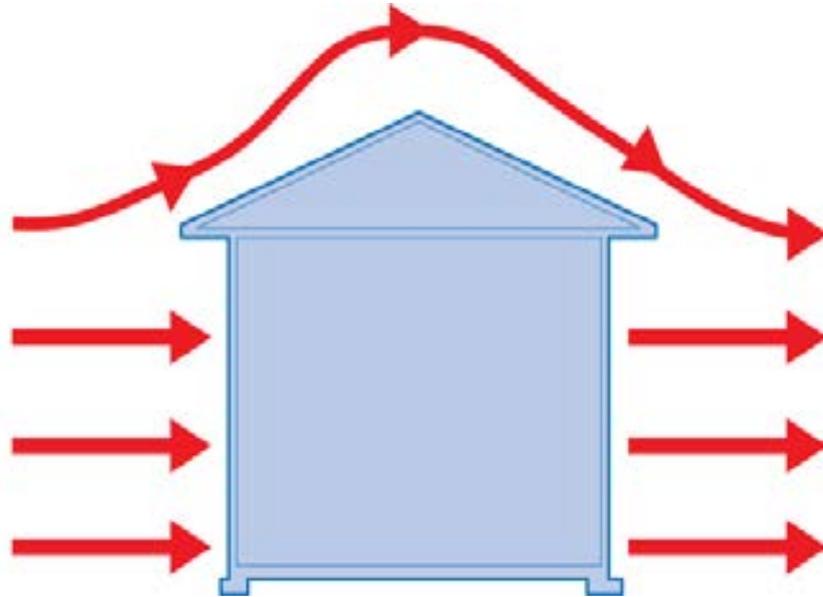
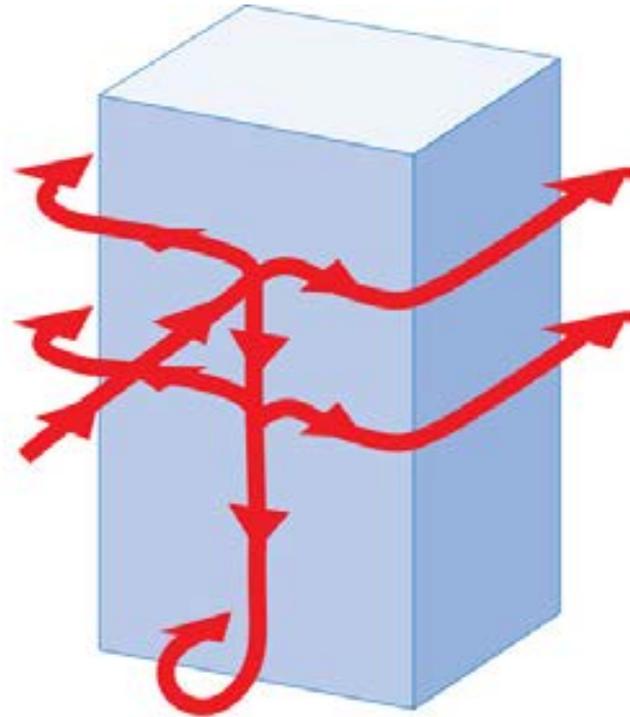
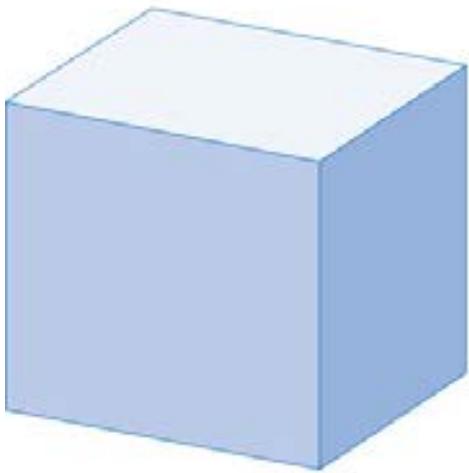
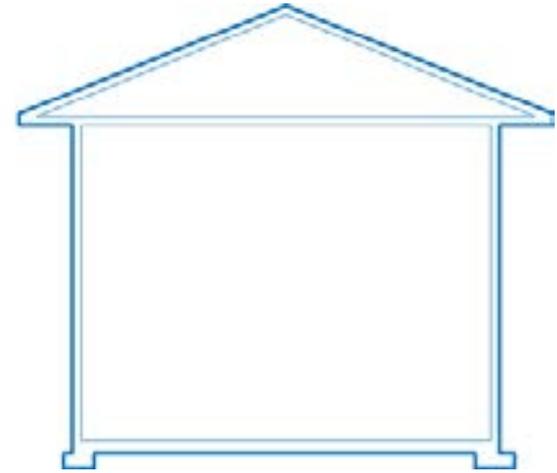
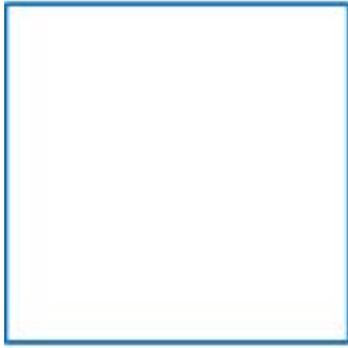


Figure 2.6
Characteristic Curve of Leakage Flow as a Function of Pressure Difference
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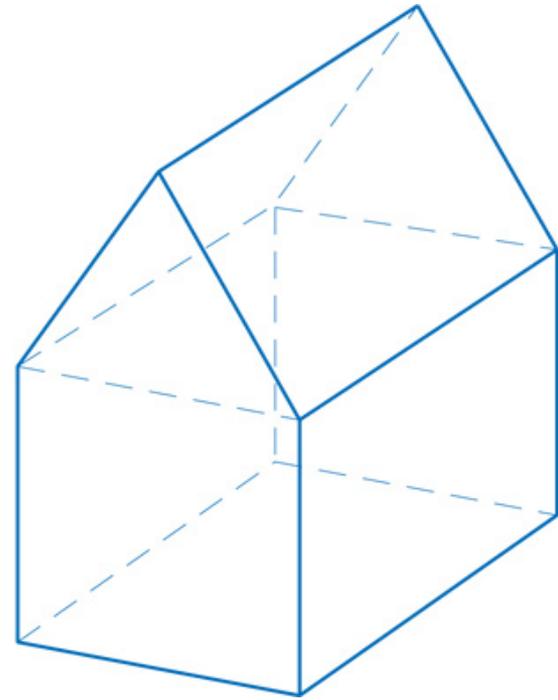
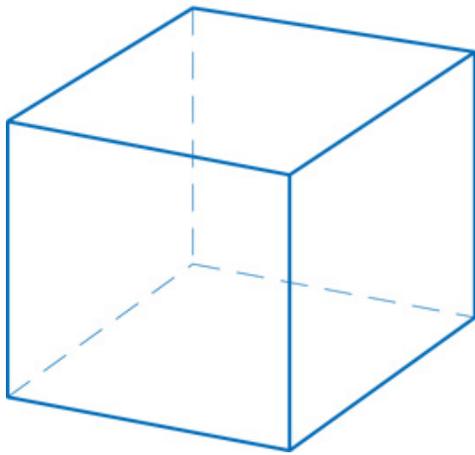




Figure 2.7
Two Dimensional Multi-Cell Analogue

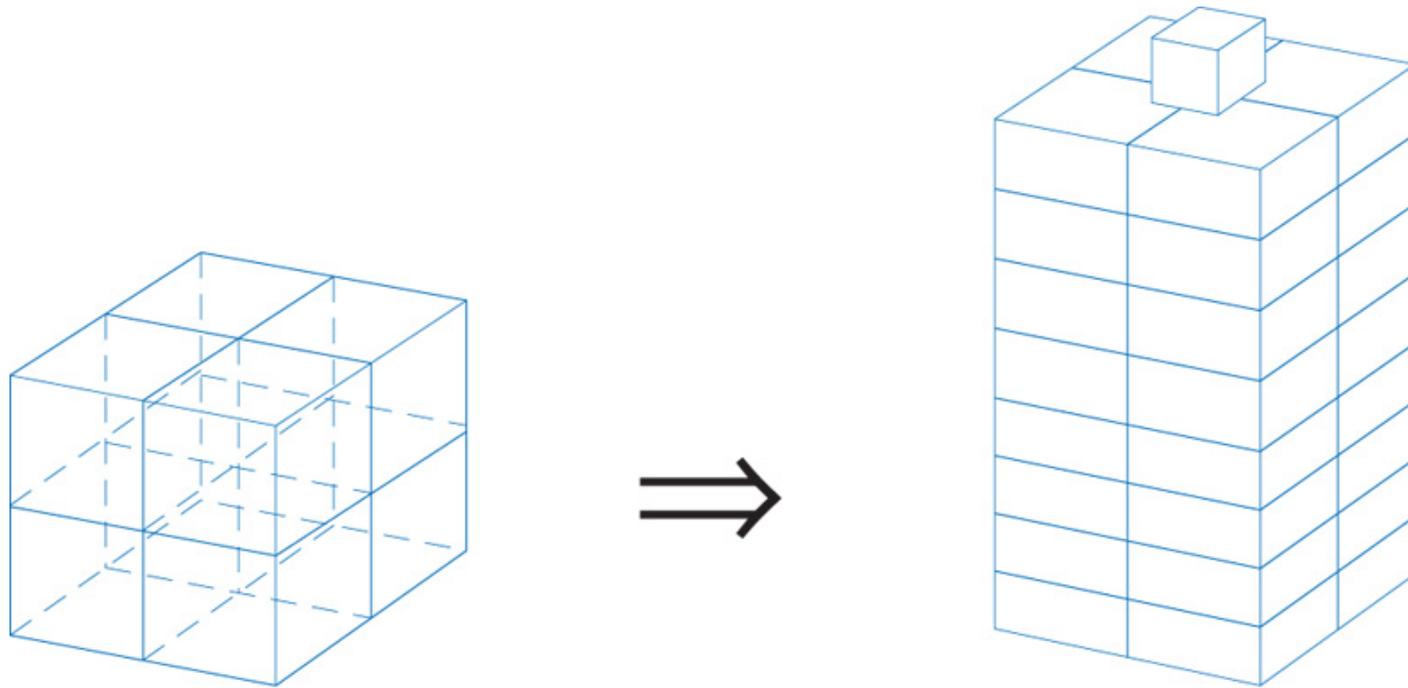


Figure 2.8
**Three Dimensional Multi-Cell
Analogue**

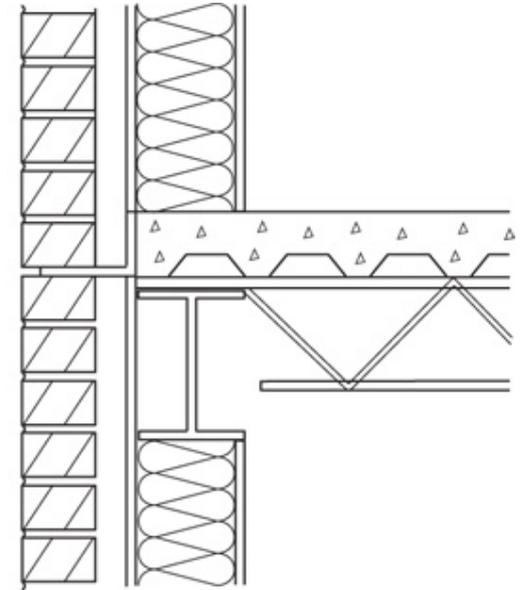
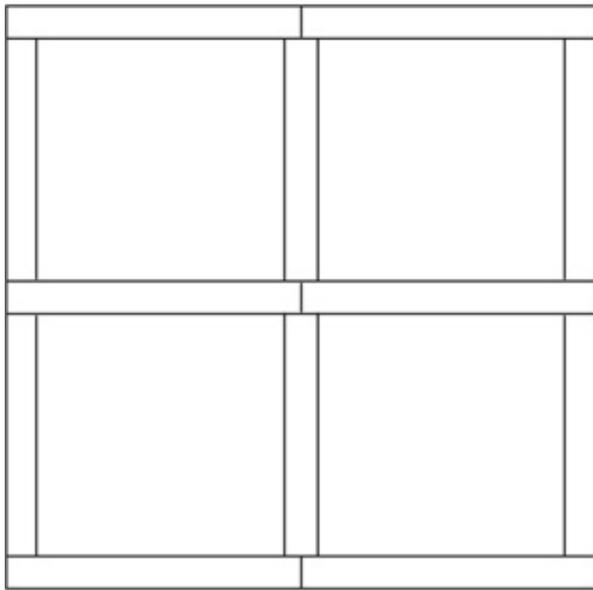
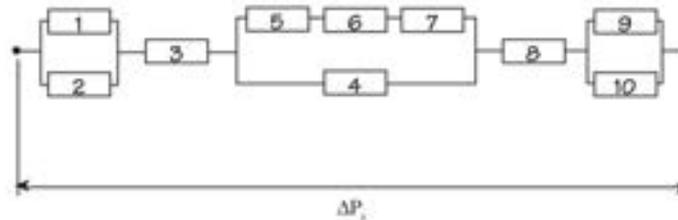
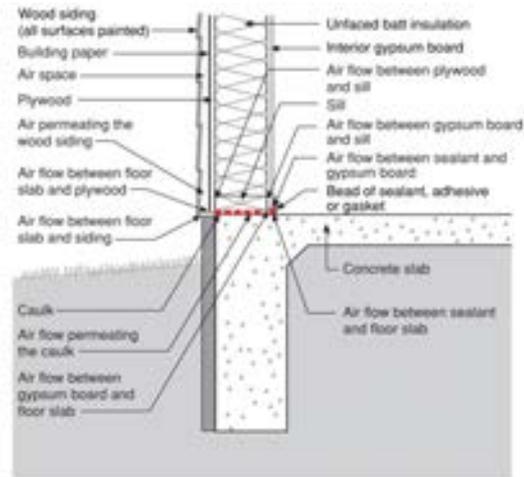


Figure 2.9
Two Dimensional Multi-Layer Multi-Cell Analogue

Possible air flows around sill of a wood-framed house modelled as a resistance network



1. Air permeating the wood-panel cladding
2. Air flow between floor slab and panel
3. Air flow between floor slab and wind protection
4. Air permeating the caulking
5. Air flow between wind protection and sill
6. Air flow between insulation material and sill
7. Air flow between inner lining and sill
8. Air flow between inner lining and floor slab
9. Air flow between fillet and inner lining
10. Air flow between fillet and floor slab

Figure 2.10
Resistance Network
 (from Kronvall, 1980)

Figure 2.11
**Three Dimensional Multi-Layer
Multi-Cell Analogue**

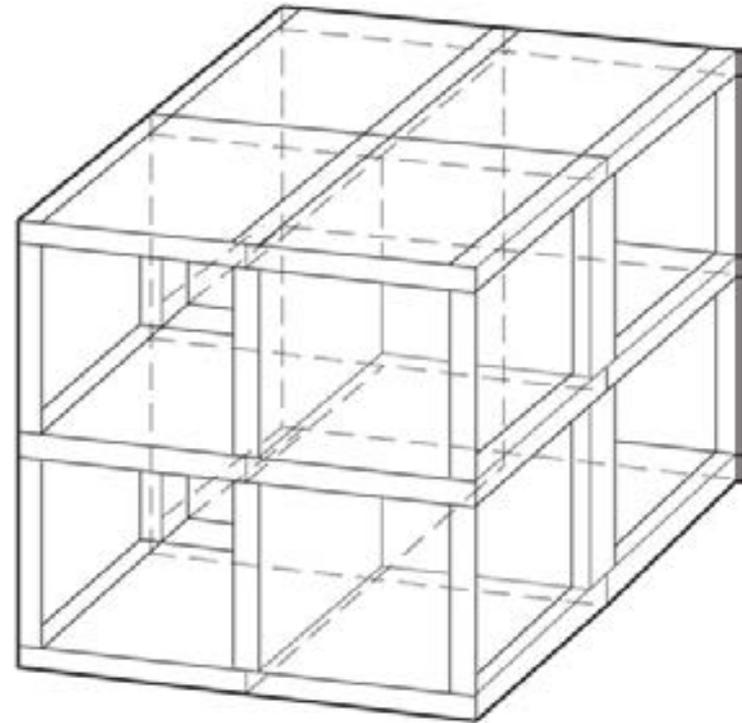


Figure 2.12
**Three Dimensional Multi-Layer
Multi-Cell Non-Contiguous
Analogue**

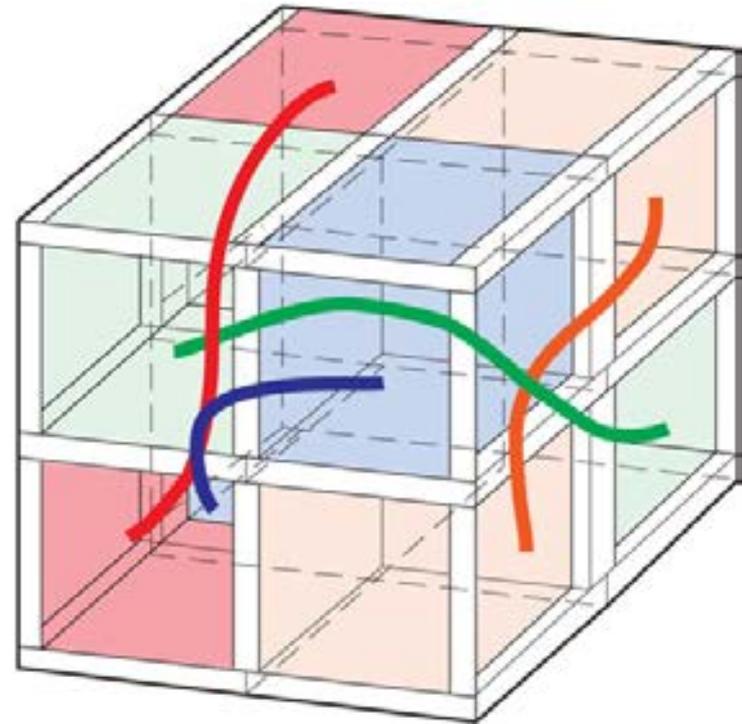
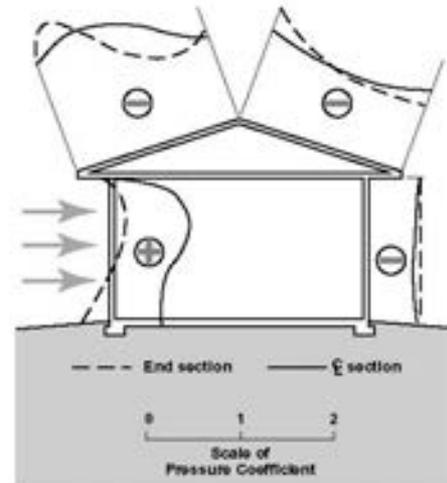
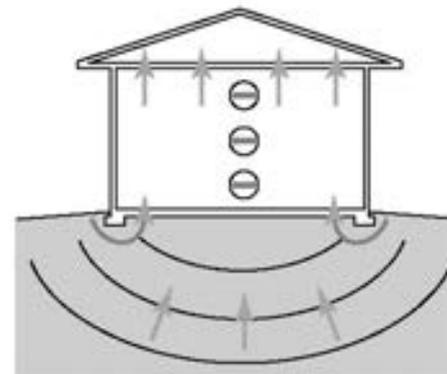


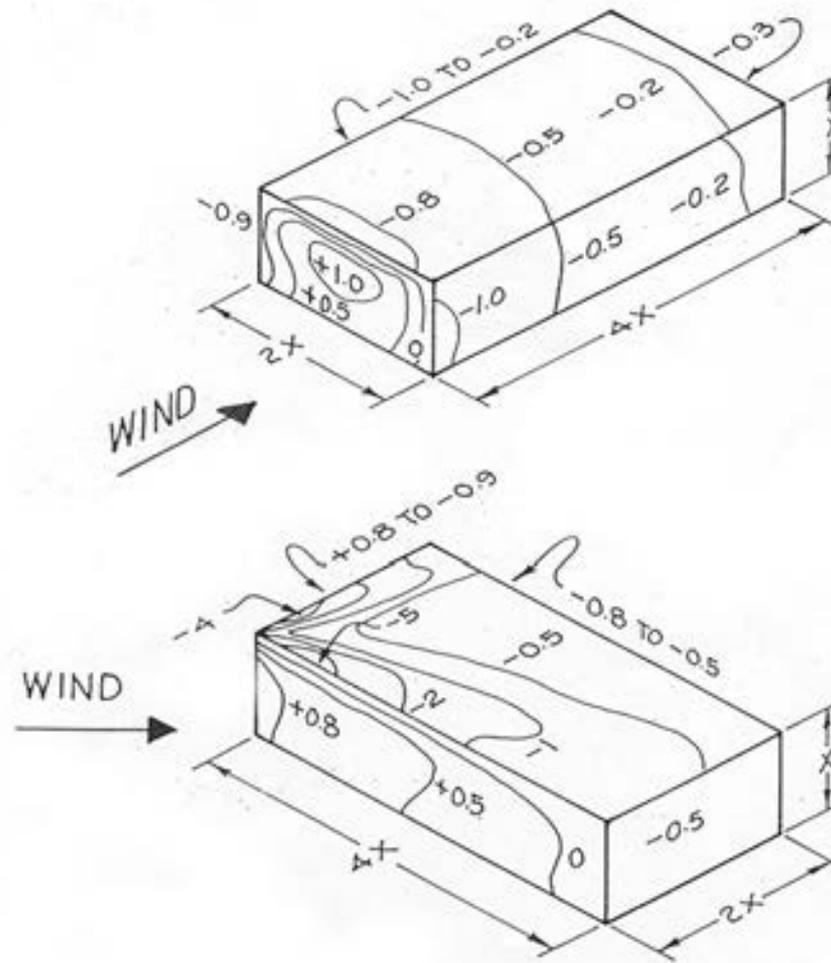
Figure 3.1
Exterior Air Pressure Field
(from Hutcheon & Handegord, 1983)



Distribution of pressures (+) and
suctions (-) on a house with a
low-sloped roof with wind
perpendicular to eave

Figure 3.2
**Exterior Air Pressure Field
Extending Below Grade**





Pressure coefficients on walls and roof of rectangular buildings without parapets.

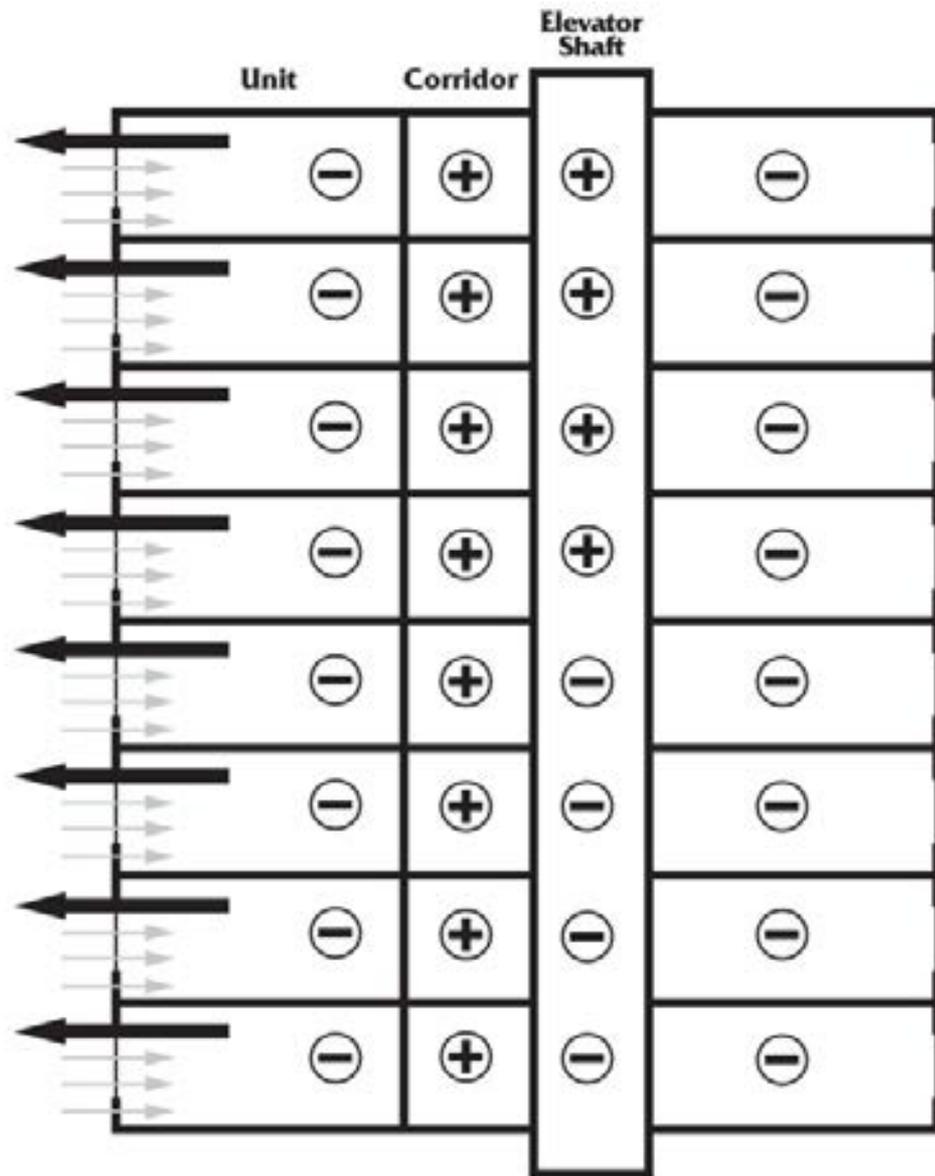


Figure 3.3
Interior Air Pressure Field

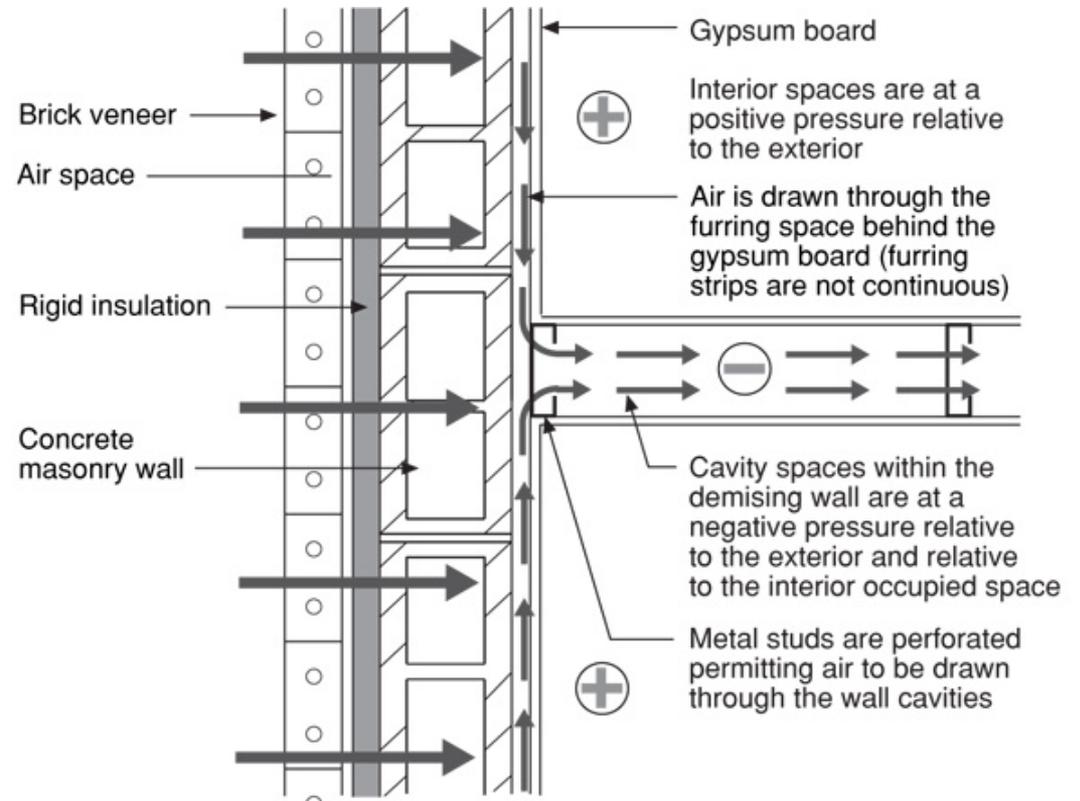


Figure 3.4
Interstitial Air Pressure Field

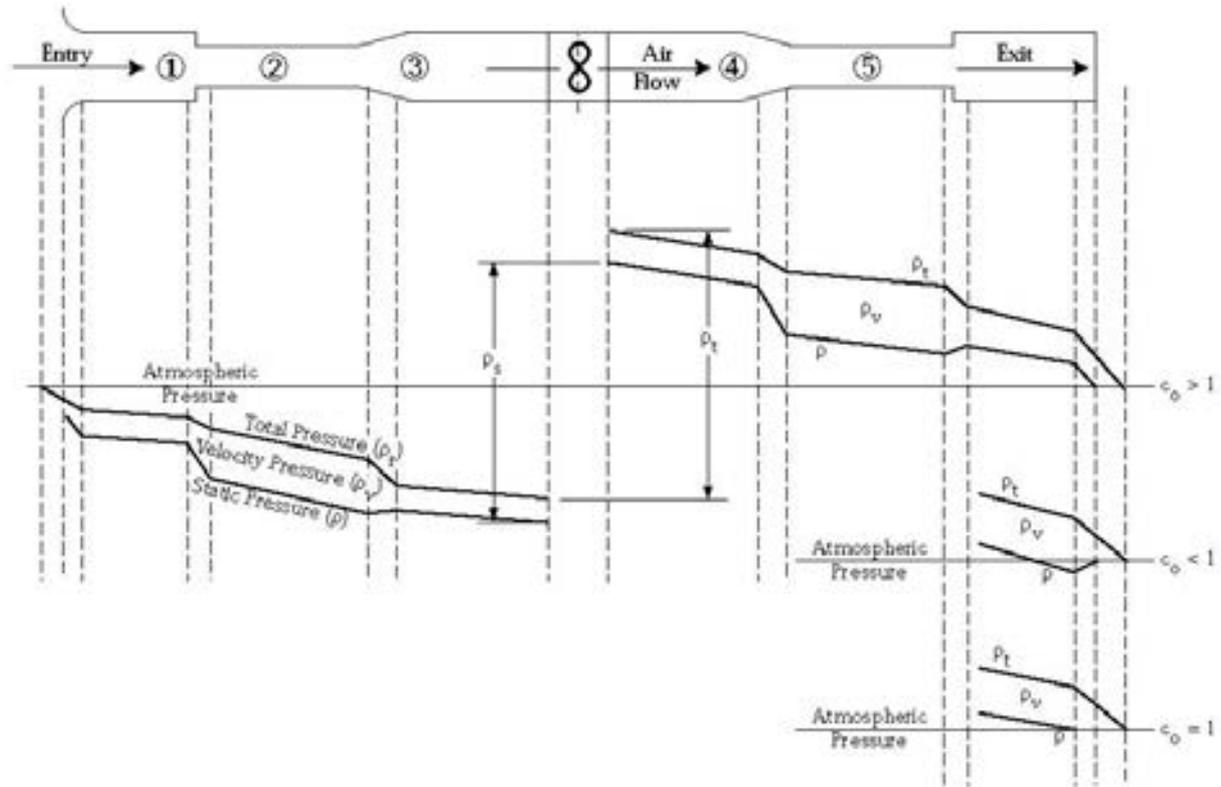
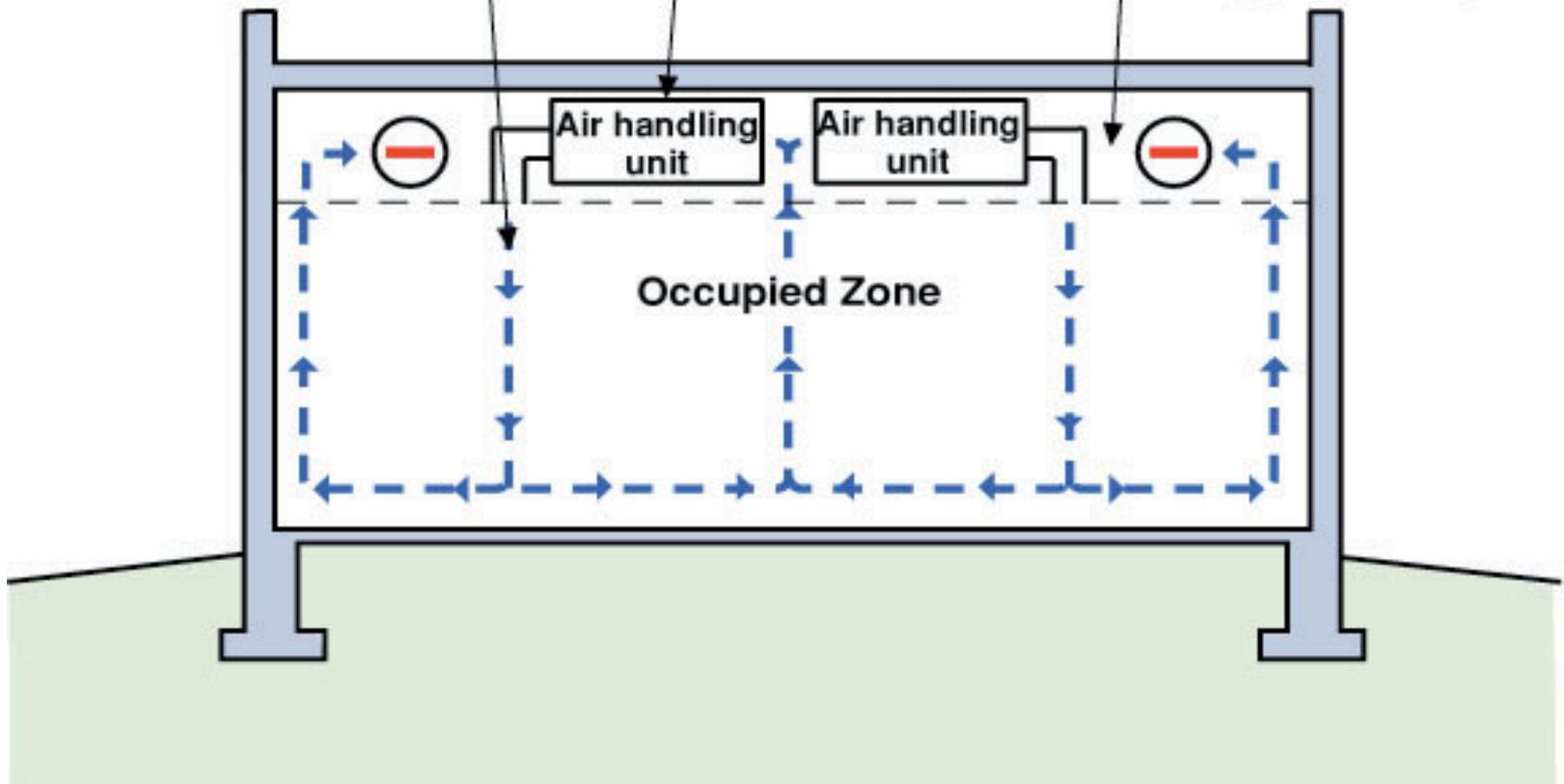


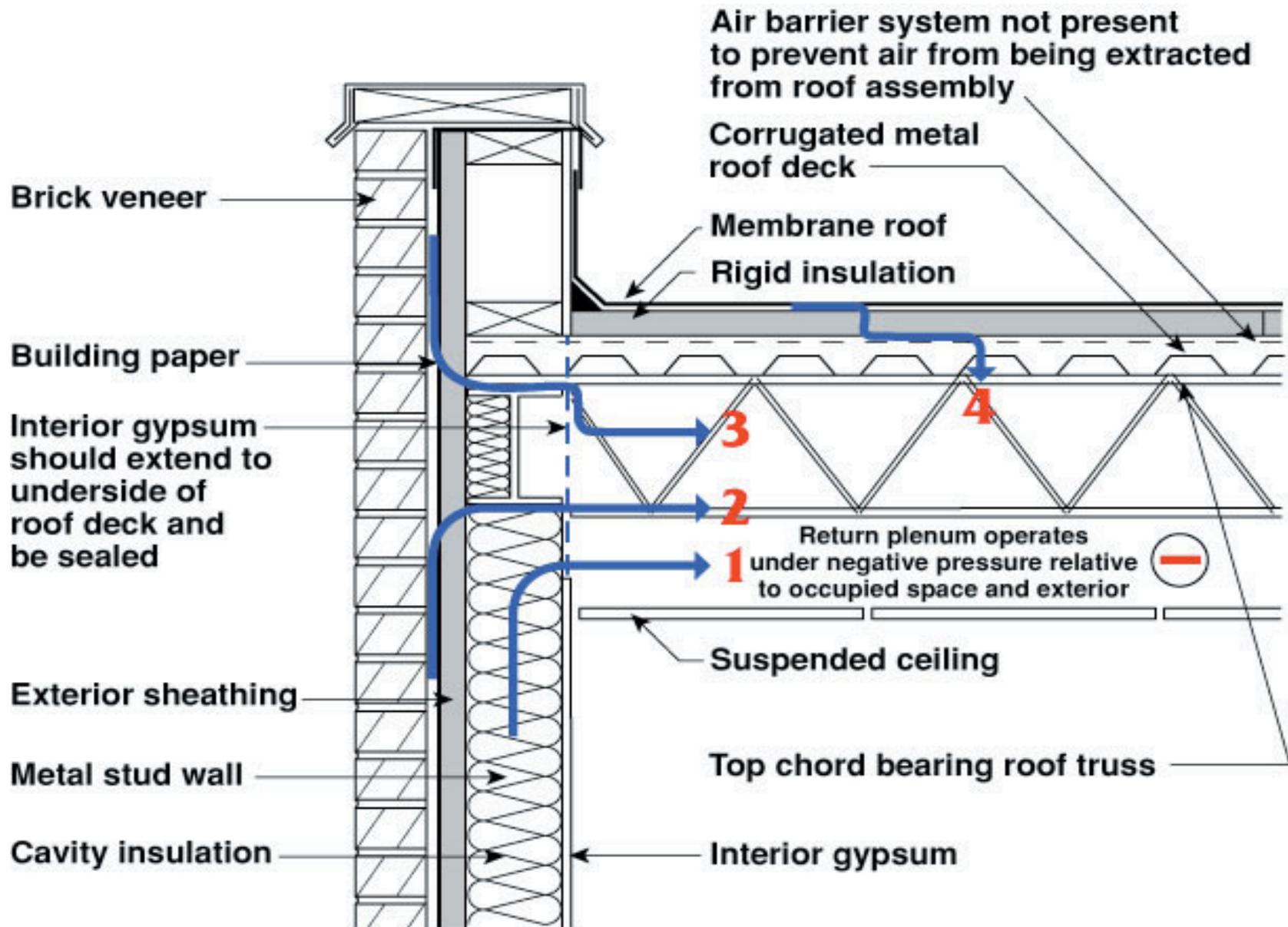
Figure 3.5
Air Conveyance System Air Pressure Field
 (from Sauer & Howell, 1990)

Supply air into occupied zone returns to AHU by passing through deliberately porous dropped ceiling or through return grilles installed in dropped ceiling

Air handling unit extracts air from dropped ceiling, conditions it and injects it into the occupied zones via supply ductwork

Dropped ceiling depressurized by air handling units extracting air from dropped ceiling











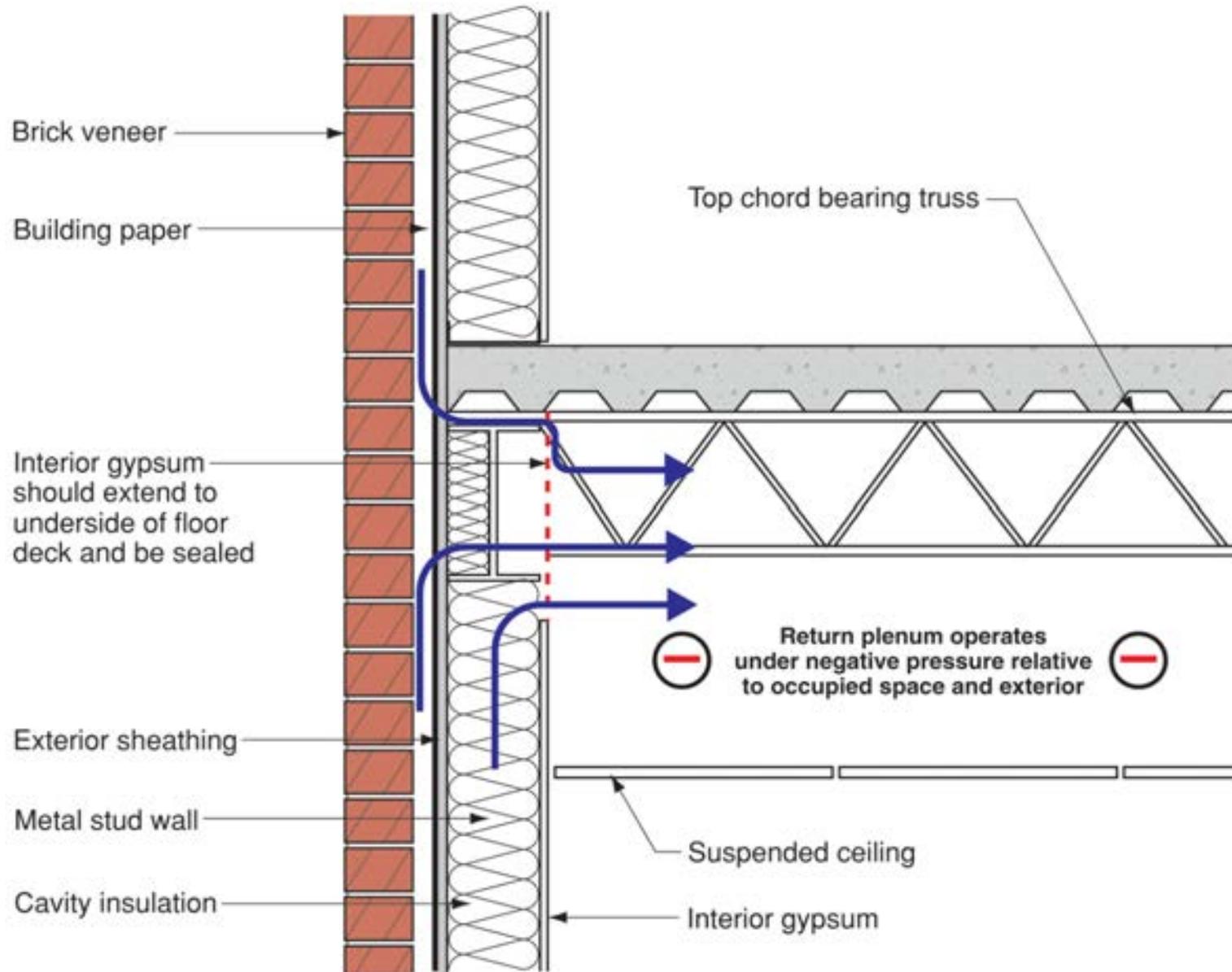














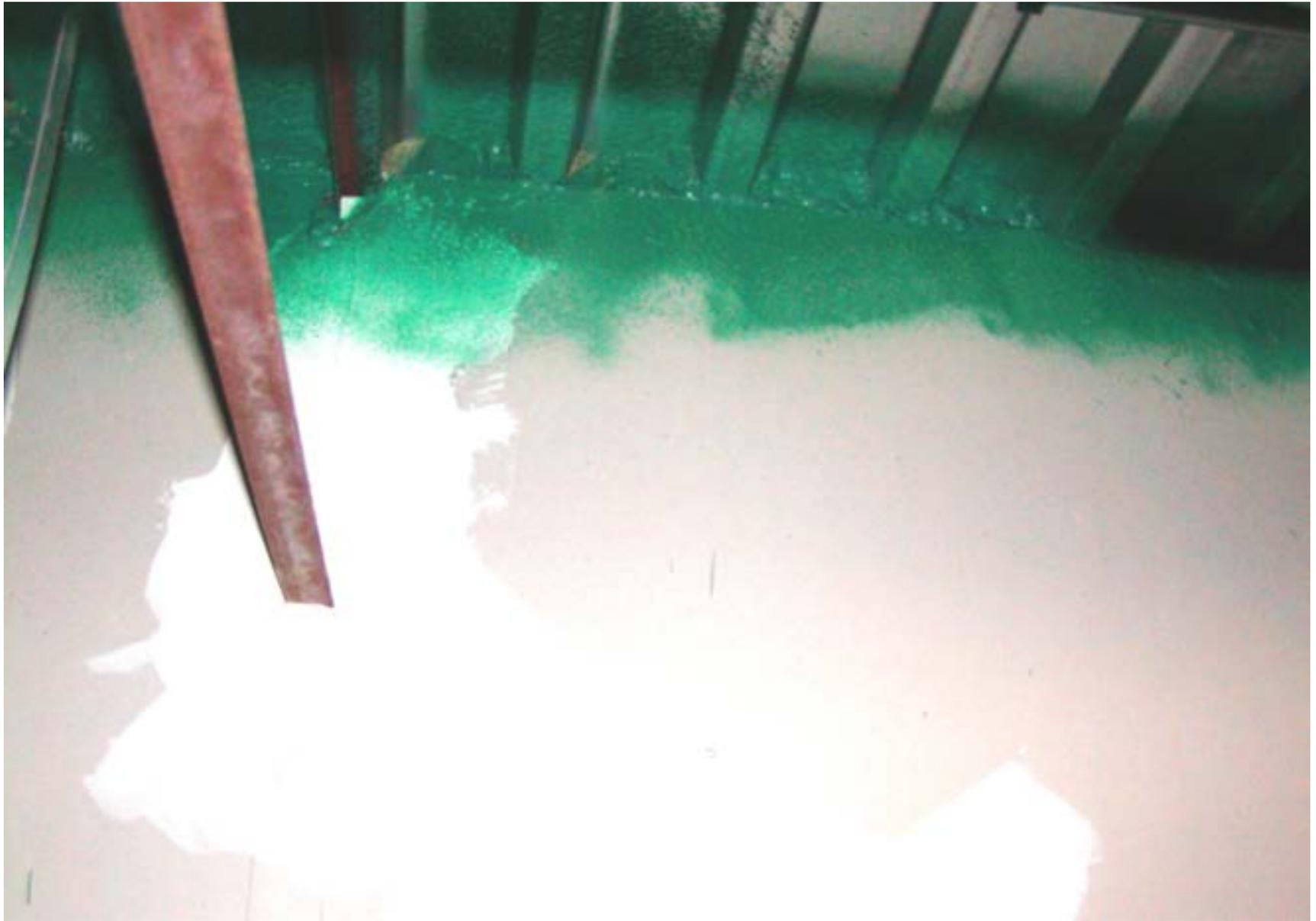




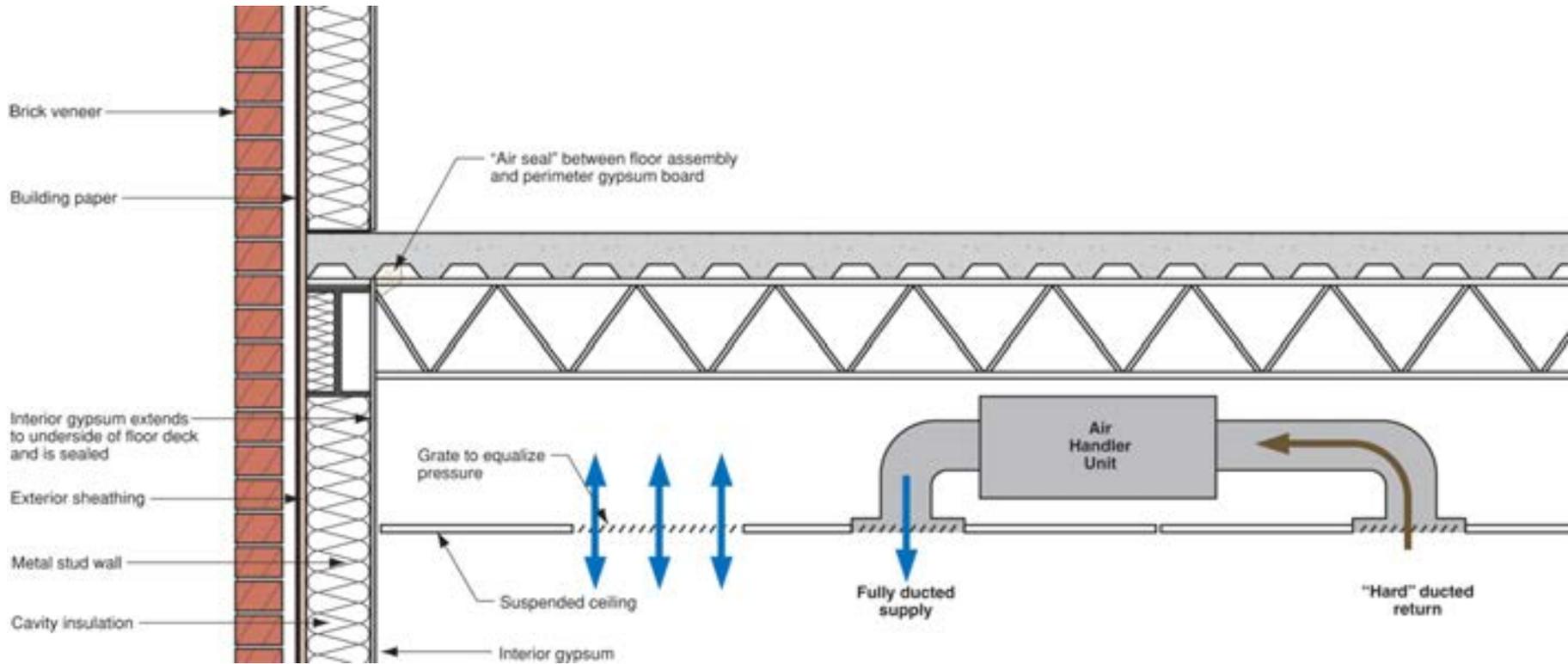












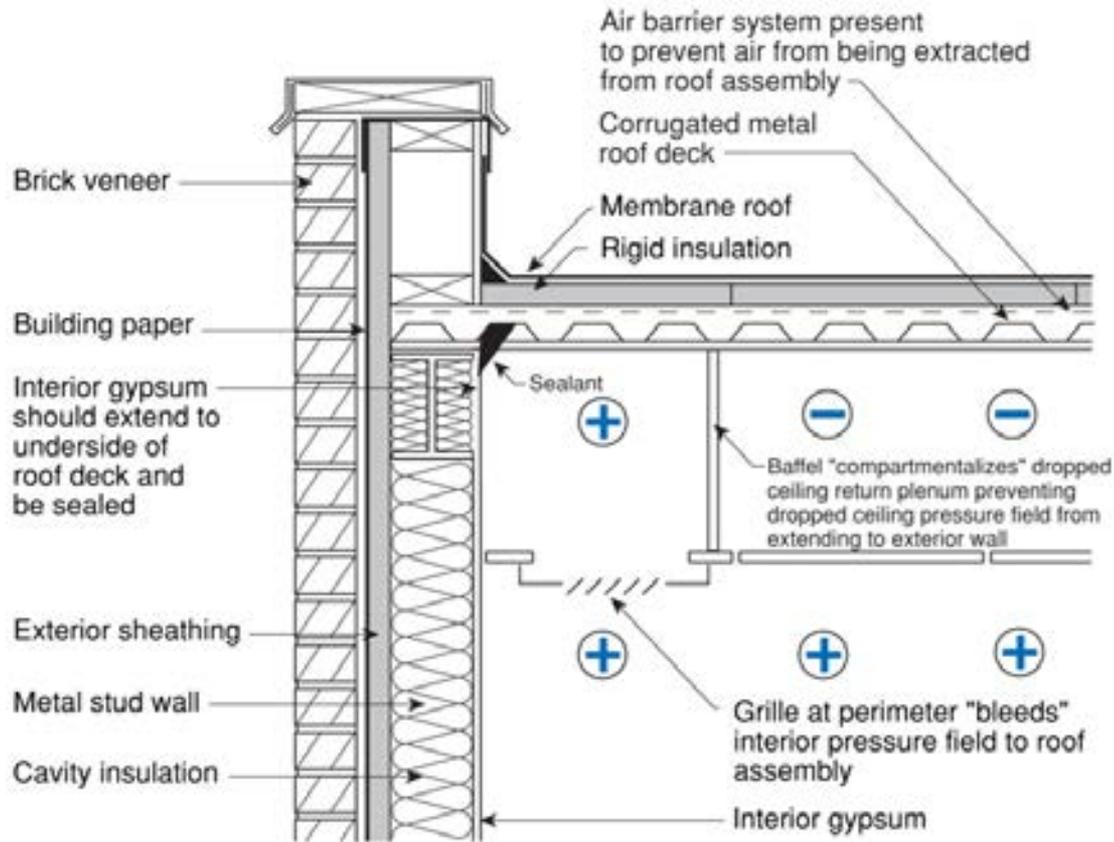
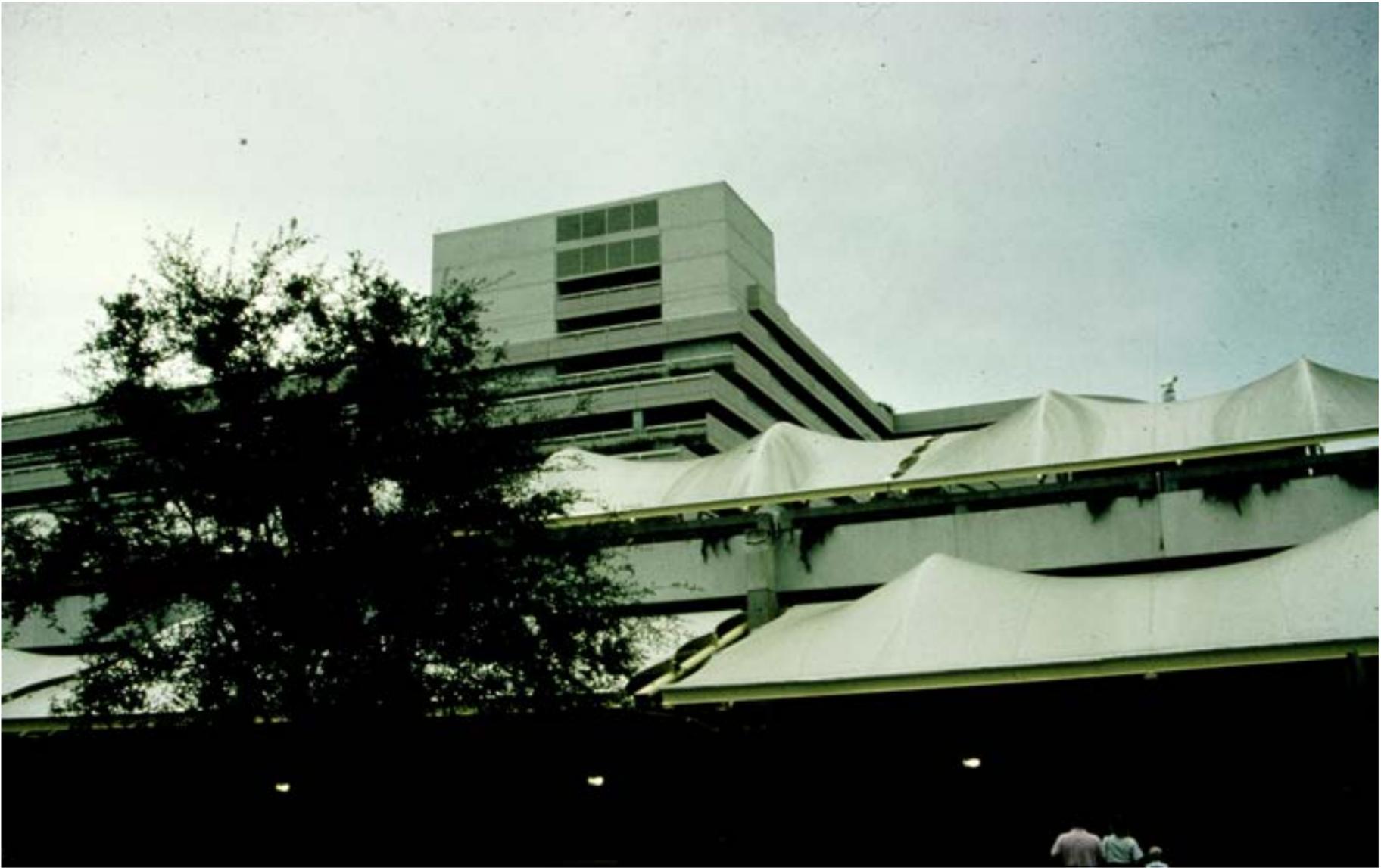


Figure 6.1
Compartmentalizing Dropped Ceiling











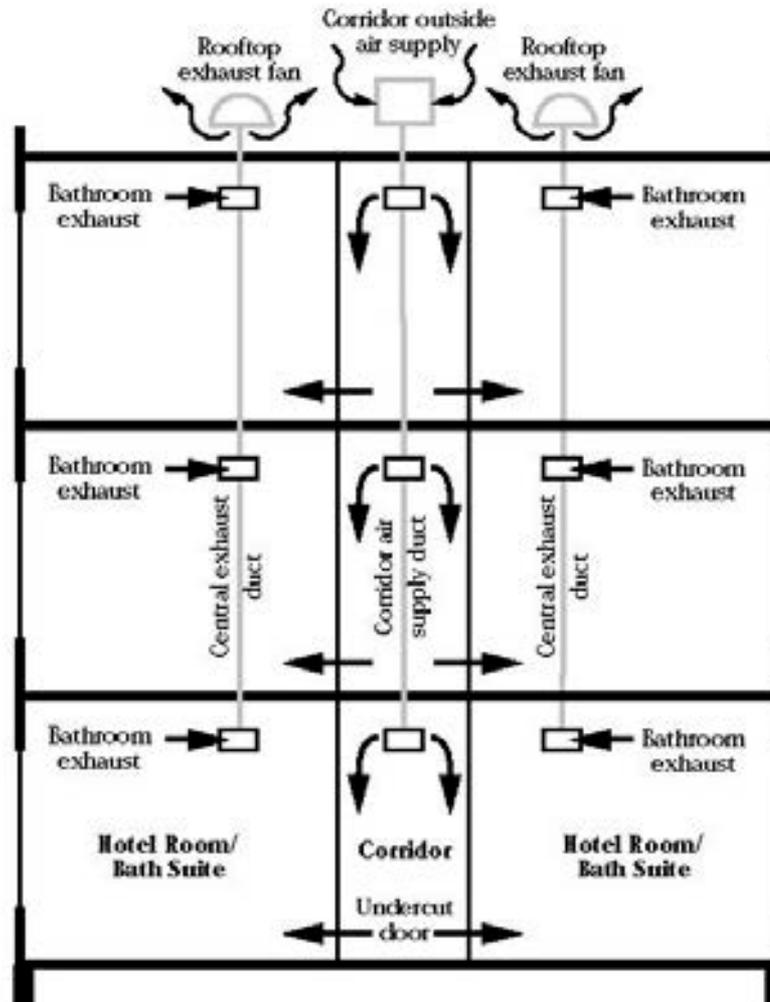


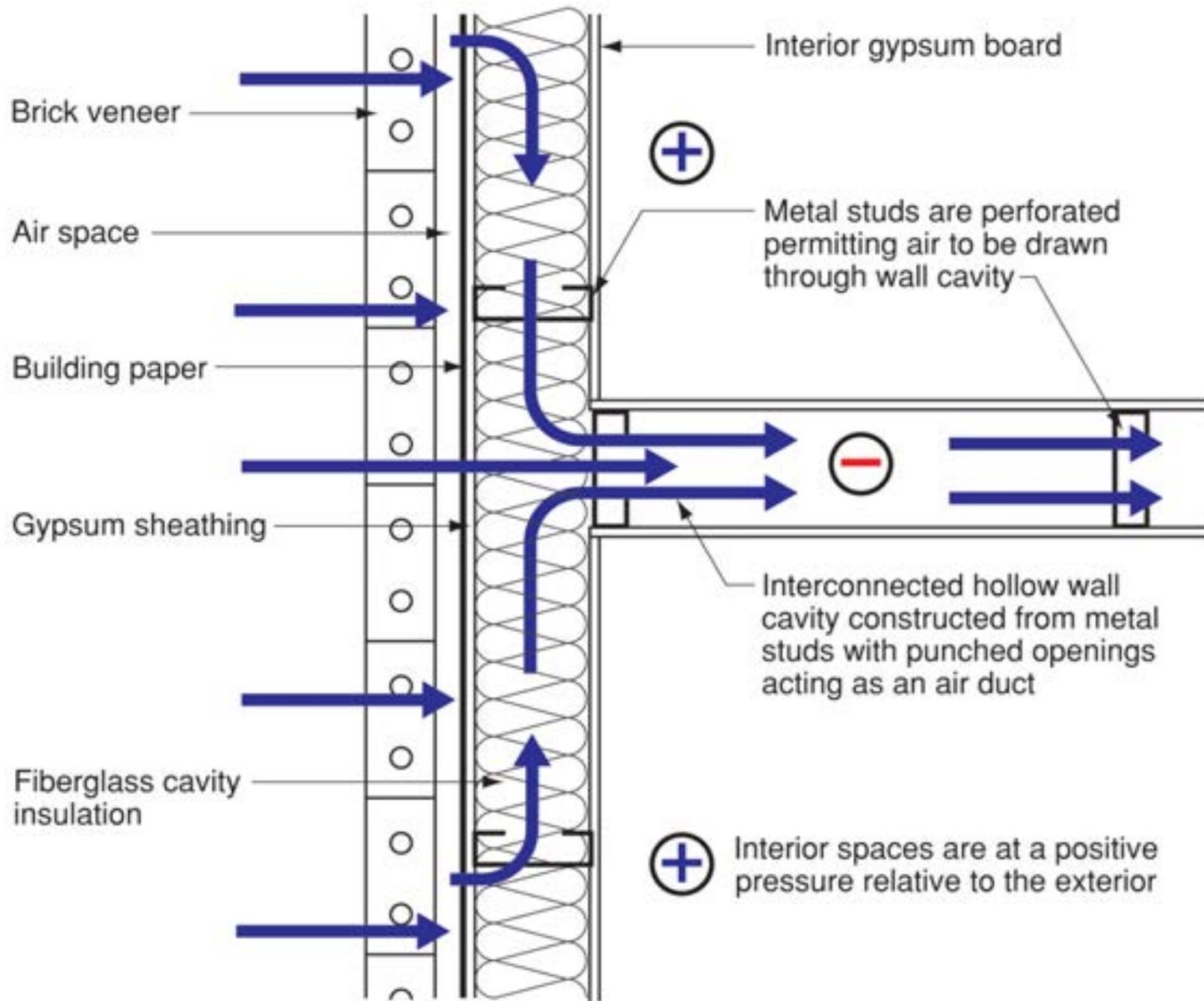
Figure 3.8

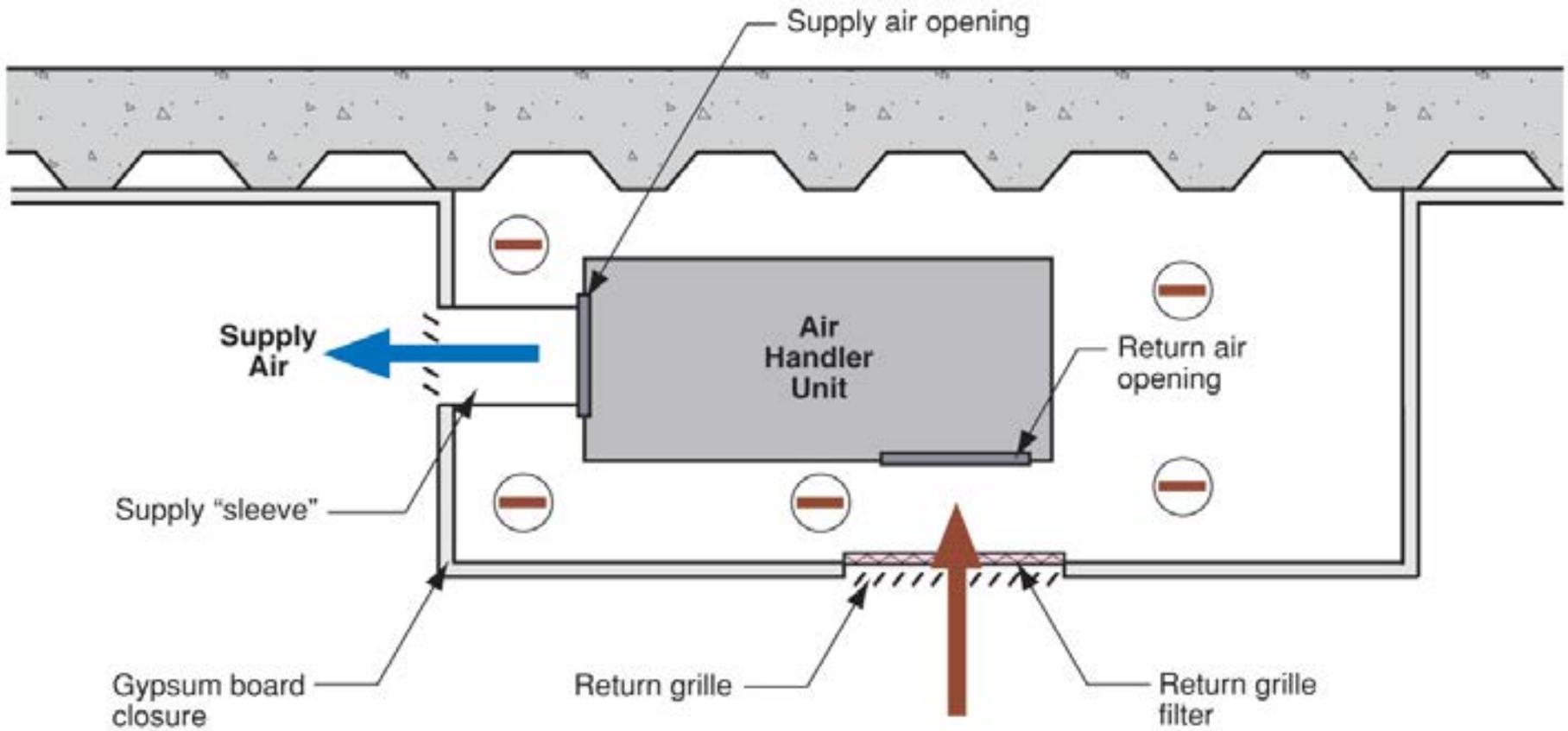
Hotel HVAC System

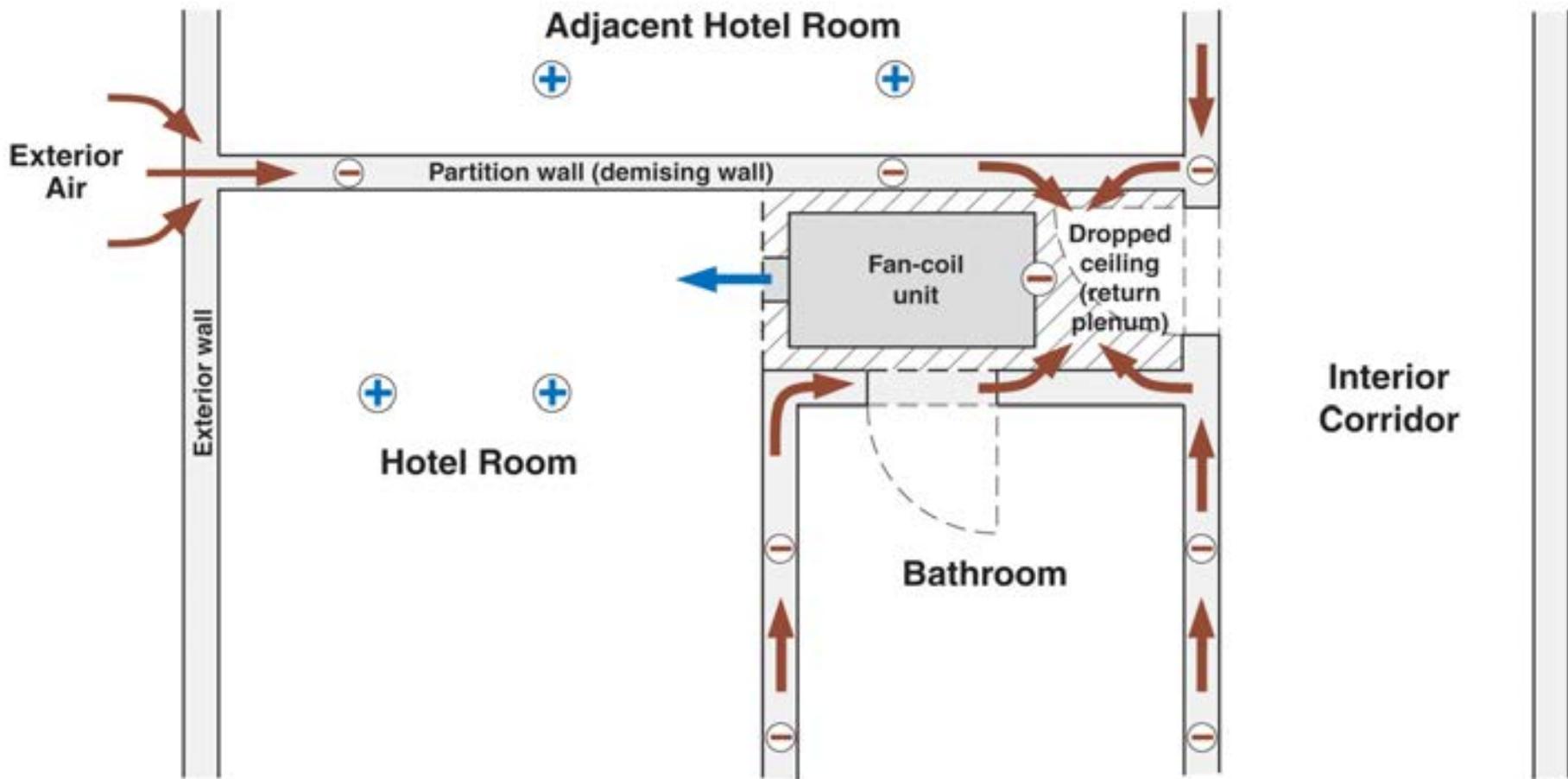
- Air exhausted from bathrooms via central rooftop exhaust fans
- Air supplied from corridors via undercut doors











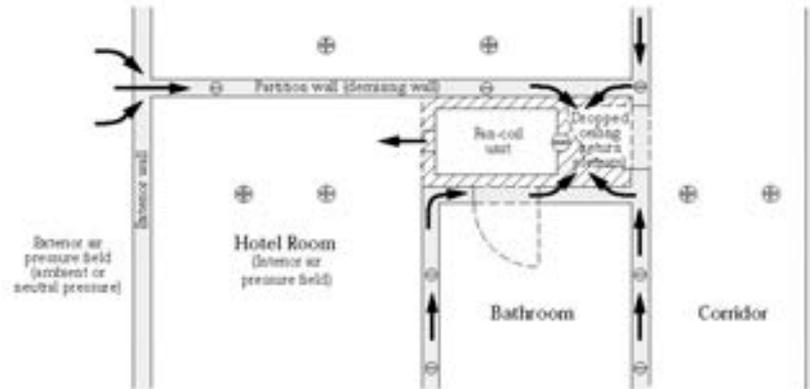


Figure 3.10
Pressure Field Due to Fan-Coil Unit
Plan View

- Room is at positive air pressure relative to exterior-driven air from corridor and air supplied to room from fan-coil unit pulling air from exterior through the demising wall
- Fan-coil unit depressurizes dropped ceiling assembly due to return plenum design
- Demising wall cavity pulled negative due to connection to dropped ceiling return plenum

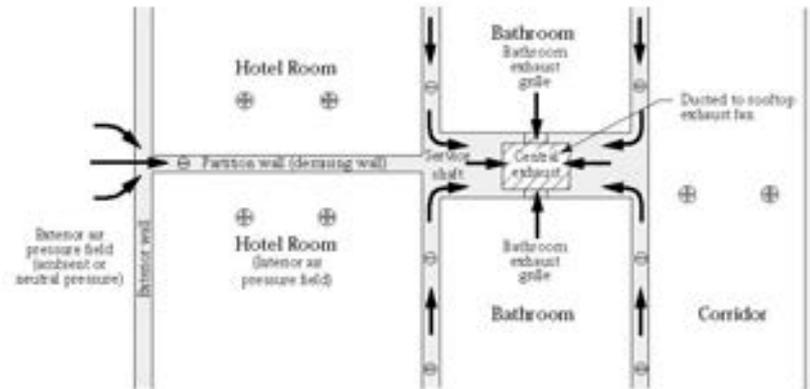


Figure 3.11
Pressure Field Due to Central Exhaust
Plan View

- Leakage of central exhaust duct pulls air out of service shaft depressurizing shaft and demising walls









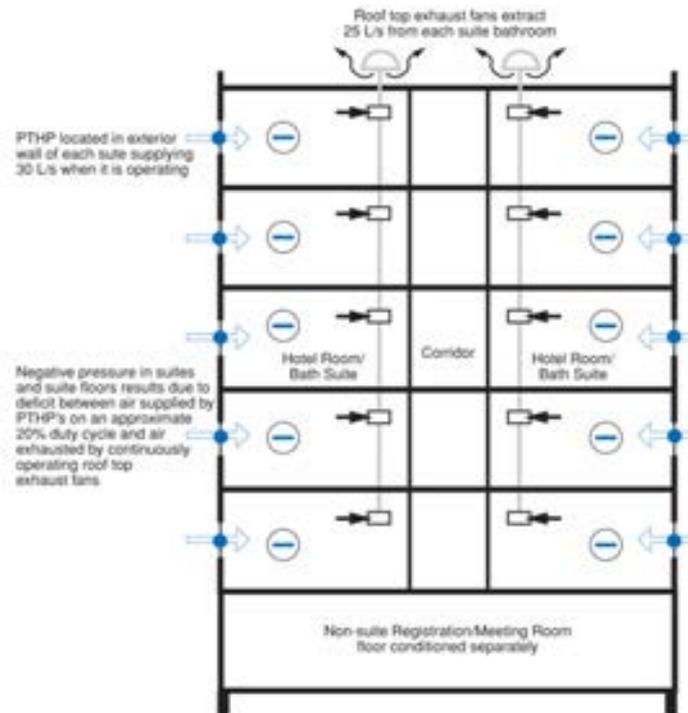
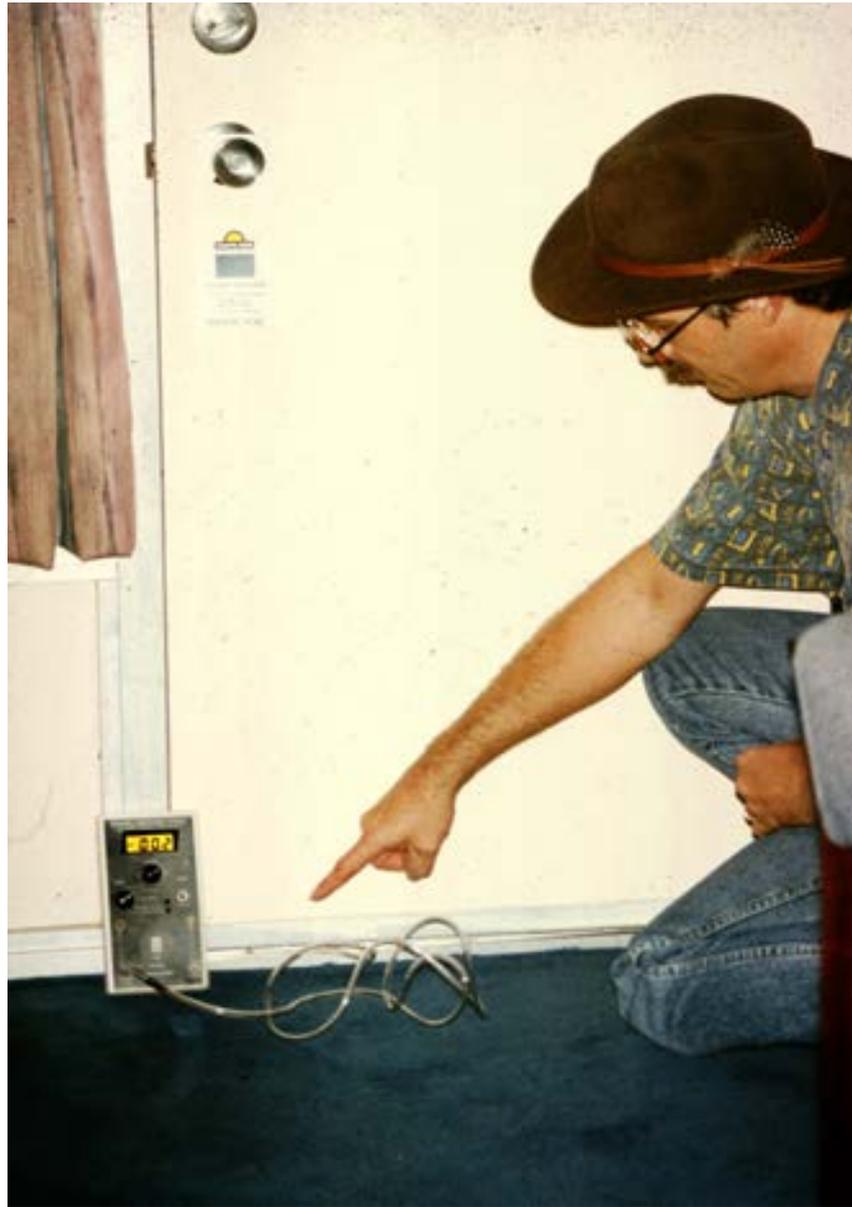


Figure 5.5

HVAC System for Hotel

- 25 L/s is extracted from each suite
- 15 suites per floor plus 100 L/s extracted from each corridor
- 475 L/s extracted per floor
- 2,850 L/s extracted from 6 floors with suites
- Each suite's PTHP supplies 30 L/s when it is operating. One additional PTHP serves each corridor supplying 100 L/s of outside air. A total of 550 L/s is supplied per floor when all the PTHP's on a floor are operating.
- However, the typical duty cycle of a PTHP is approximately 20%, i.e. 80% of the units are off at any one time.
- When 3 suite PTHP's and the corridor PTHP are operating only 190 L/s supplied to a floor. If 475 L/s is extracted per floor, a deficit of 285 L/s exists per floor or 1,710 L/s for all the suite floors combined.













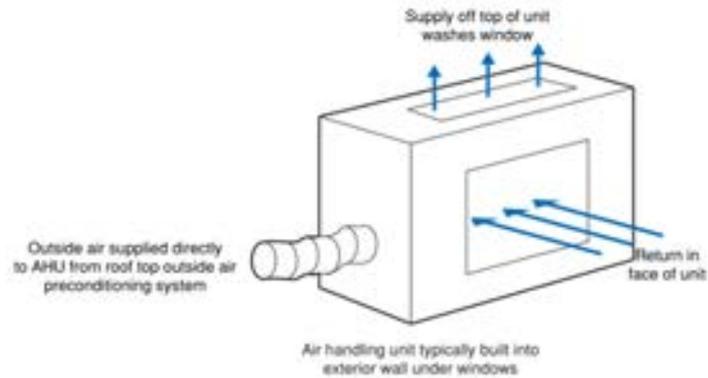
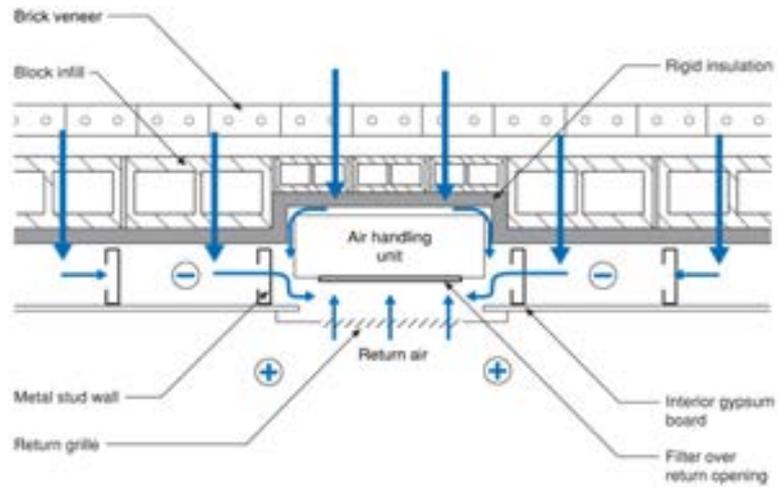


Figure 3.23
**AHU Depressurizing Exterior Wall Assembly
 Plan View**

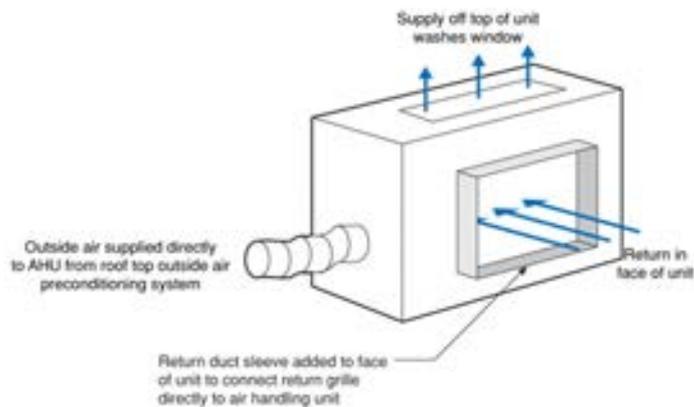
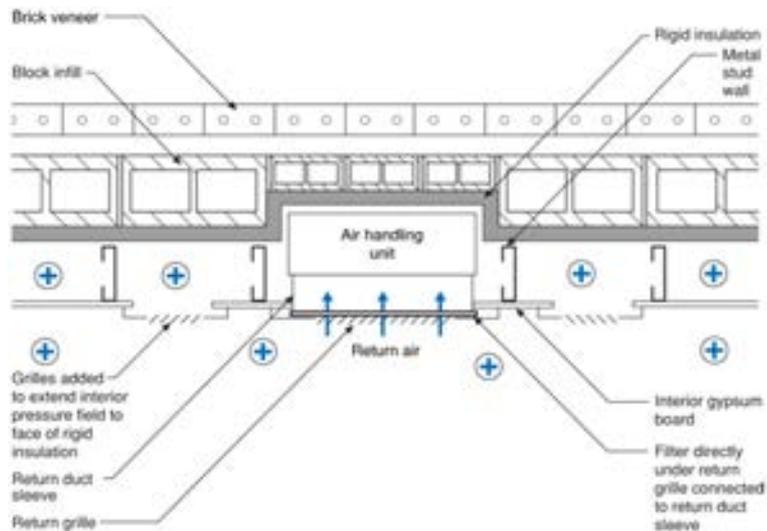


Figure 6.3
"Bleeding" Pressure Fields



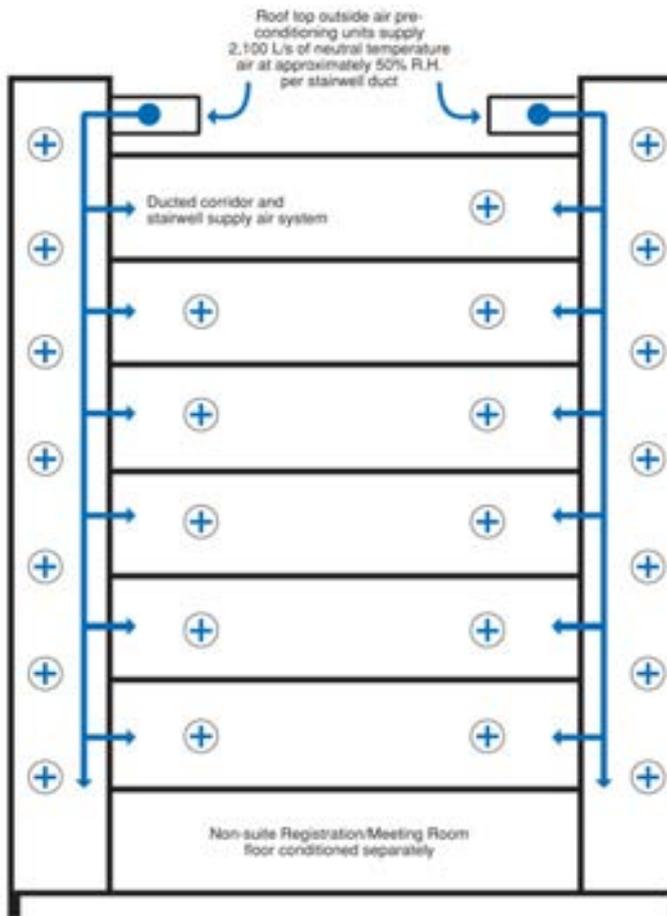


Figure 5.7

New Air Pressure Relationships

- Hotel suite floors supplied with 4,200 L/s of preconditioned air
- Hotel suite floors are exhausted to a total of 2,850 L/s
- Surplus of 1350 L/s pressurizes suite floors
- Stairwell held open with magnetic latches.

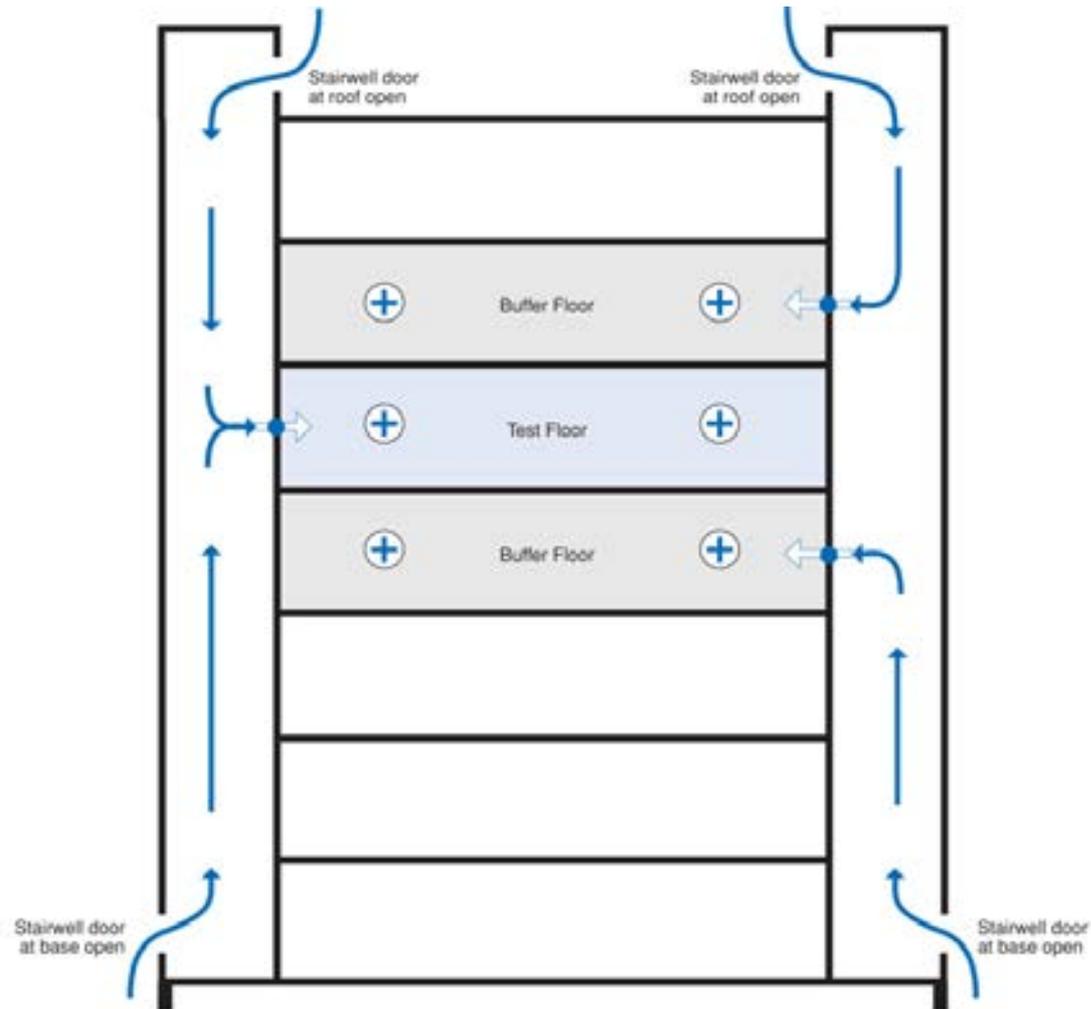


Figure 5.6
Air Leakage Test Zones







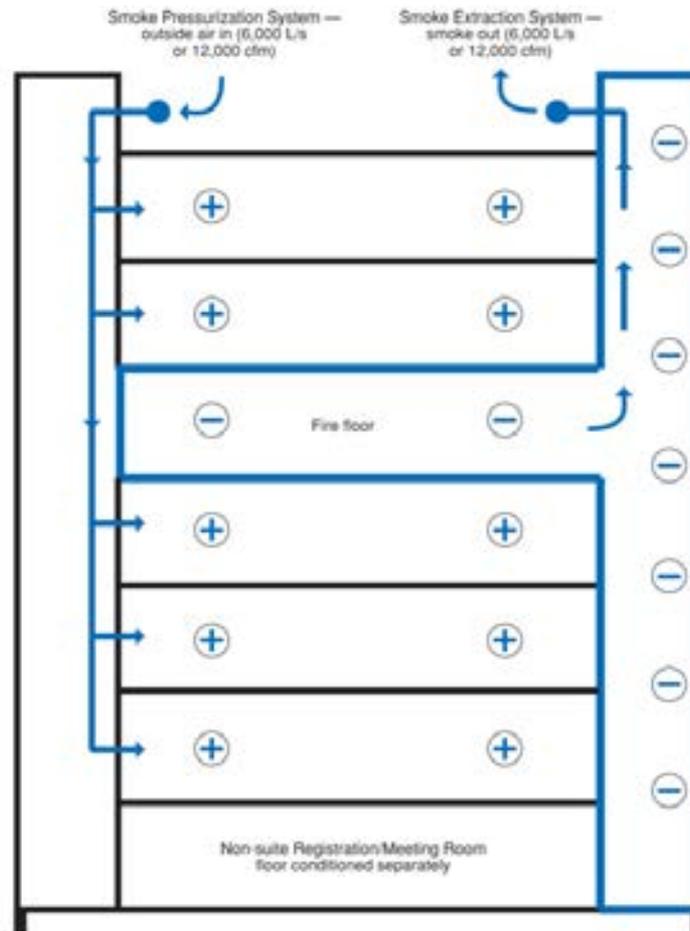


Figure 5.9

Smoke Extraction System

- If hotel is pressurized 25 Pa and smoke floor/floors are depressurized 25 Pa, net minimum smoke control pressure difference is greater than the design specified 25 Pa
- Approximately 1,000 L/s per floor is required to pressurized each floor 25 Pa relative to the exterior or approximately 6,000 L/s to pressurize the 6 hotel floors with suites when the roof top exhaust systems are not operating







Figure 3.12

Ductwork and Air Handlers in Basements

- No air pressure differences result in a house with an air handler and ductwork located in a basement if there are no leaks in the supply ducts, the return ducts or the air handler and if the amount of air delivered to each room equals the amount removed

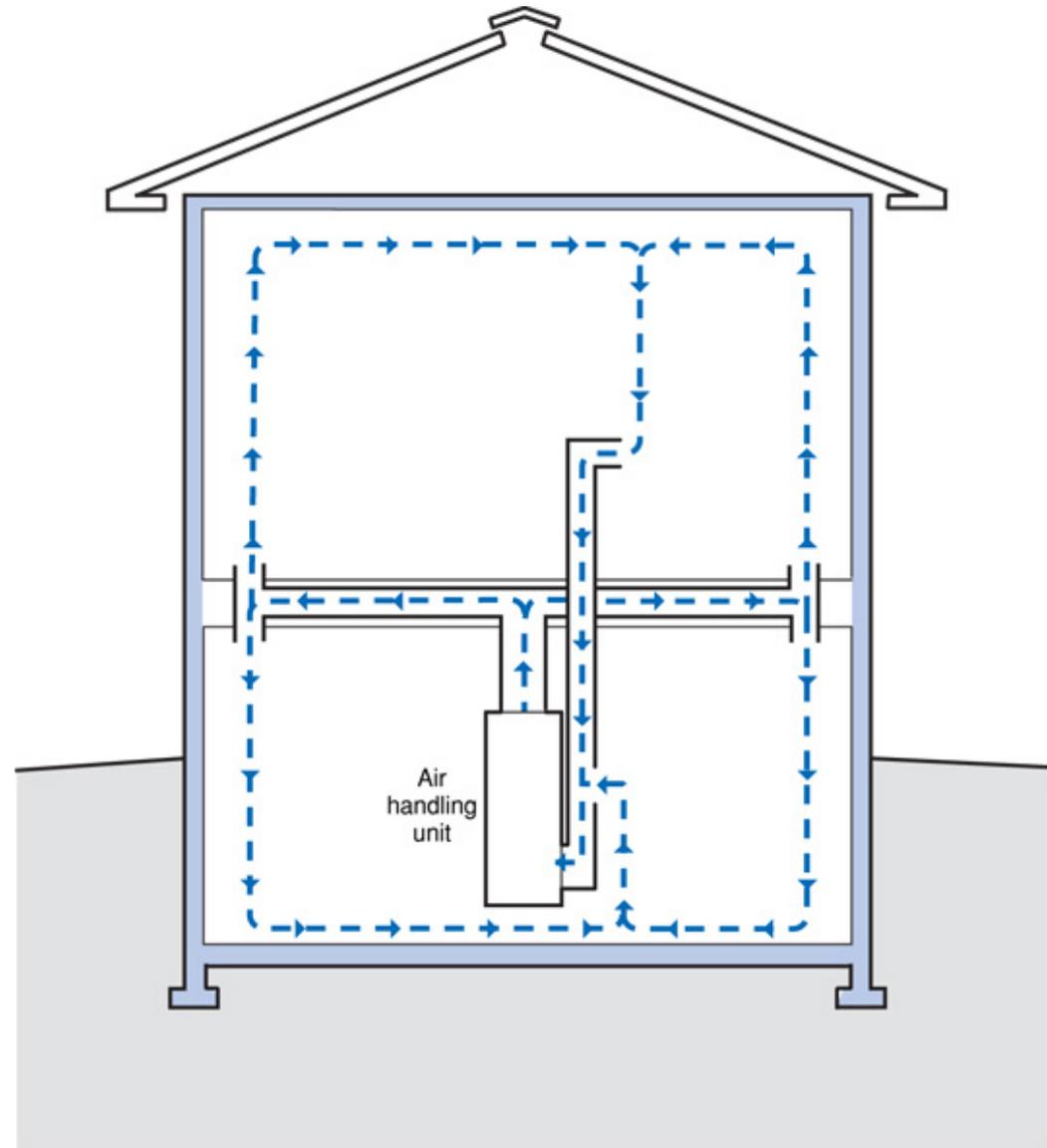
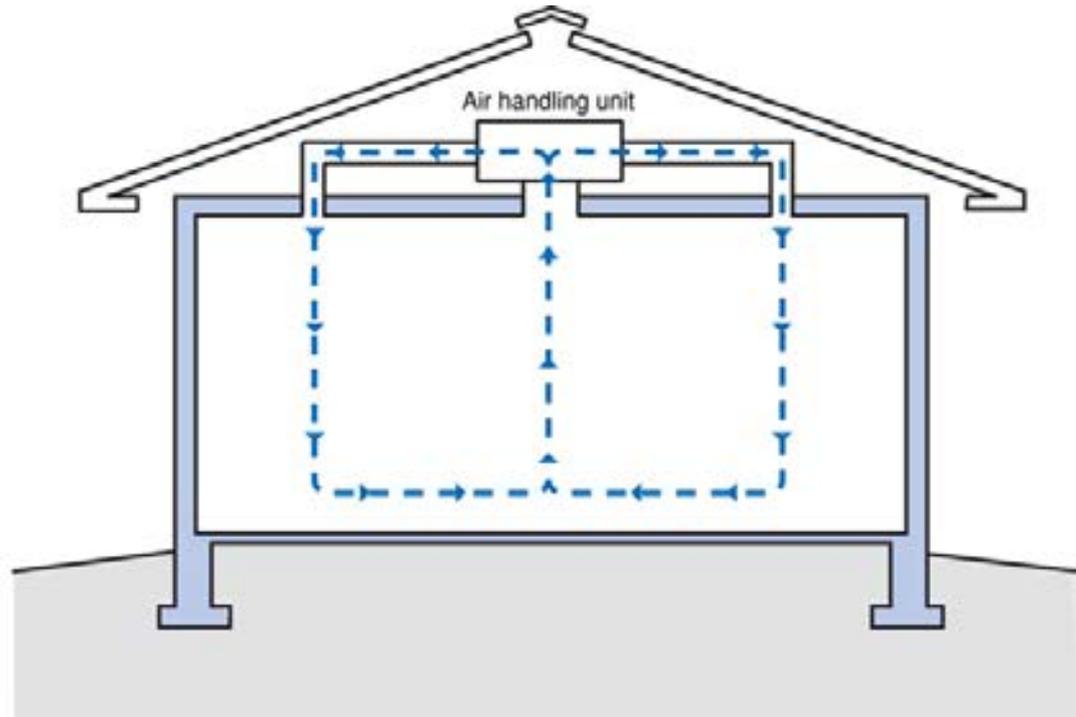


Figure 3.13

Ductwork and Air Handlers in Vented Attics

- No air pressure differences result in a house with an air handler and ductwork located in a vented attic if there are no leaks in the supply ducts, the return ducts or the air handler and if the amount of air delivered to each room equals the amount removed



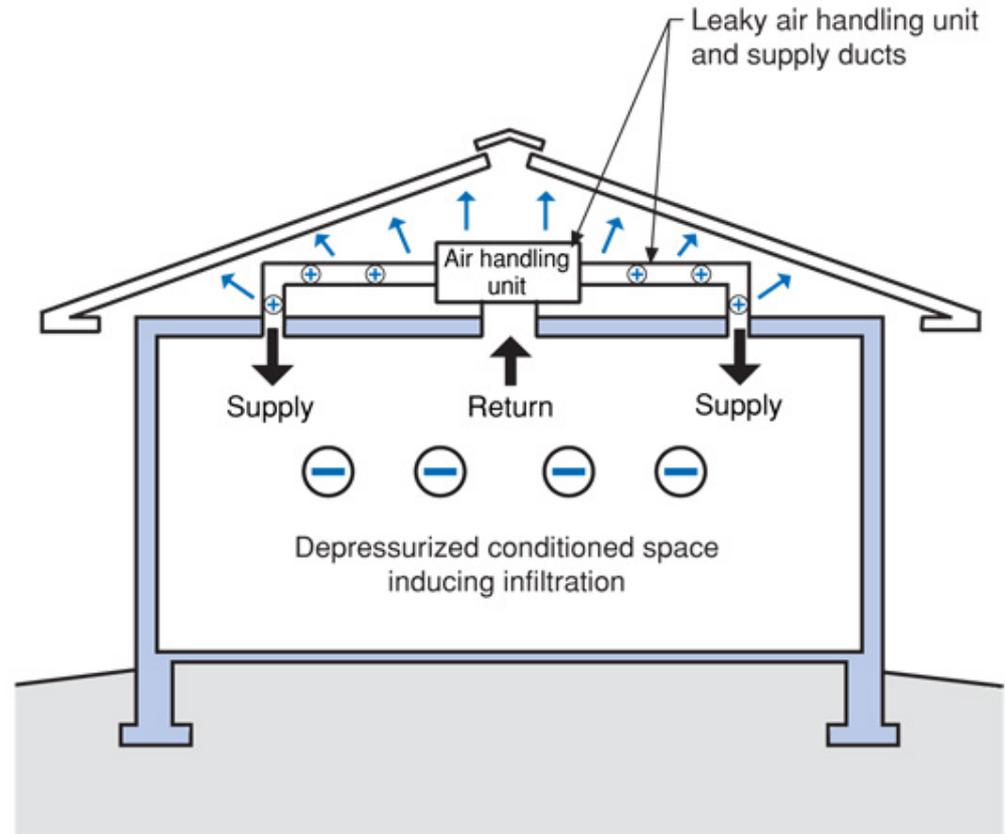


Figure 3.15

Leaky Ductwork and Air Handlers in Vented Attics

- Supply ductwork and air handler leakage is typically 20% or more of the flow through the system





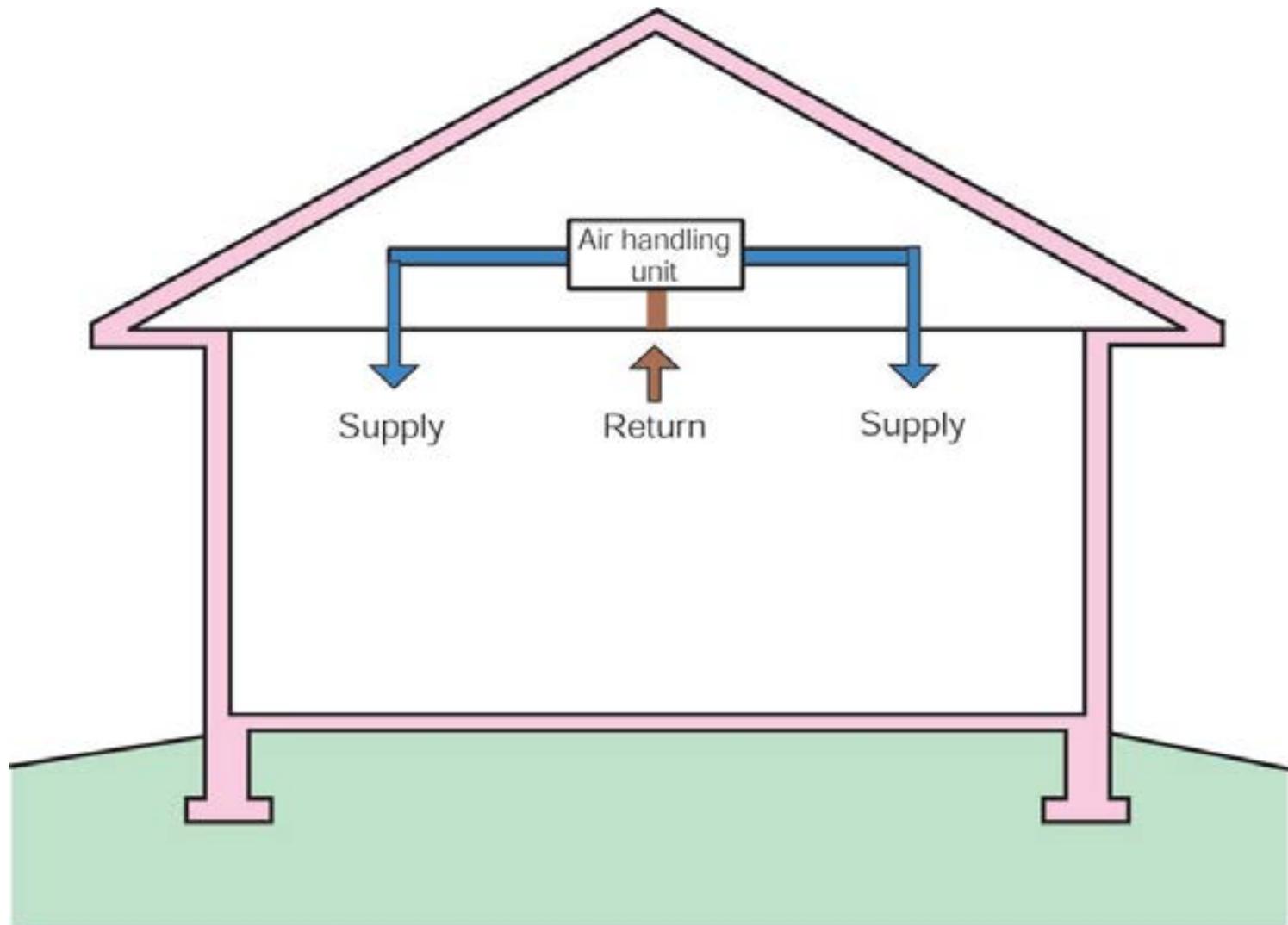








Duct Leakage Should Be Less Than 5% of Rated Flow As Tested By Pressurization To 25 Pascals



Note: Colored shading depicts the building's thermal barrier and pressure boundary. The thermal barrier and pressure boundary enclose the conditioned space.



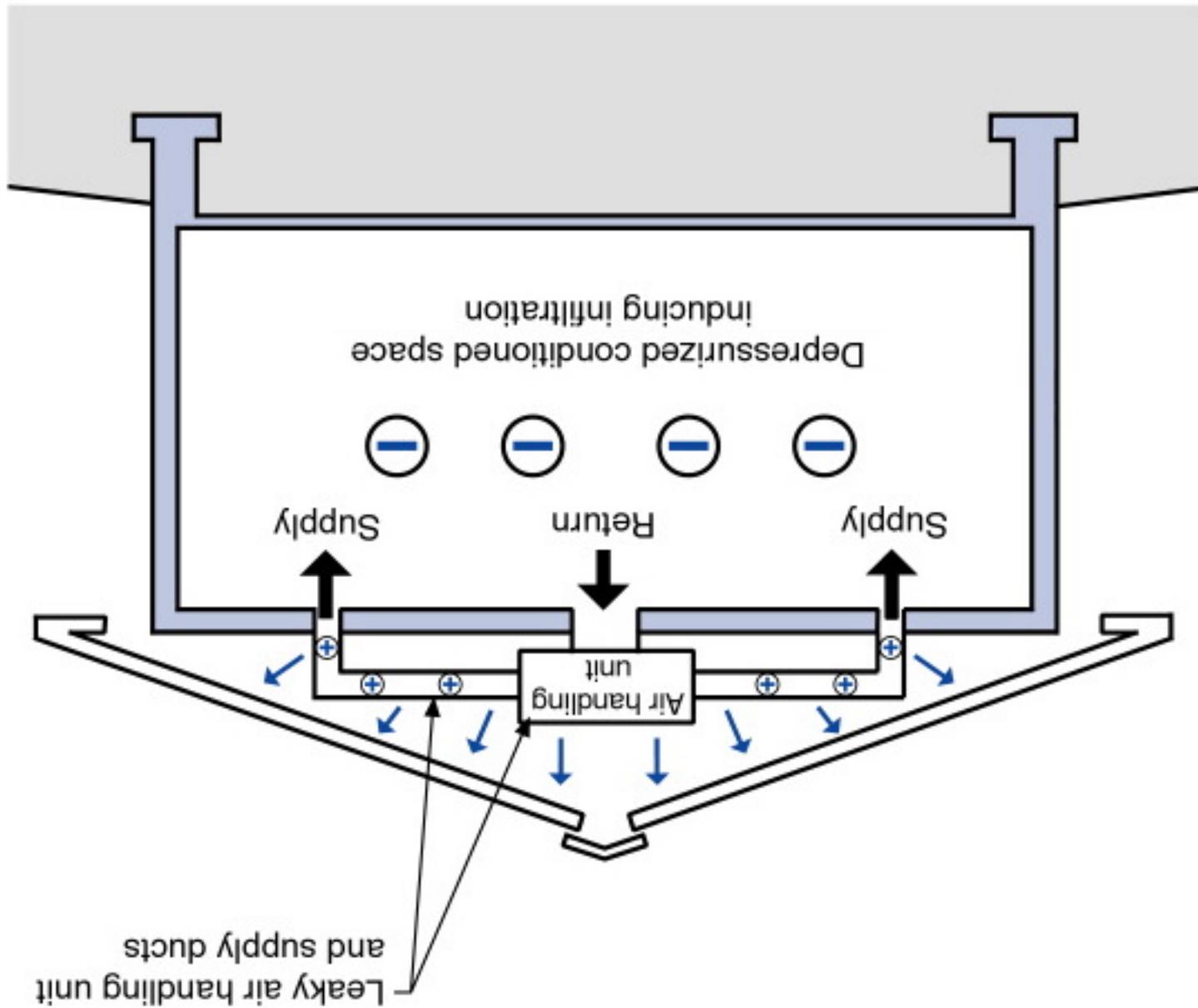
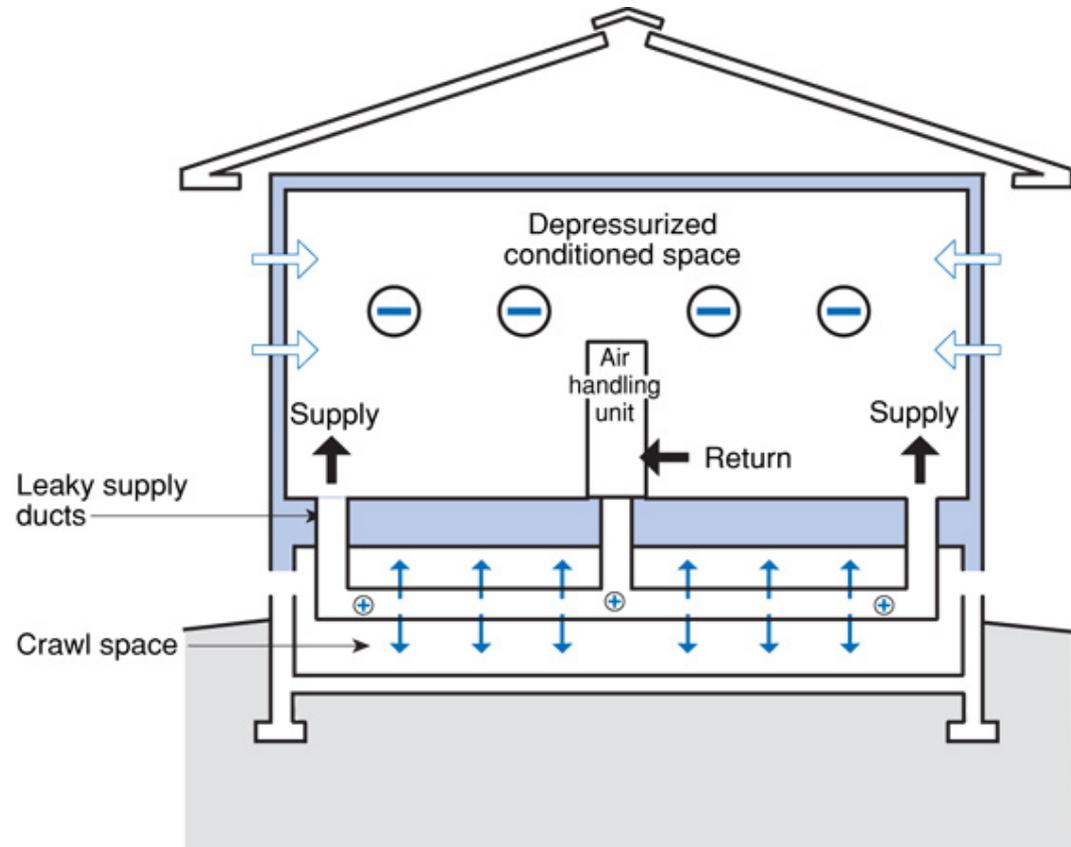


Figure 3.16

Leaky Supply Ductwork in Vented Crawl Space

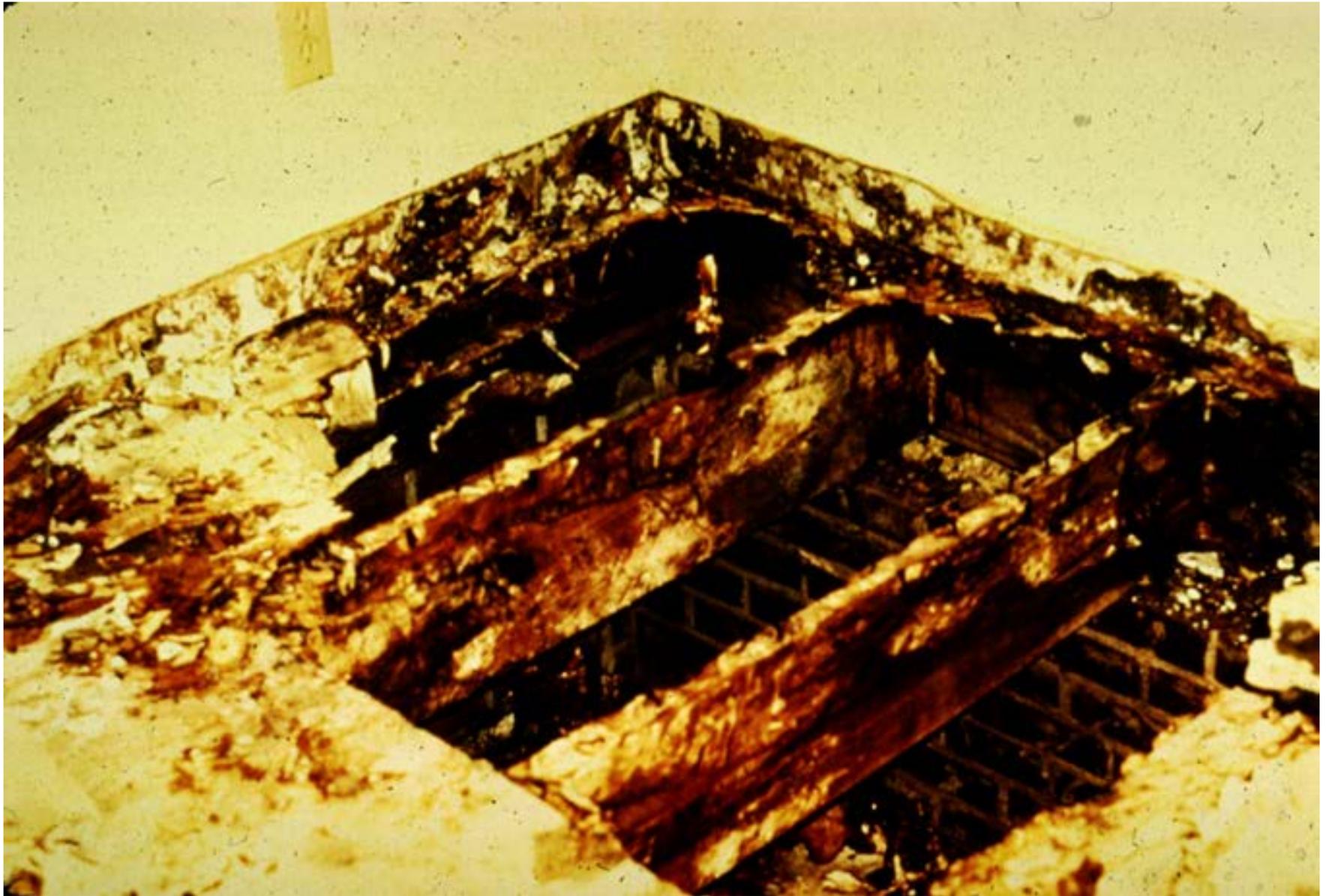
- Air pressurization pattern with mechanical system ducts in the crawl space















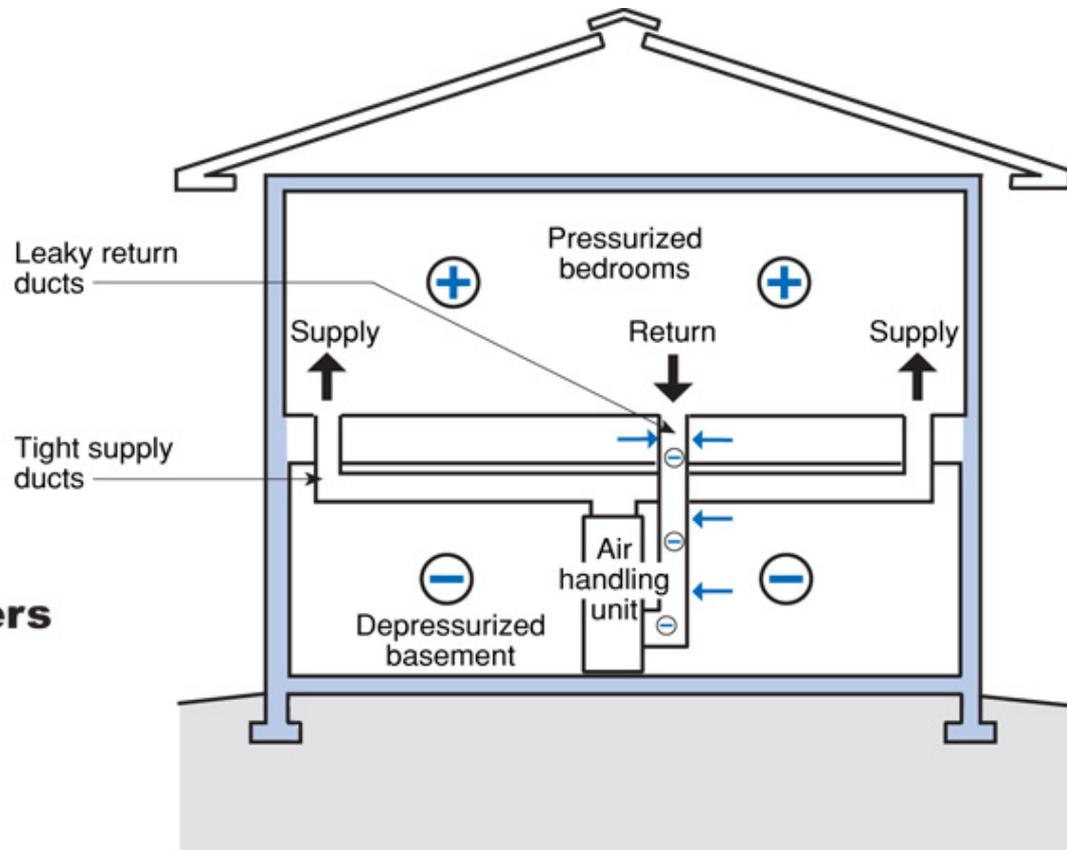


Figure 3.14
Leaky Ductwork and Air Handlers in Basements

- Air pressurization patterns in a house with leaky ductwork in the basement



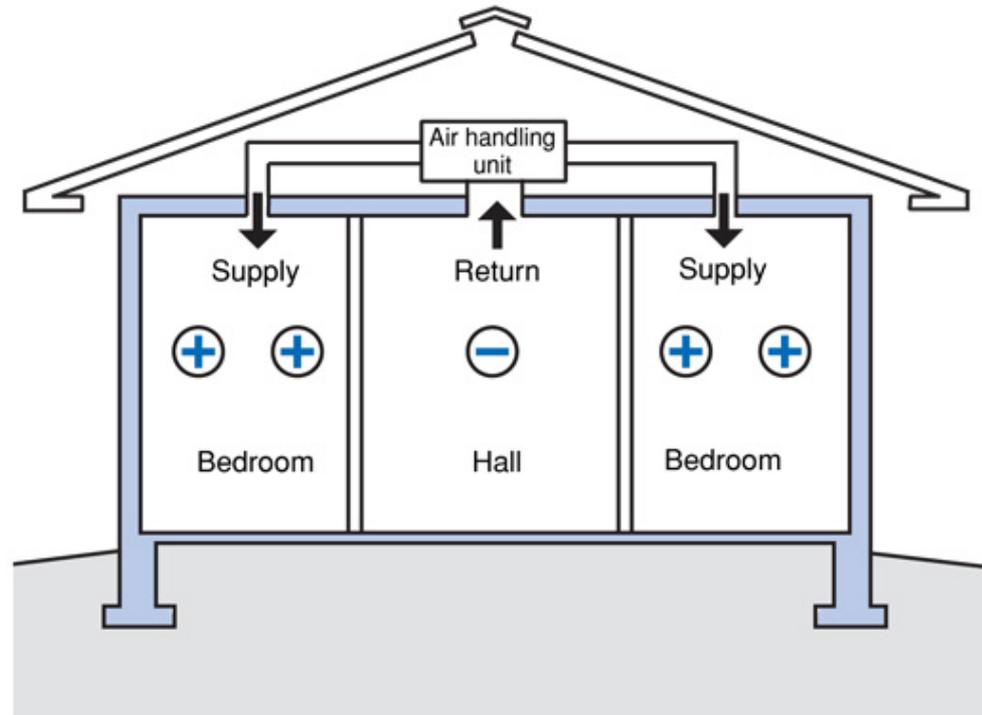


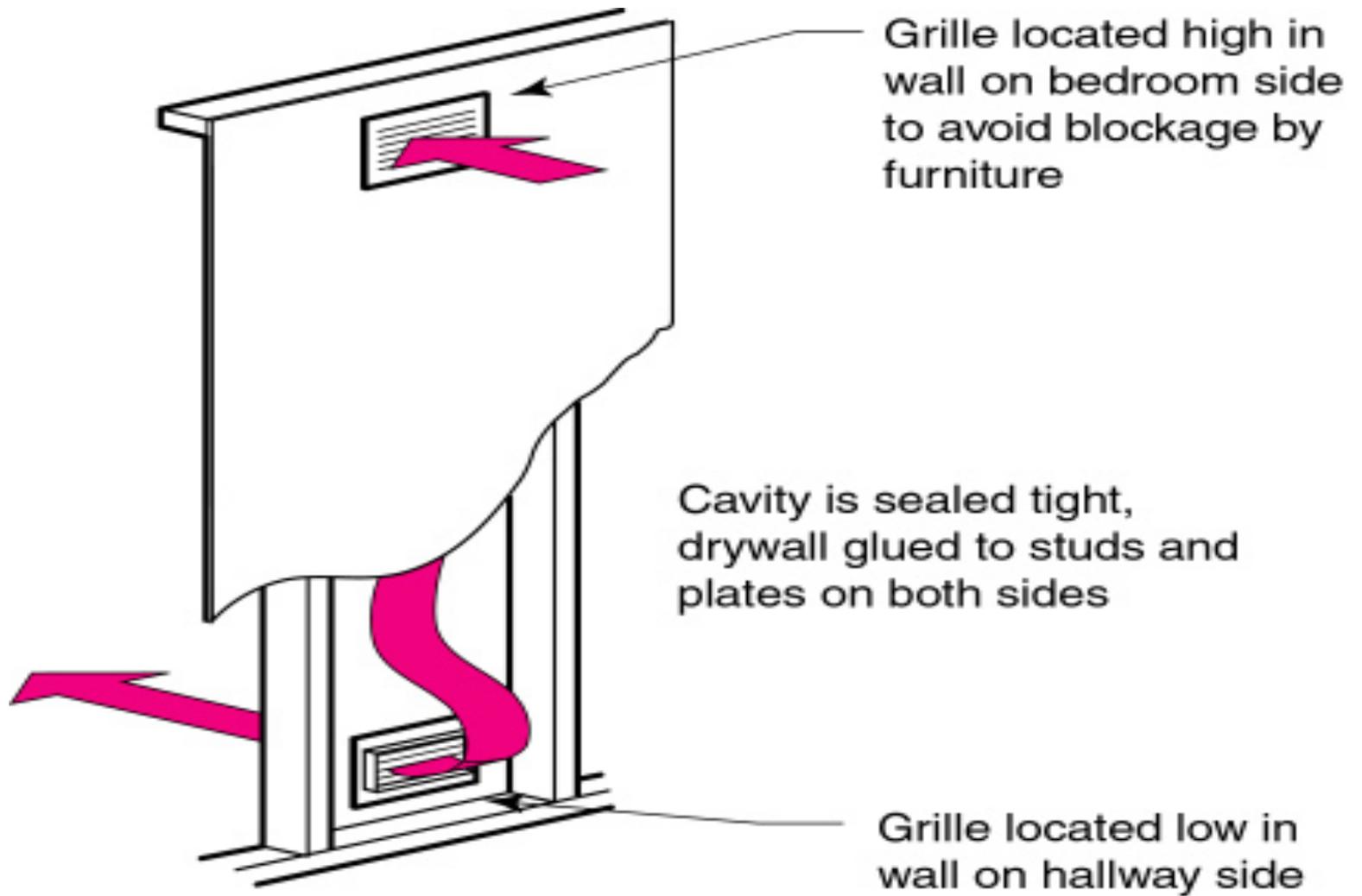


Figure 3.18

Insufficient Return Air Paths

- Pressurization of bedrooms often occurs if insufficient return pathways are provided; undercutting bedroom doors is usually insufficient; transfer grilles, jump ducts or fully ducted returns may be necessary to prevent pressurization of bedrooms
- Master bedroom suites are often the most pressurized as they typically receive the most supply air
- When bedrooms pressurized, common areas depressurize; this can have serious consequences when fireplaces are located in common areas and subsequently backdraft







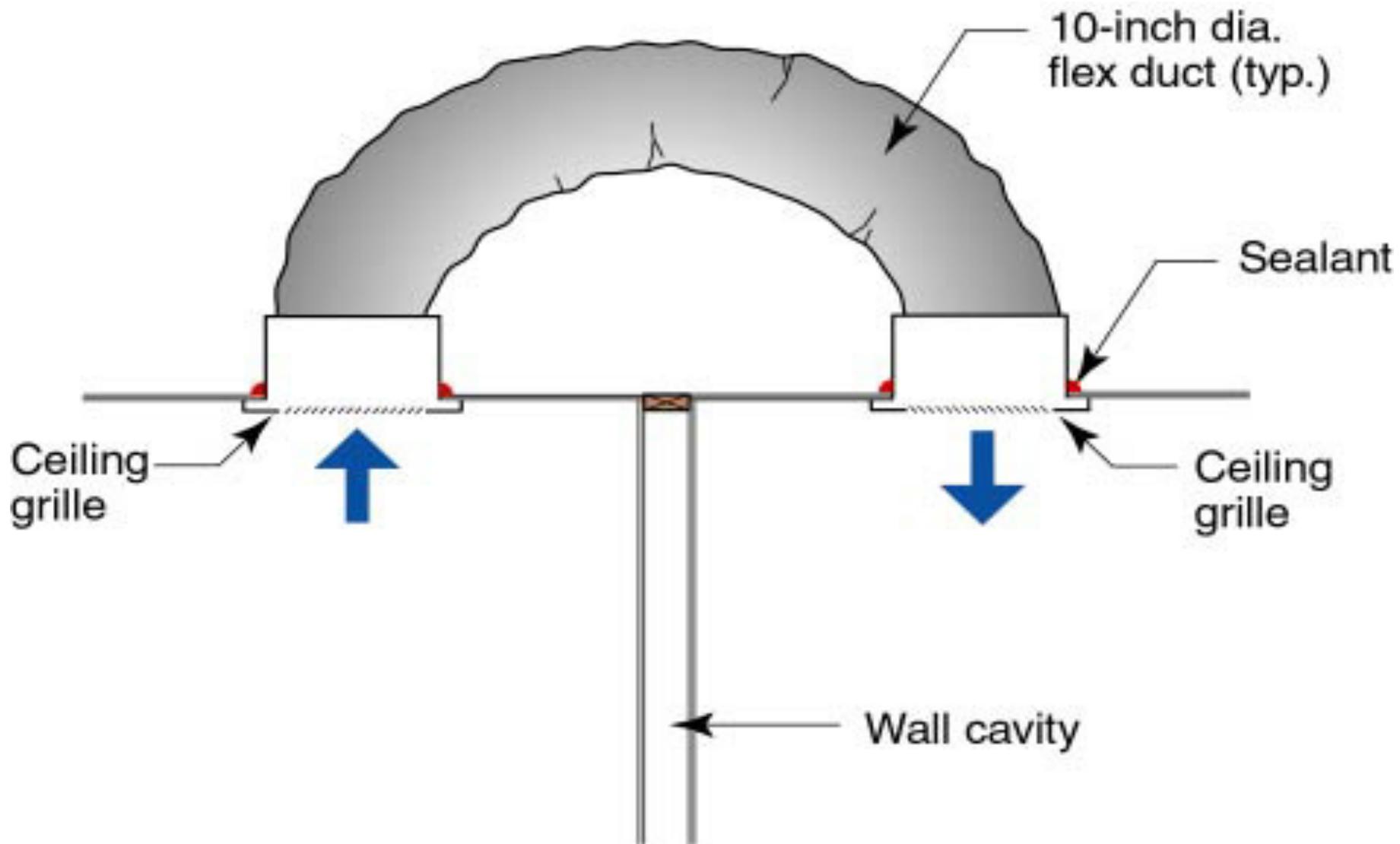




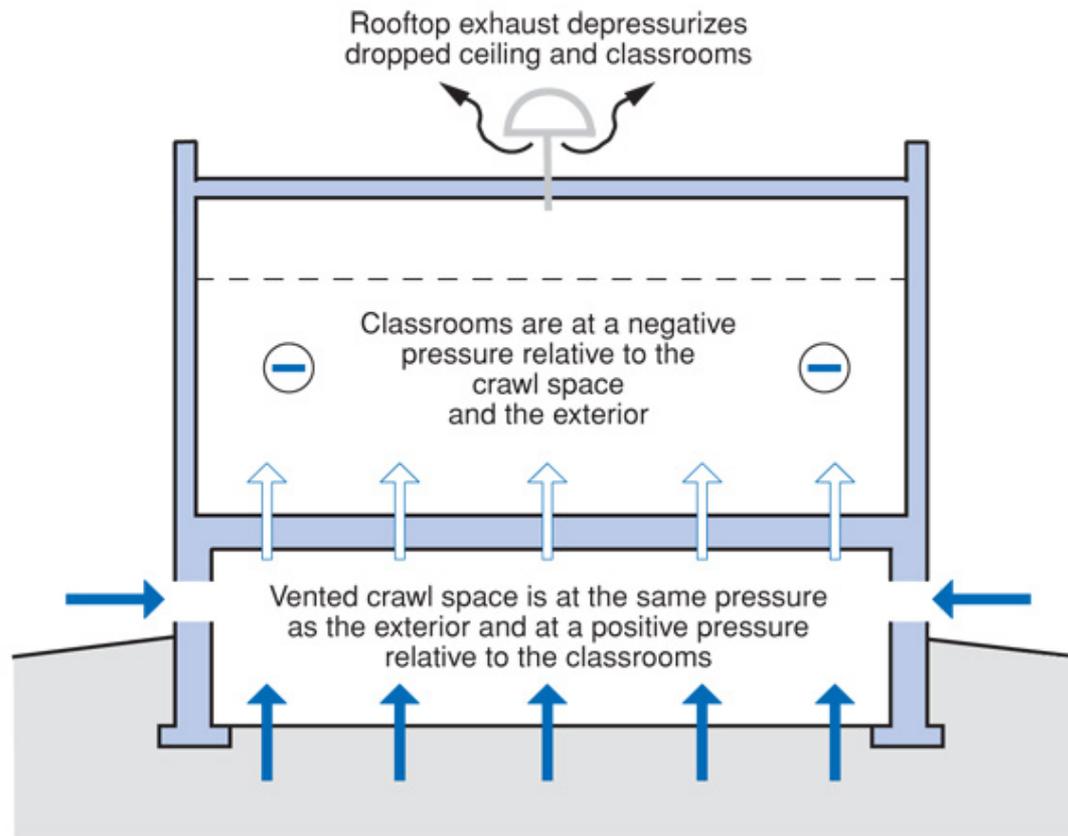






Figure 5.1
Problem Pressure Relationship

- The classrooms in this school operate at a negative pressure with respect to the crawl space



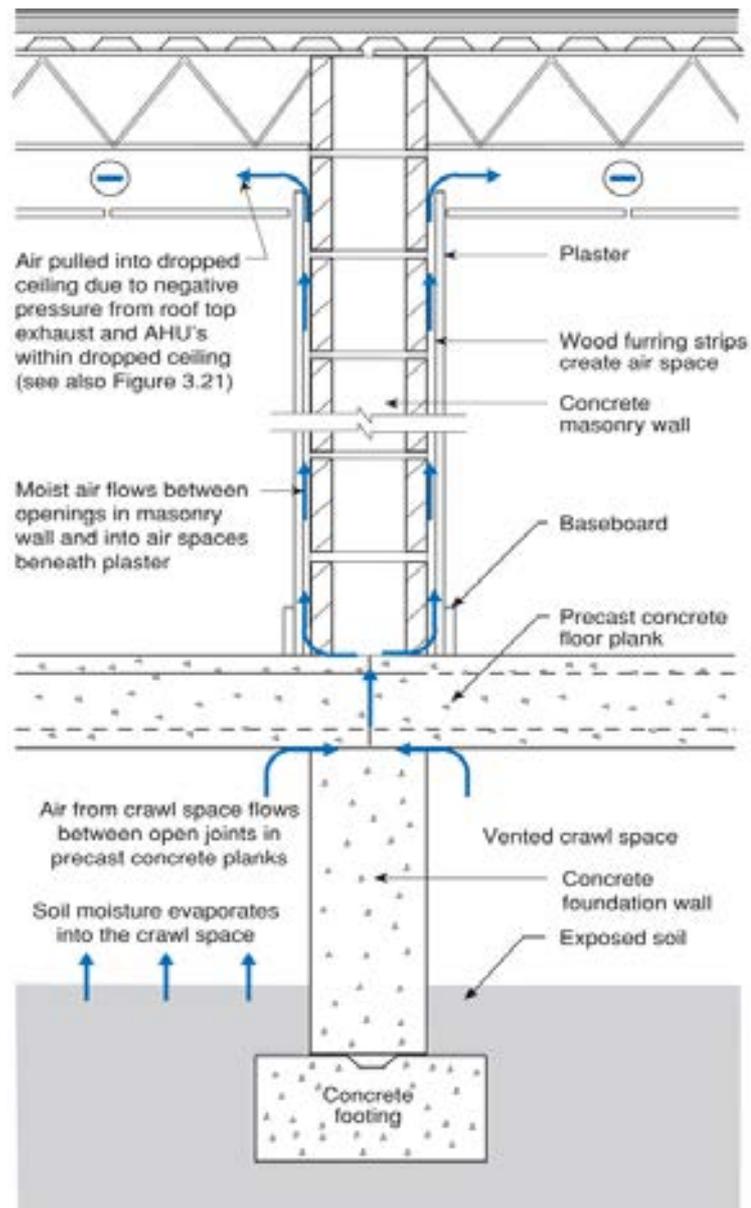


Figure 5.2

Moisture Movement

- This wall section illustrates moisture movement from the crawl space into the wall cavities and dropped ceiling

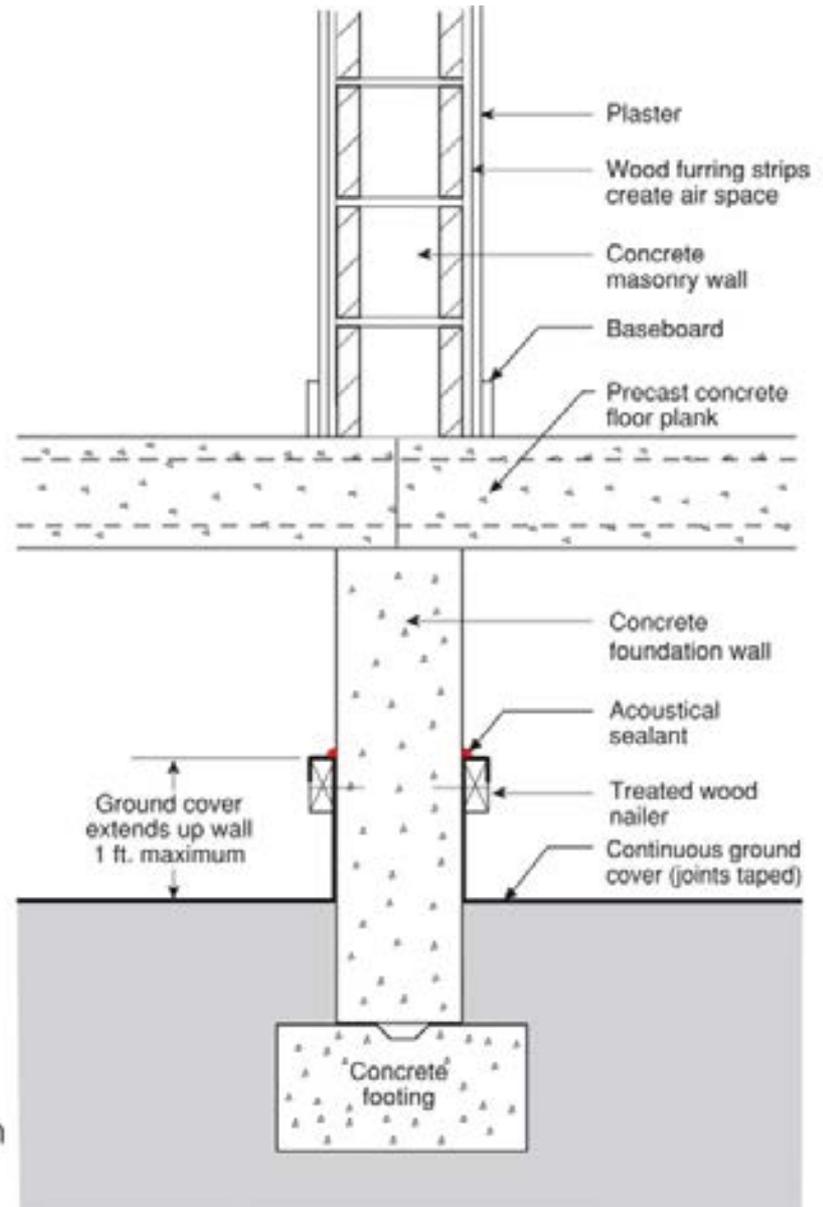


Figure 5.3

Ground Cover Installation

- This wall section illustrates proper installation of the polyethylene ground cover

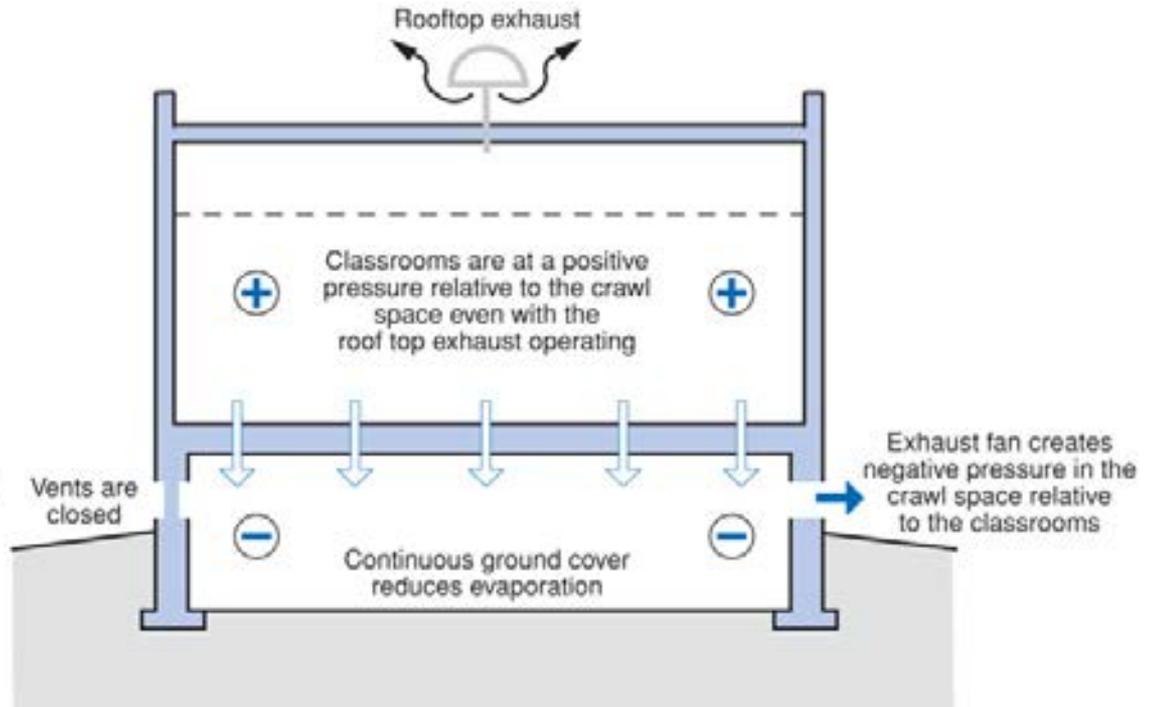




Figure 5.4

New Air Pressure Relationship

- Closing the crawl space vents and using an exhaust fan in the crawl space depressurizes the crawl space relative to the classrooms















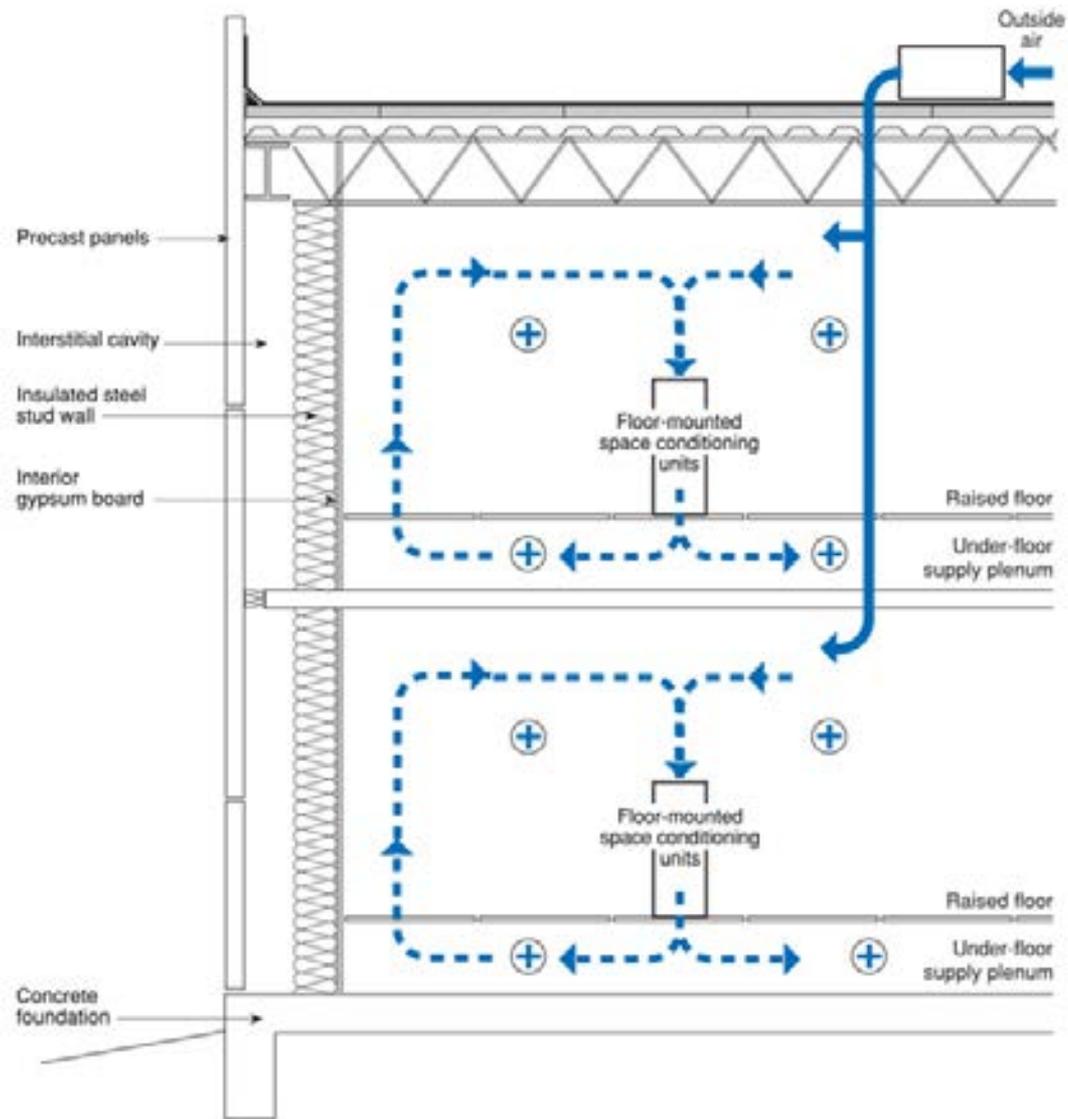


Figure 5.10
HVAC System as Designed

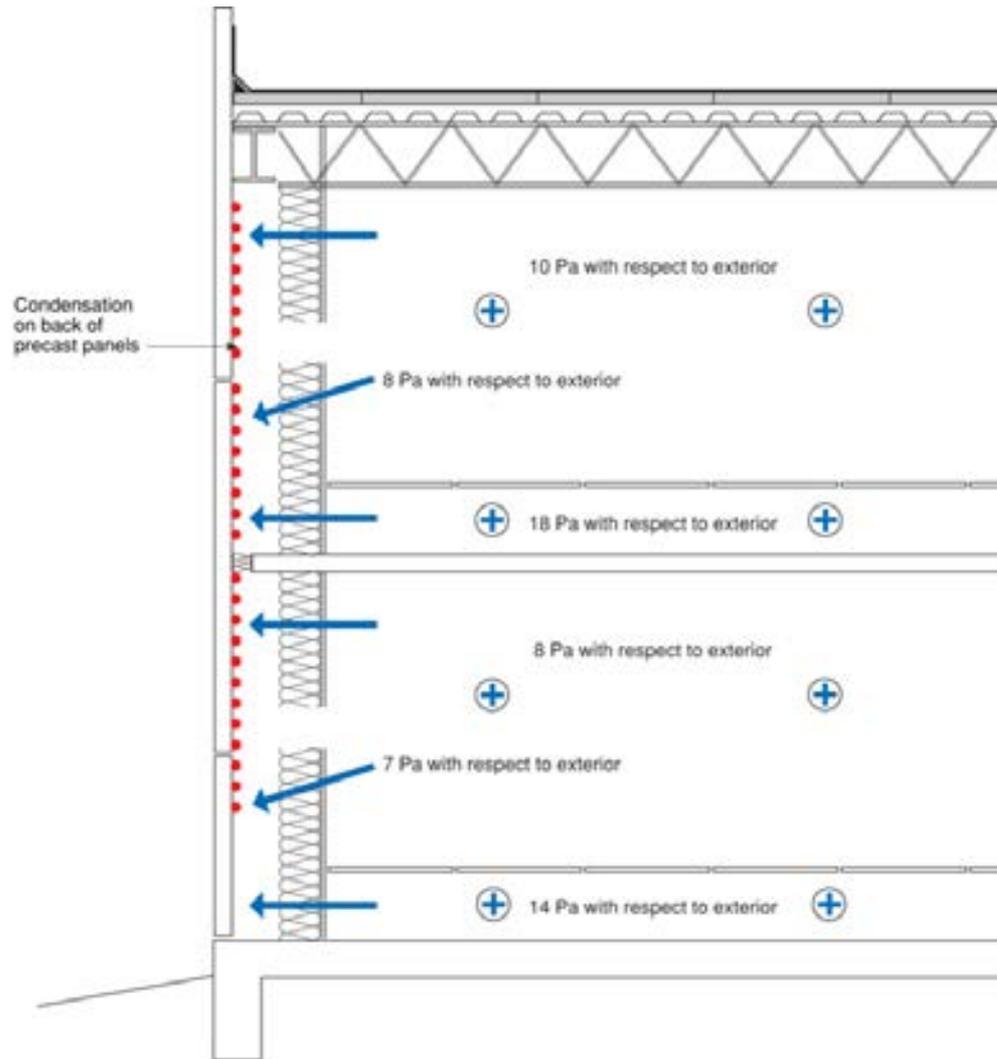


Figure 5.11
Unintended Pressurization of Interstitial Cavity

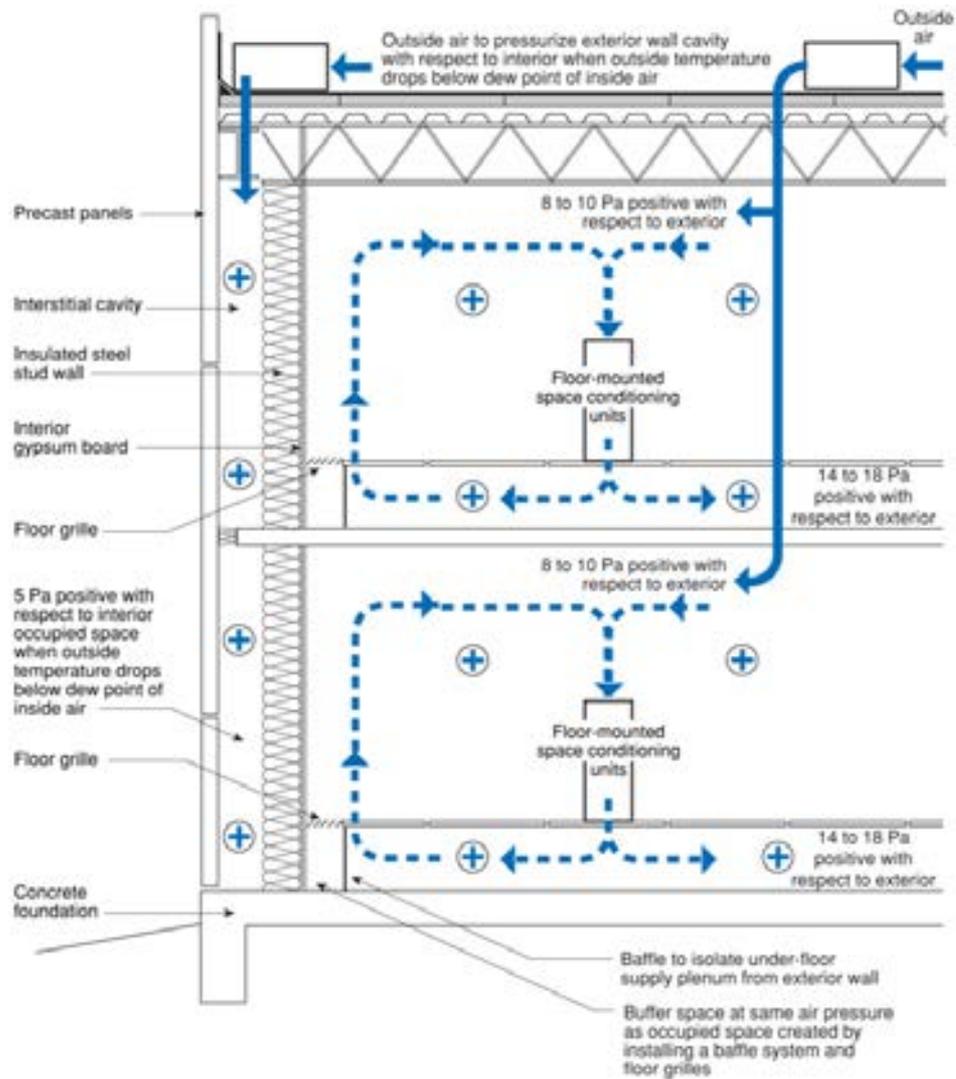
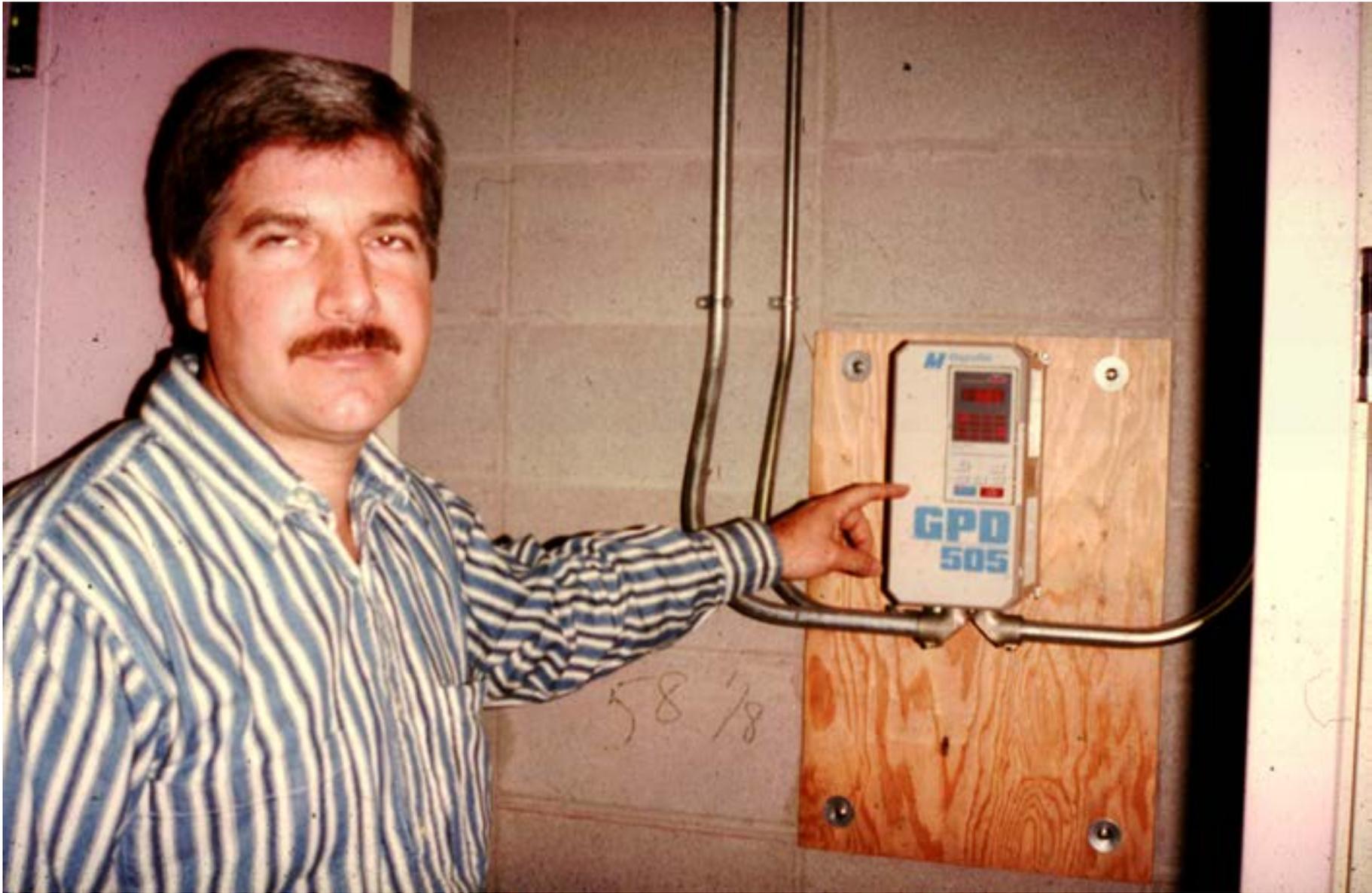


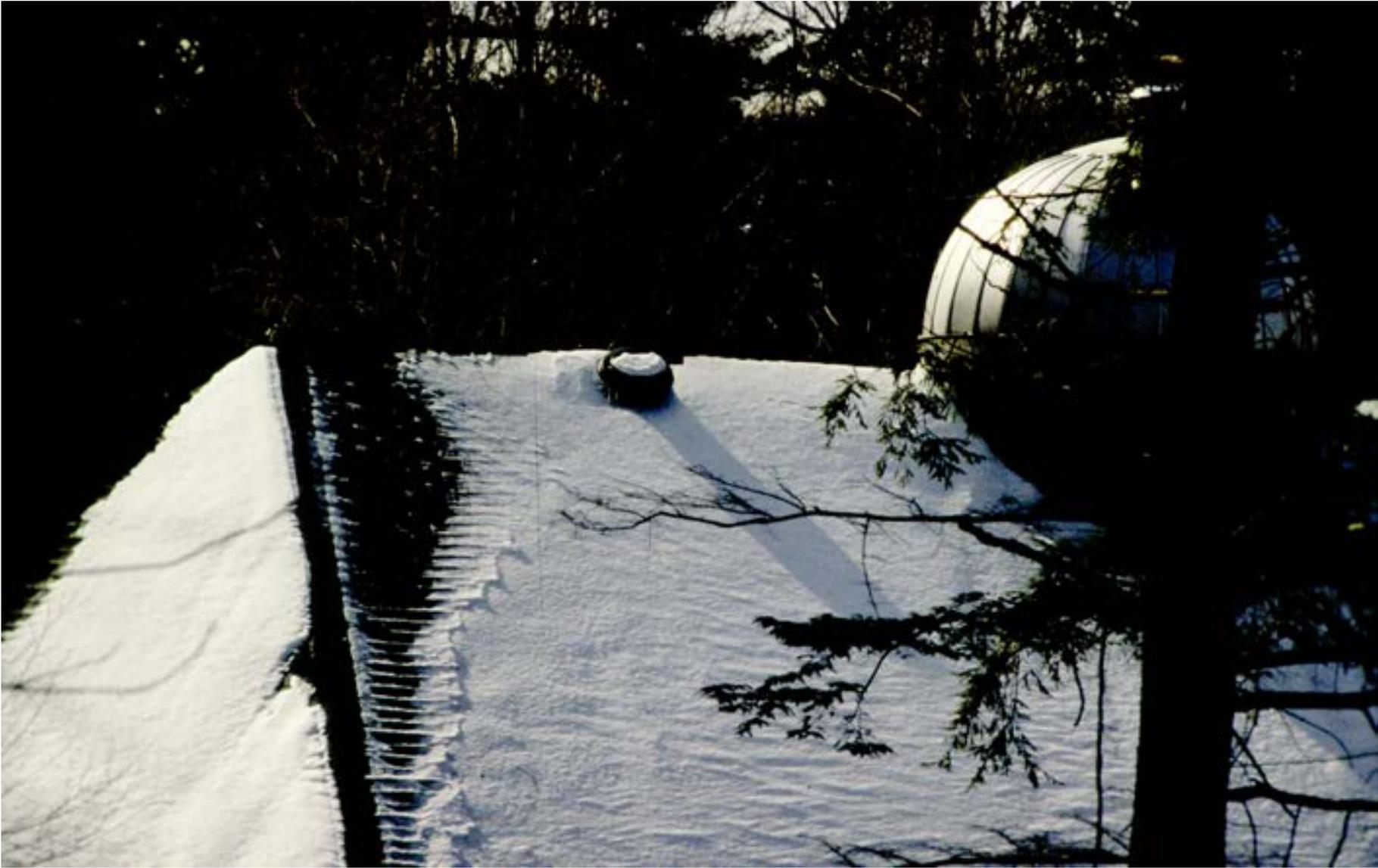
Figure 5.12
Modified Pressure Relationship











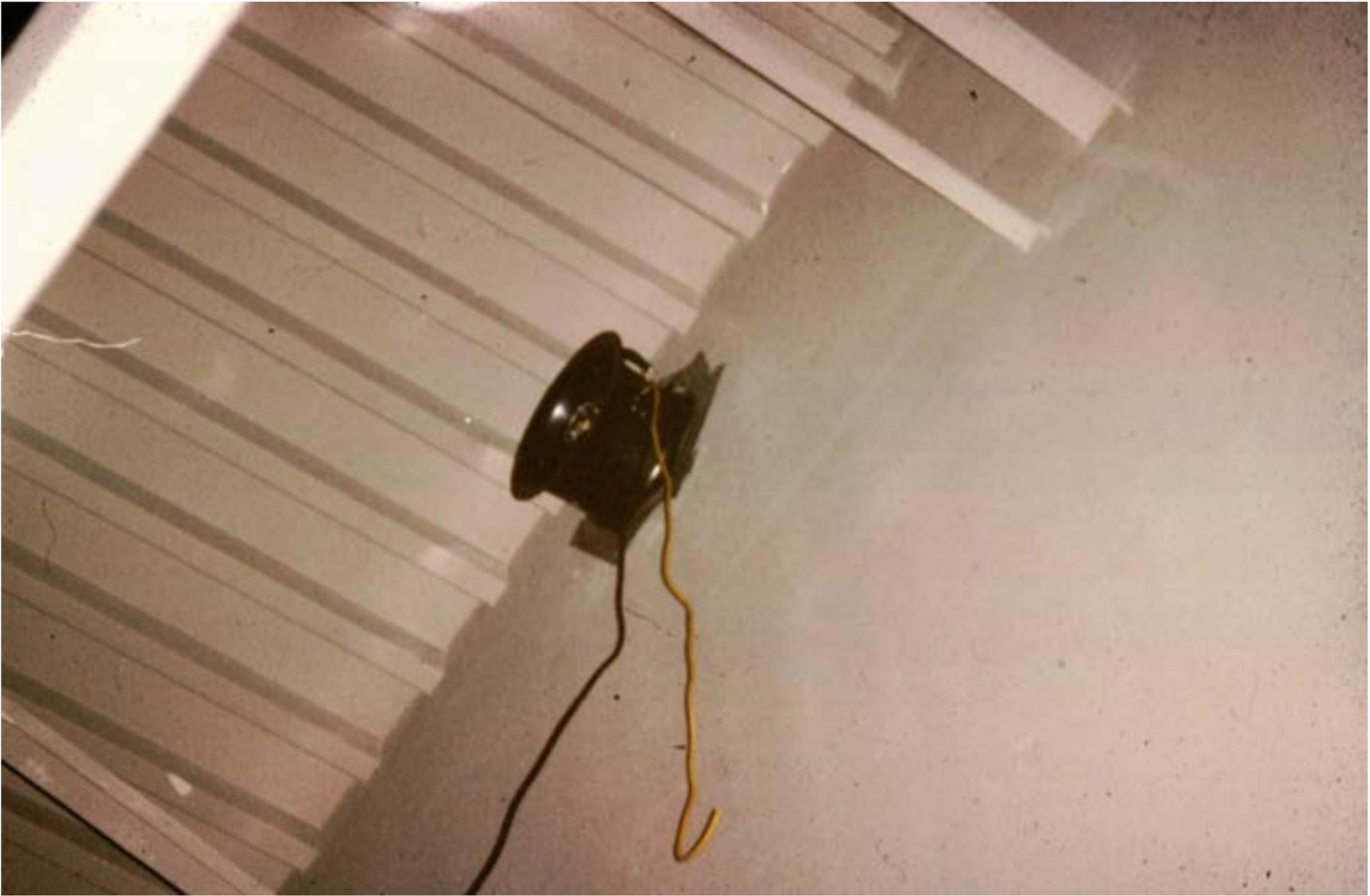












- P_o = outside air pressure
- P_i = inside air pressure
- P_c = cavity air pressure
- A_E = leakage area across exterior of wall assembly
- A_I = leakage area across interior of wall assembly

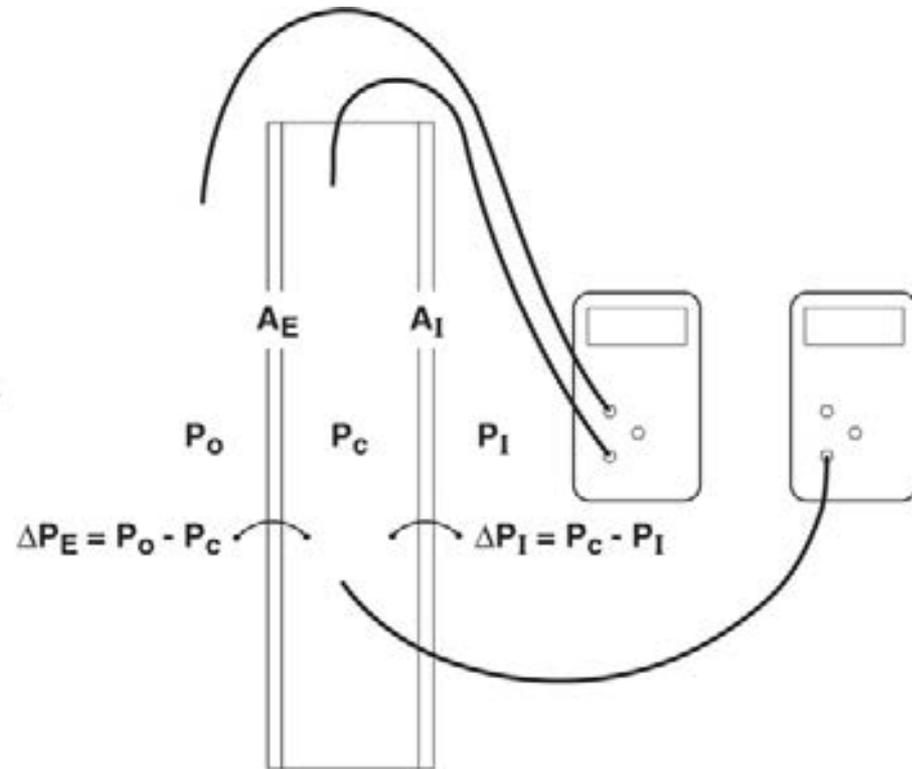


Figure 4.5

Measurement of Series Differential Pressure

$$Q = C_E A_E \Delta P_E^n = C_I A_I \Delta P_I^n$$

$$\frac{C_E A_E}{C_I A_I} = \frac{\Delta P_I^n}{\Delta P_E^n}$$

$$\cong \frac{A_E}{A_I} = \left(\frac{\Delta P_I}{\Delta P_E} \right)^n$$

$$\cong \left(\frac{A_E}{A_I} \right)^{\frac{1}{n}} = \frac{\Delta P_I}{\Delta P_E}$$

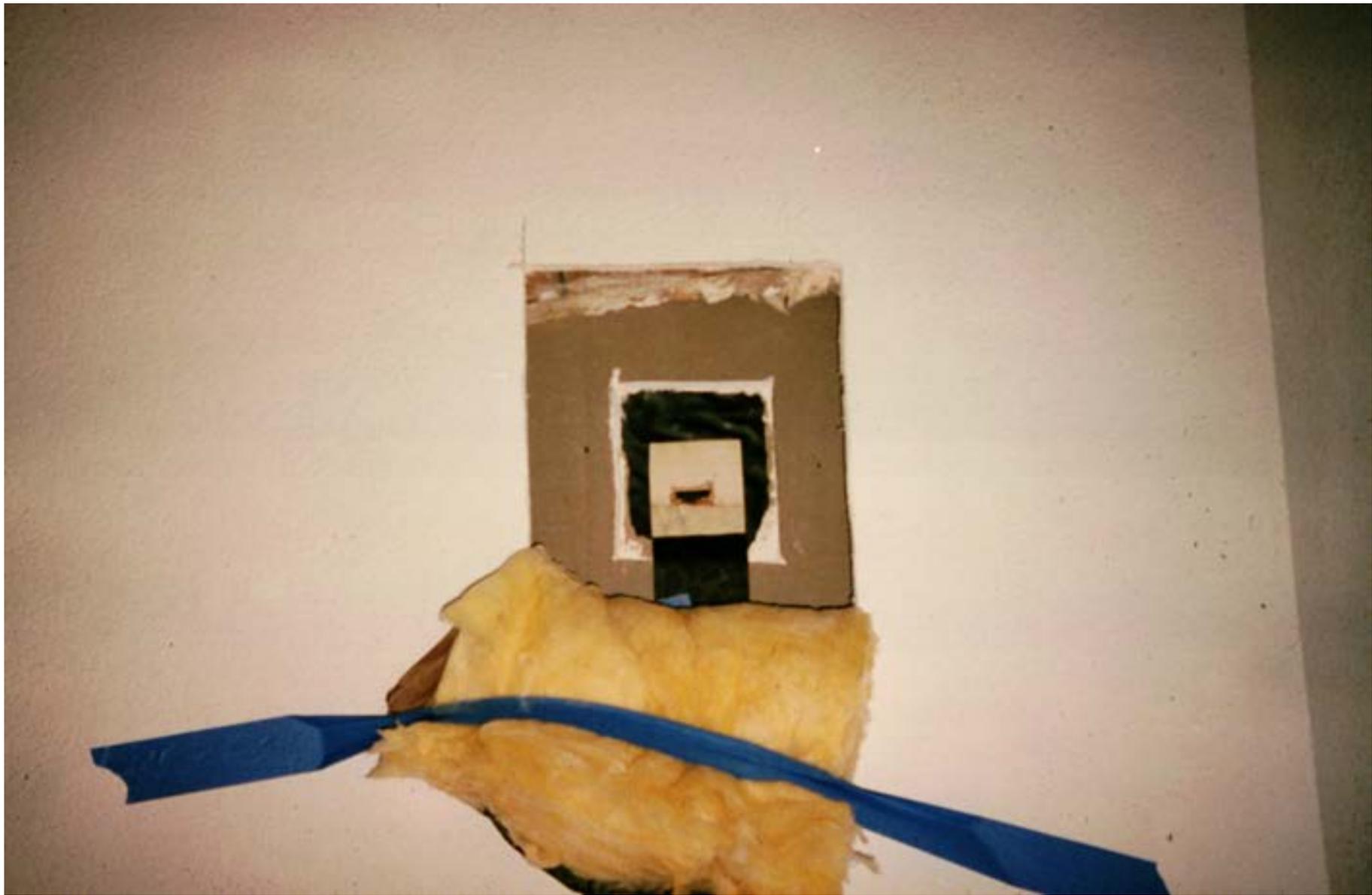
from Hutcheon & Handegord, 1983

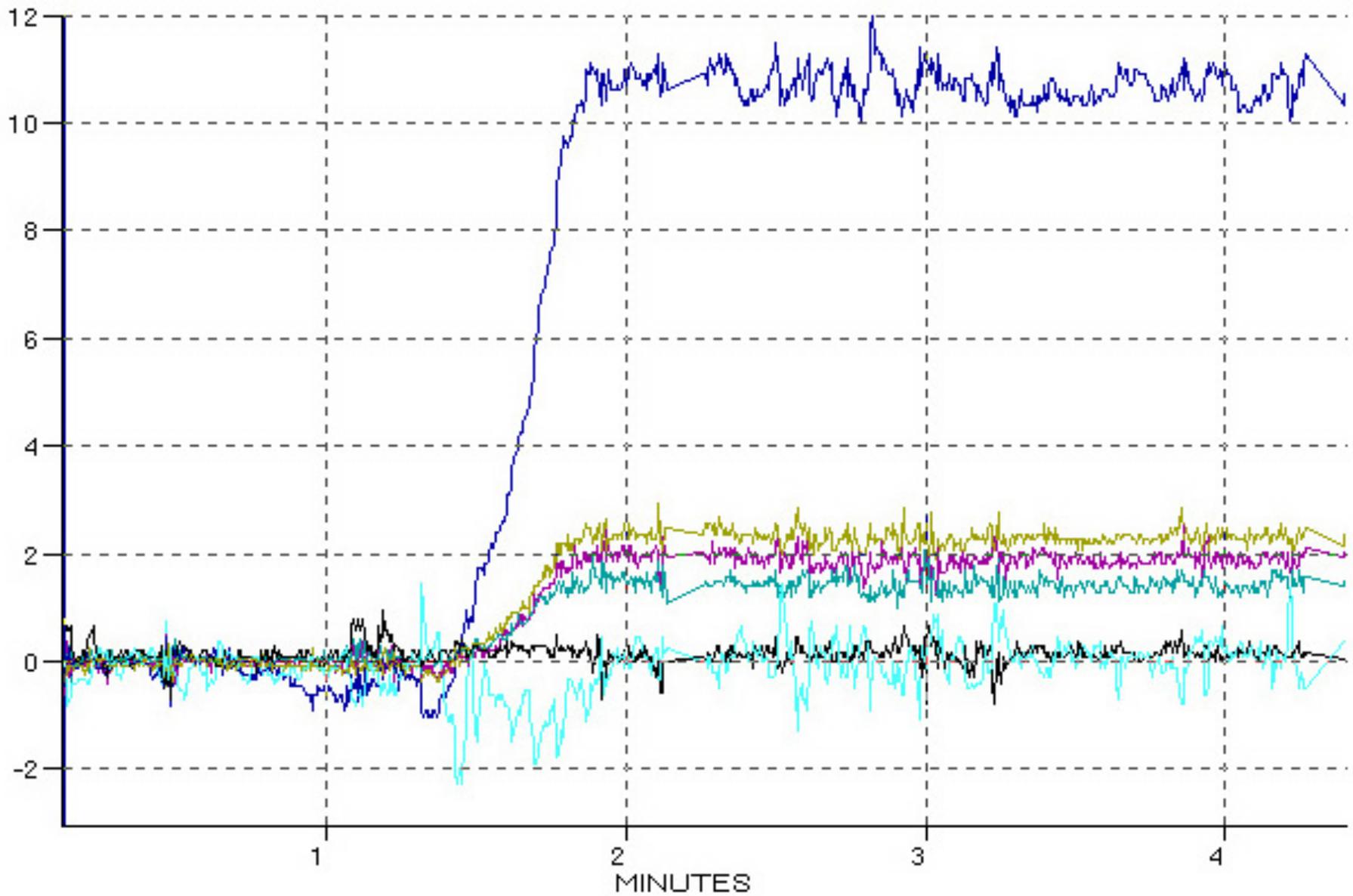
















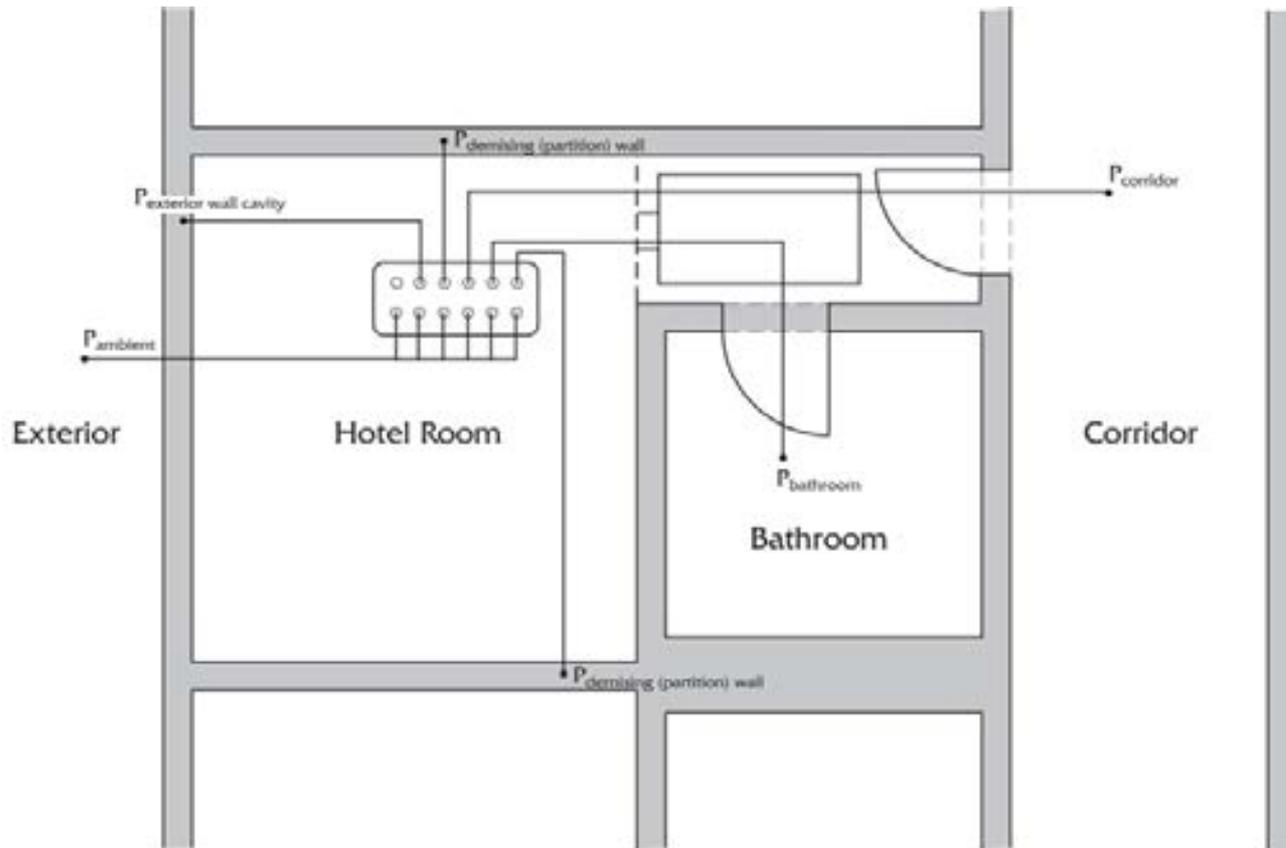
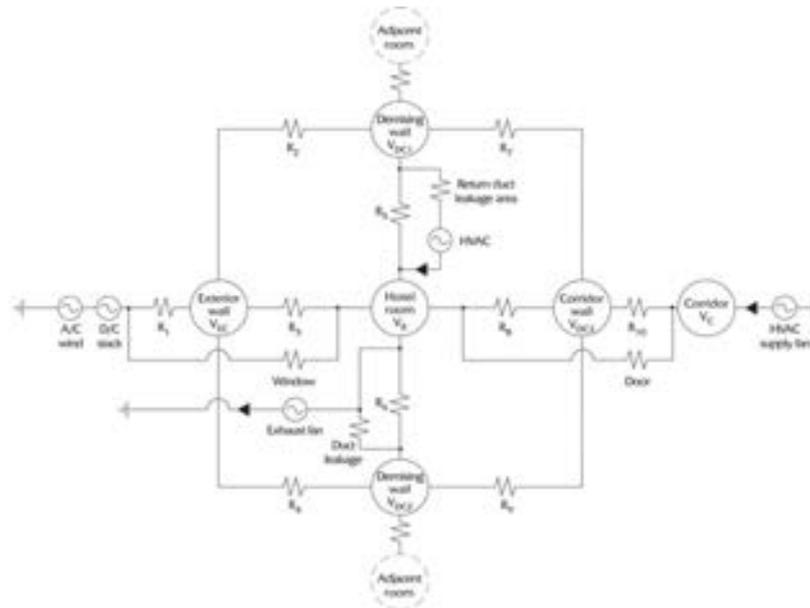


Figure 4.3

Multi-Channel Pressure Measurements

- Six channel micromanometer connected to laptop computer used to map pressure in the hotel room described in Figure 3.6
- All pressures measured relative to exterior air pressure
- Pressure response determined by opening and closing doors, cycling fan-coil, rooftop exhaust and corridor make-up air systems



- R_1 = Leakage area between ambient and exterior wall cavity
- R_2 = Leakage area between exterior wall cavity and demising (partition) wall cavity
- R_3 = Leakage area between exterior wall cavity and hotel room
- R_4 = Leakage area between exterior wall cavity and demising (partition) wall cavity
- R_5 = Leakage area between demising (partition) wall cavity and hotel room
- R_6 = Leakage area between demising (partition) wall cavity and hotel room
- R_7 = Leakage area between demising (partition) wall cavity and corridor demising wall cavity
- R_8 = Leakage area between hotel room and corridor demising wall cavity
- R_9 = Leakage area between demising (partition) wall cavity and corridor demising wall cavity
- R_{10} = Leakage area between corridor demising wall cavity and corridor
- V_{EC} = Pressure in exterior wall
- V_R = Pressure in hotel room
- V_C = Pressure in corridor
- V_{DC1} = Pressure in demising (partition) wall cavity
- V_{DC2} = Pressure in demising (partition) wall cavity
- V_{DC3} = Pressure in corridor demising wall cavity

Figure 3.25
Electrical Analogue of Hotel Room

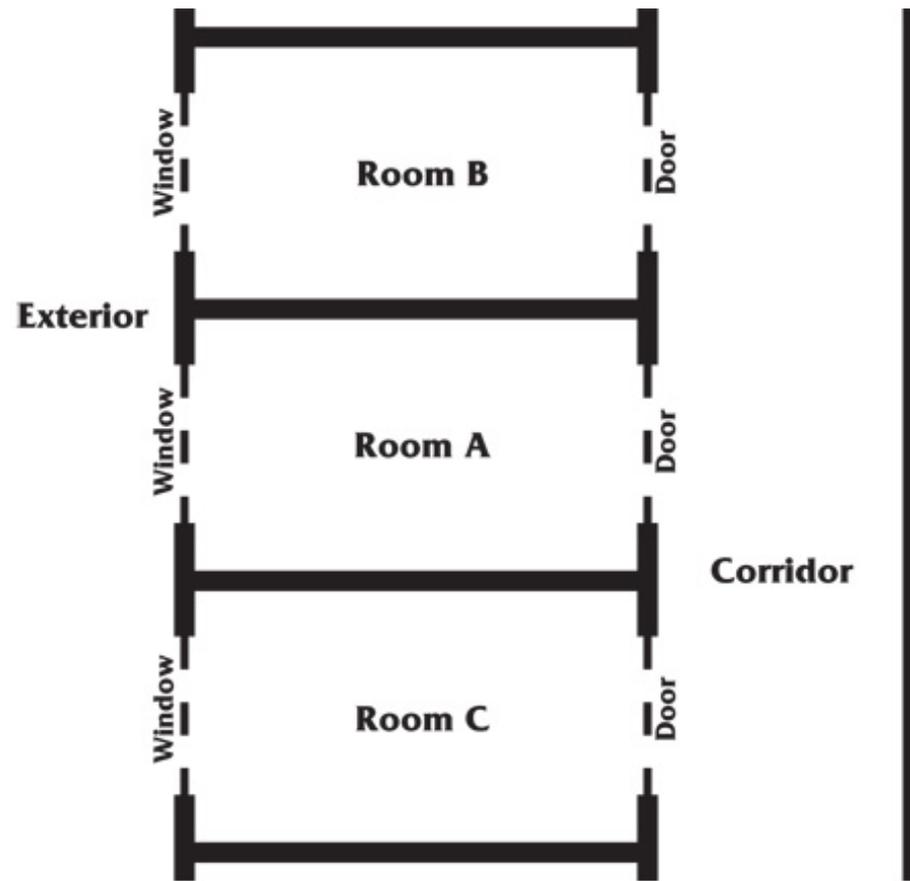
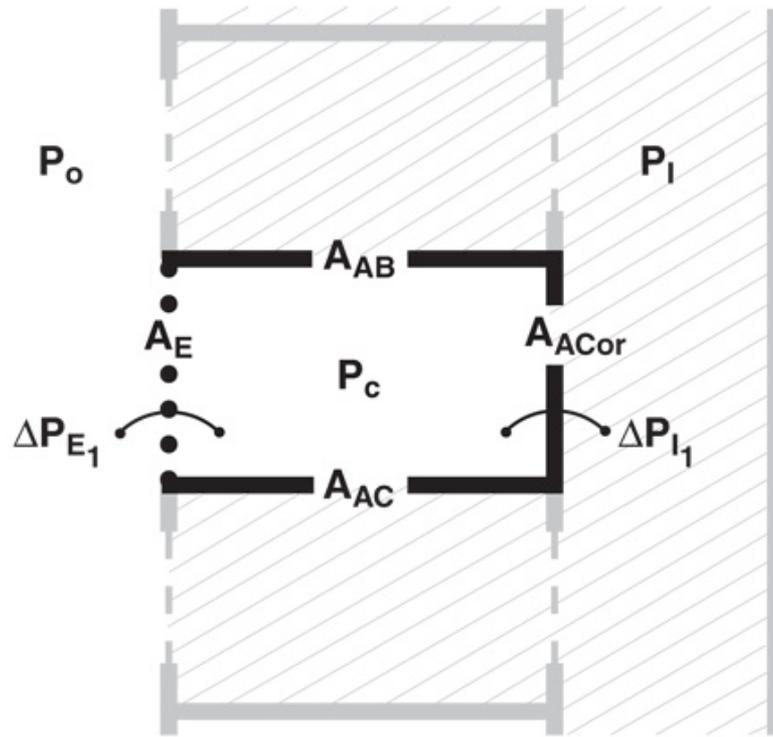


Figure 4.6
**Series of Rooms Connected
to Corridor**



$$A_I = A_{AB} + A_{AC} + A_{ACor}$$

Figure 4.7

Initial Pressure Measurements

- Door to corridor in Room A closed
- Doors to corridor for Rooms B and C are open
- Windows in all rooms closed

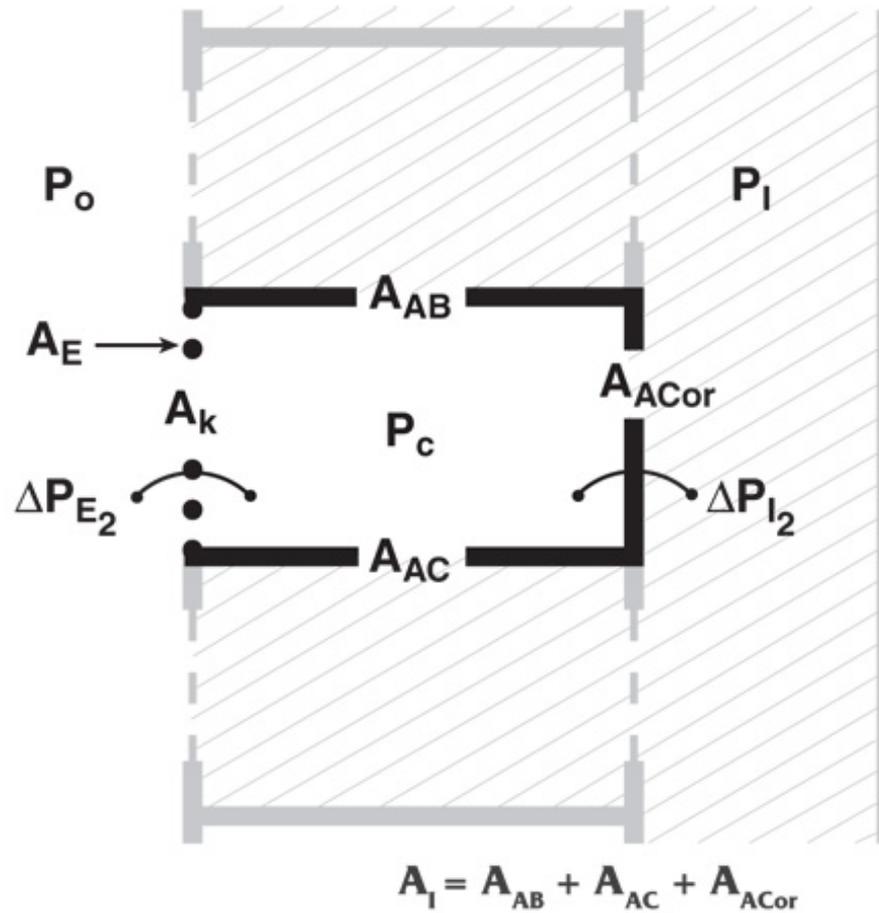


Figure 4.8

Subsequent Pressure Measurements

- An opening of known size, A_k , is added to A_E (i.e. window in Room A is opened)

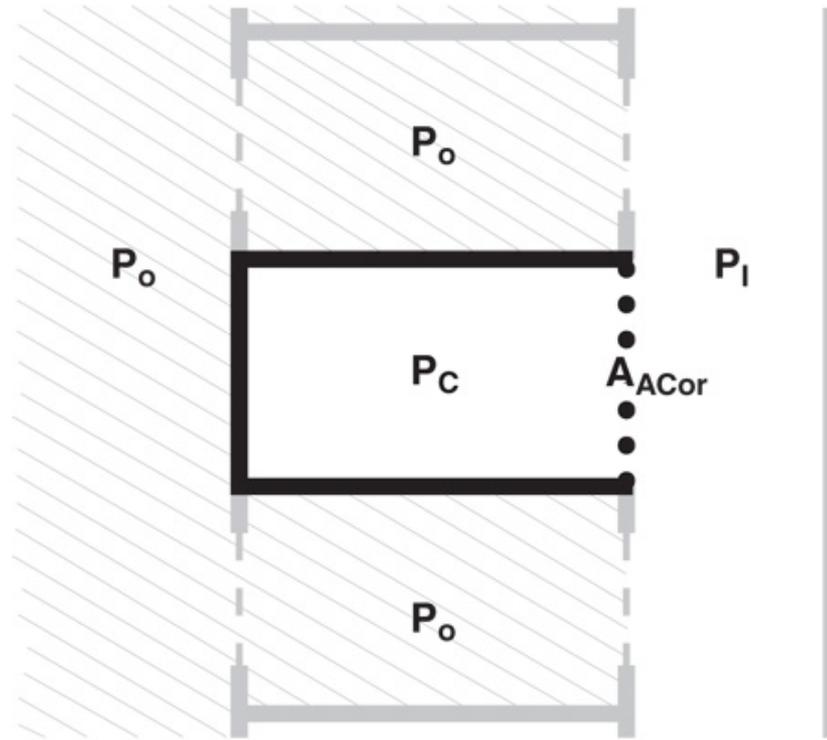
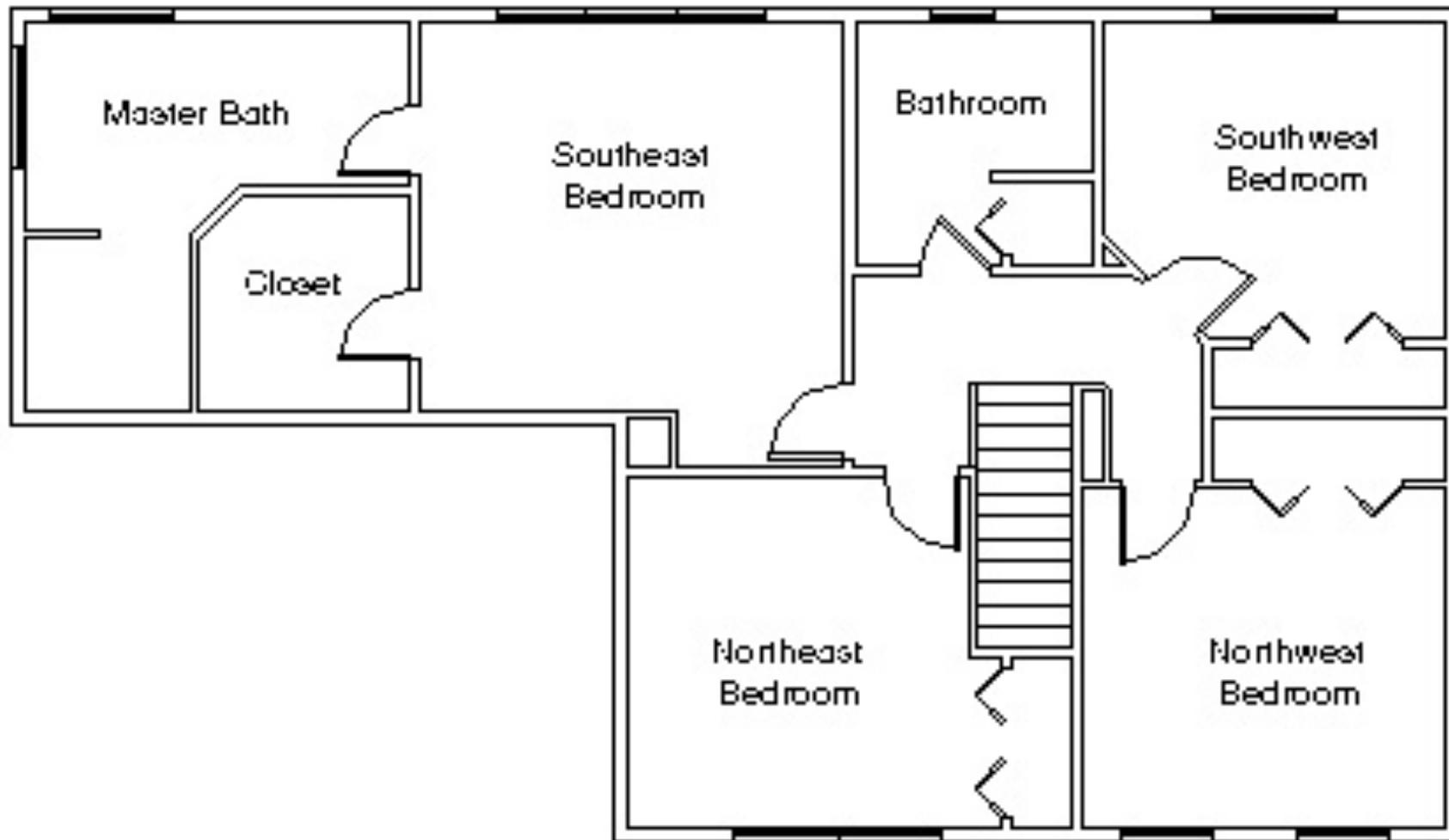


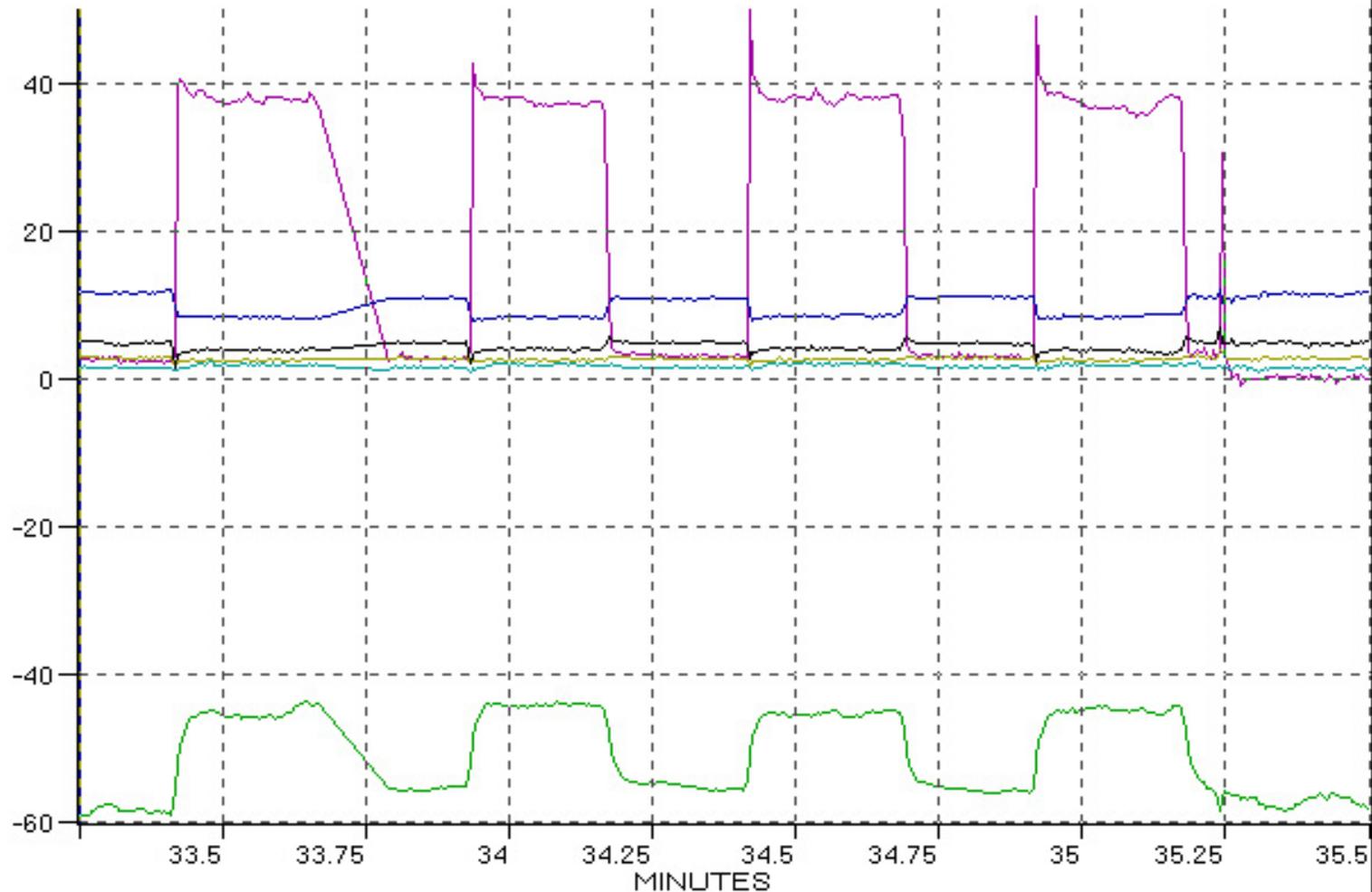
Figure 4.9

Determining Leakage Area A_{ACor}

- Windows in Rooms B and C are opened
- Windows in Room A are closed
- Doors to corridor for all rooms are initially closed; door in Room A subsequently opened

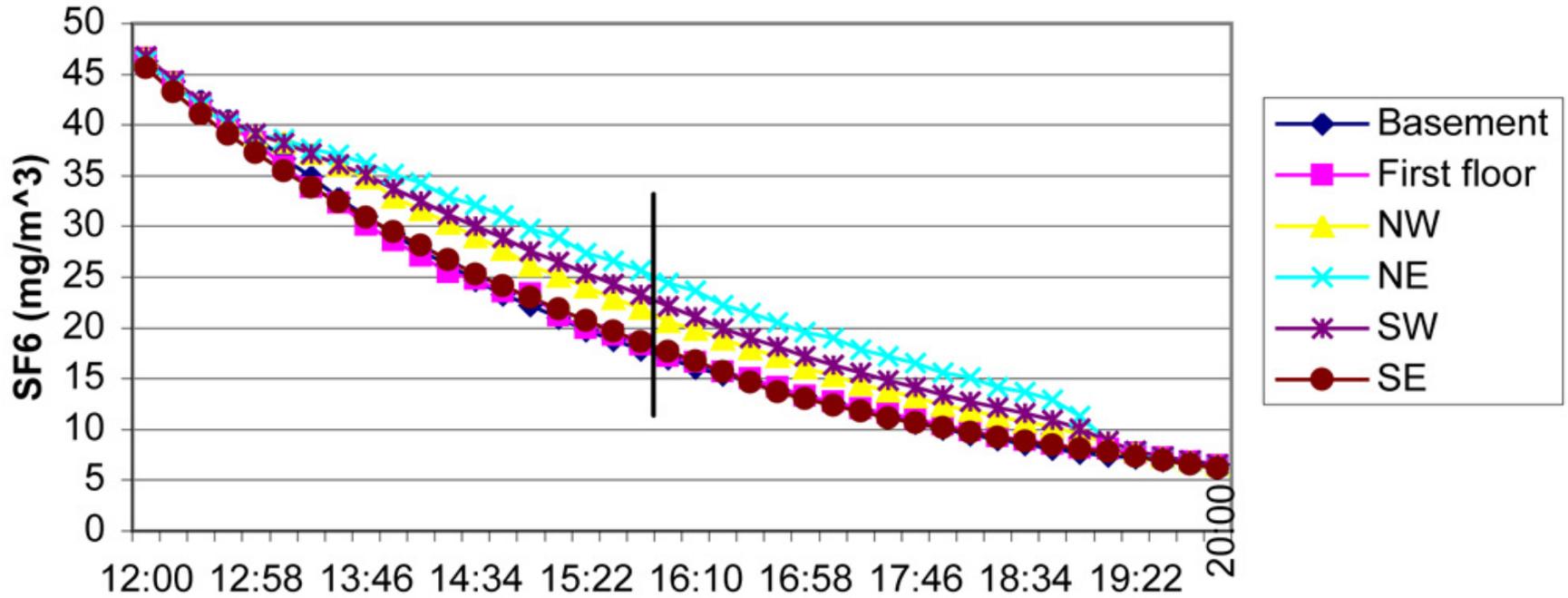




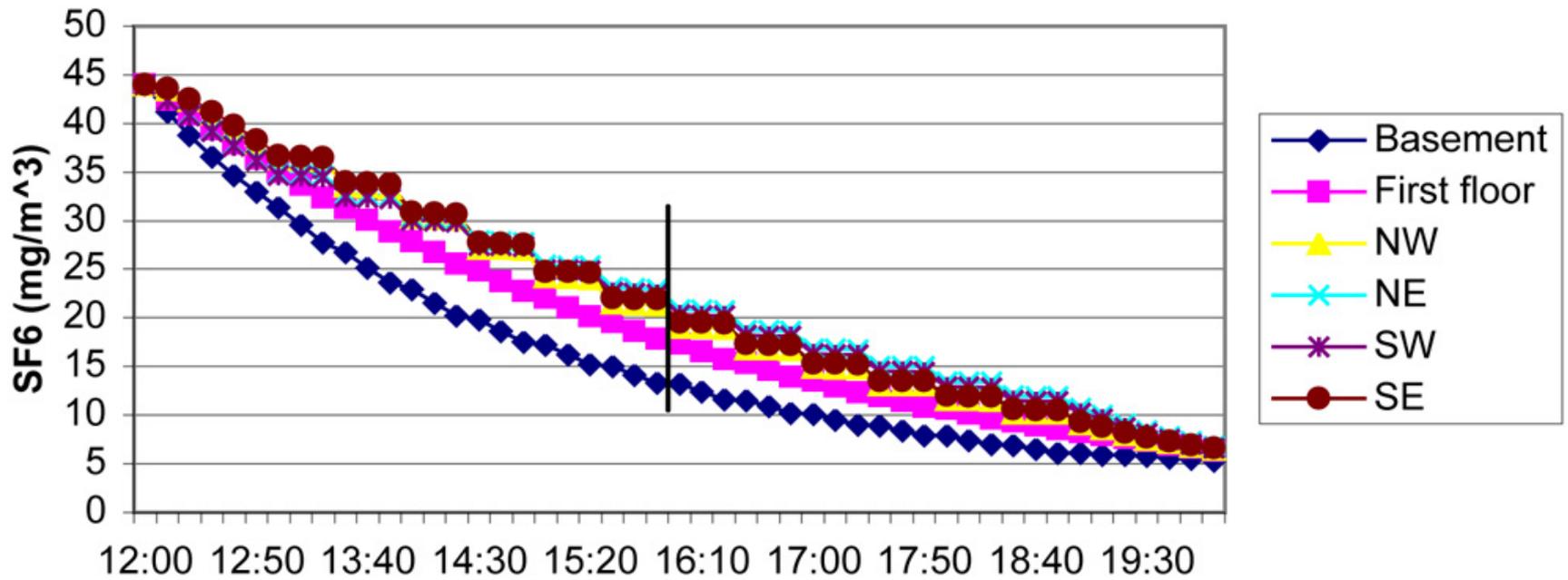


+ NWBED	o SWBED	• SEBED	o NEBED
▪ bsmt	□ outref		

Tracer Gas Data (HVAC System on intermittently)



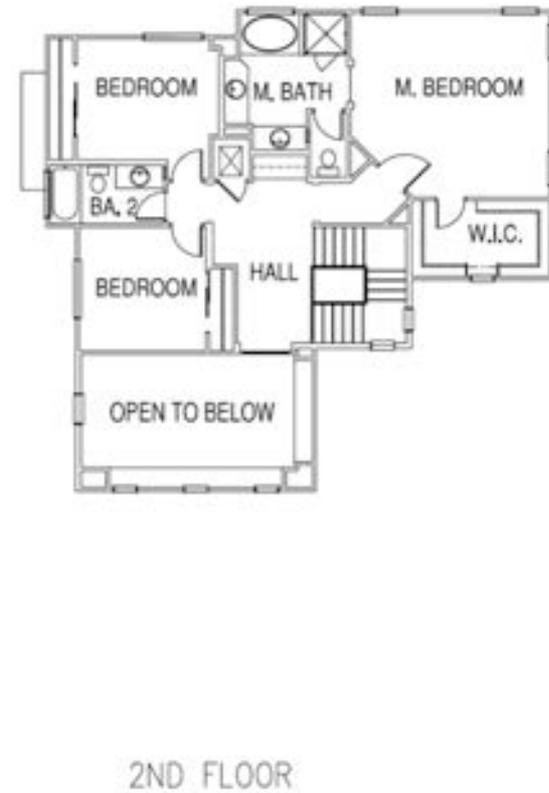
Standard Contam96 Analytical Model (HVAC System on)



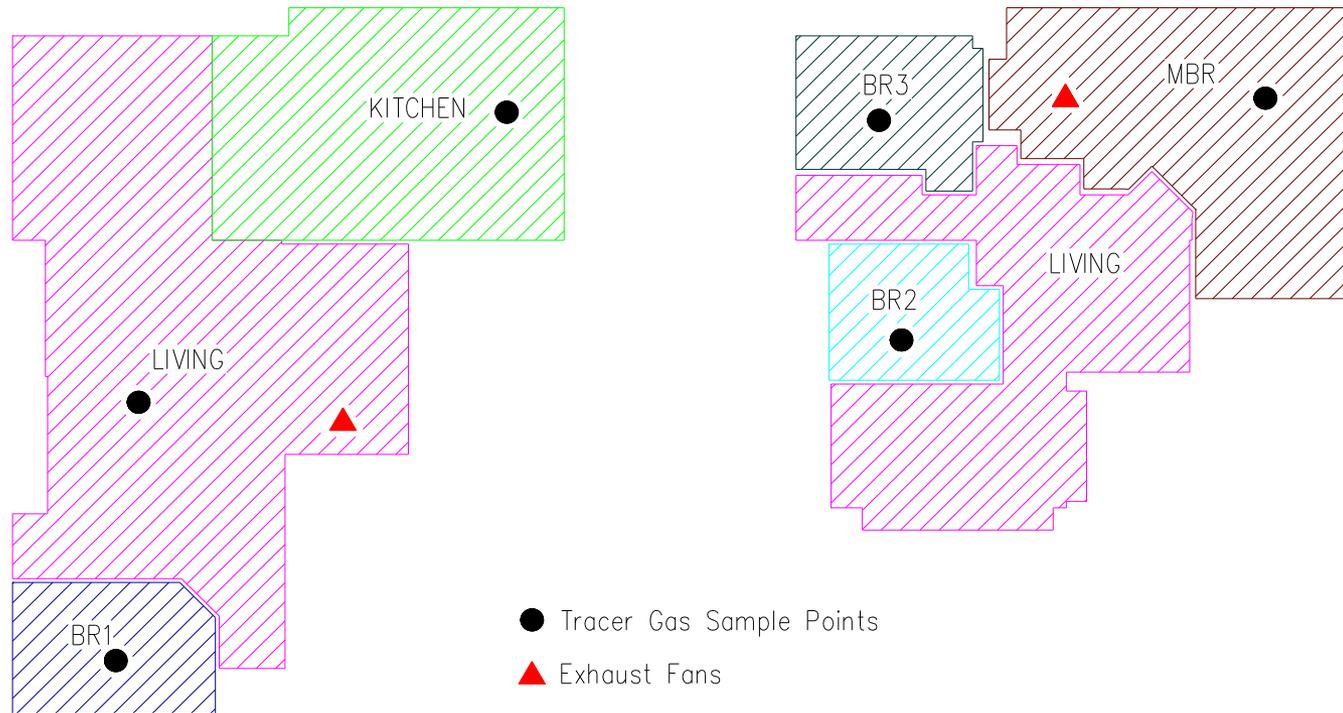


- Tracer gas test of a production Building America house in Sacramento
- 2-story, 4 bedrooms, ~2500 square feet
- Ventilation systems tested: supply and exhaust ventilation, with and without mixing via central air handler

Floor Plan - 2 Story House



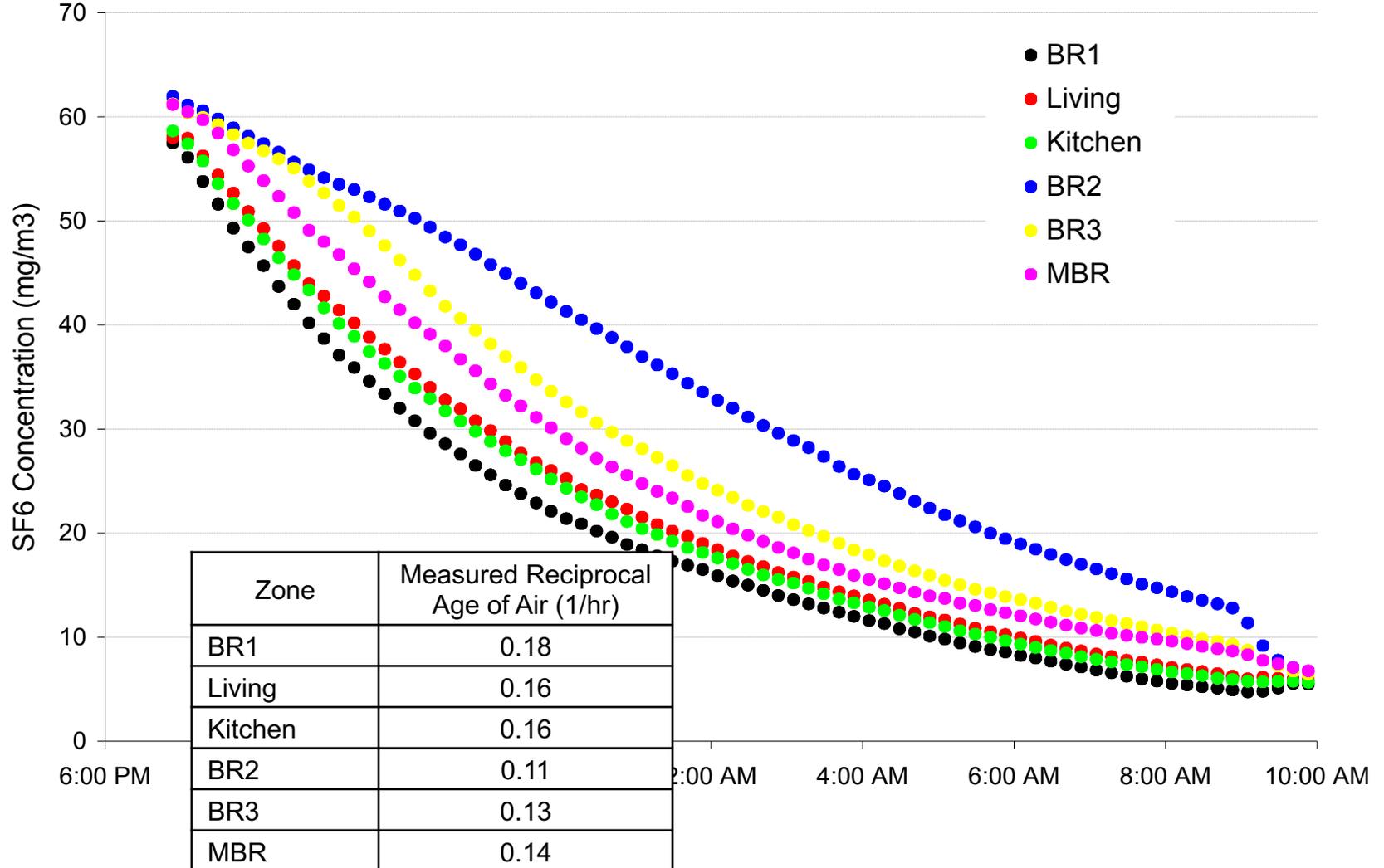
Zones – 2 Story House



- Tracer gas decay tests—establish uniform concentration of tracer gas and then activate ventilation system to remove it
- Reciprocal age-of-air can be calculated from decay curves (if weather conditions are sufficiently constant)

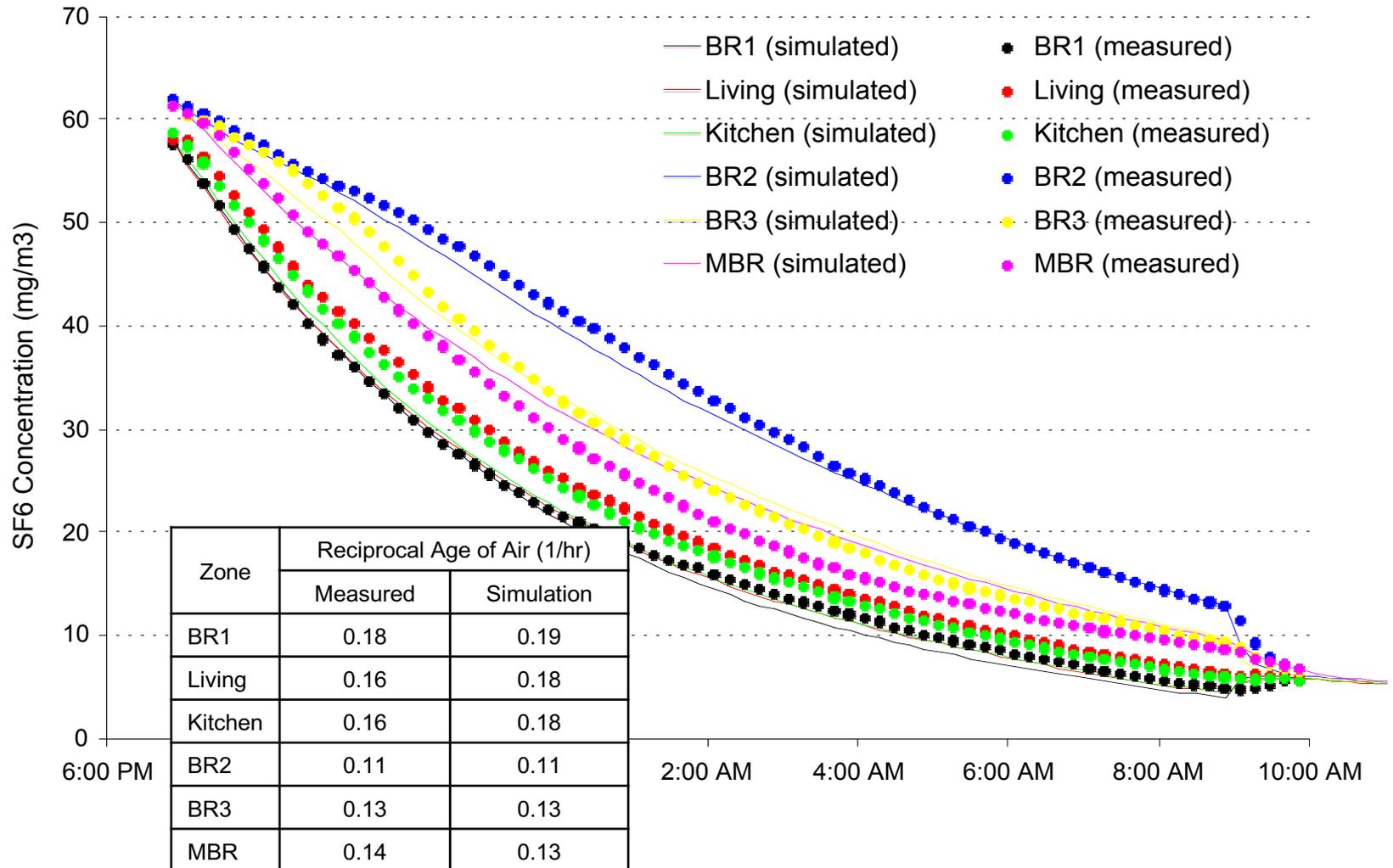
Example Results of Tracer Gas Testing

Laundry Exhaust, 100% of 62.2 Rate, Doors Closed, Transfer Grills Open, No Mixing

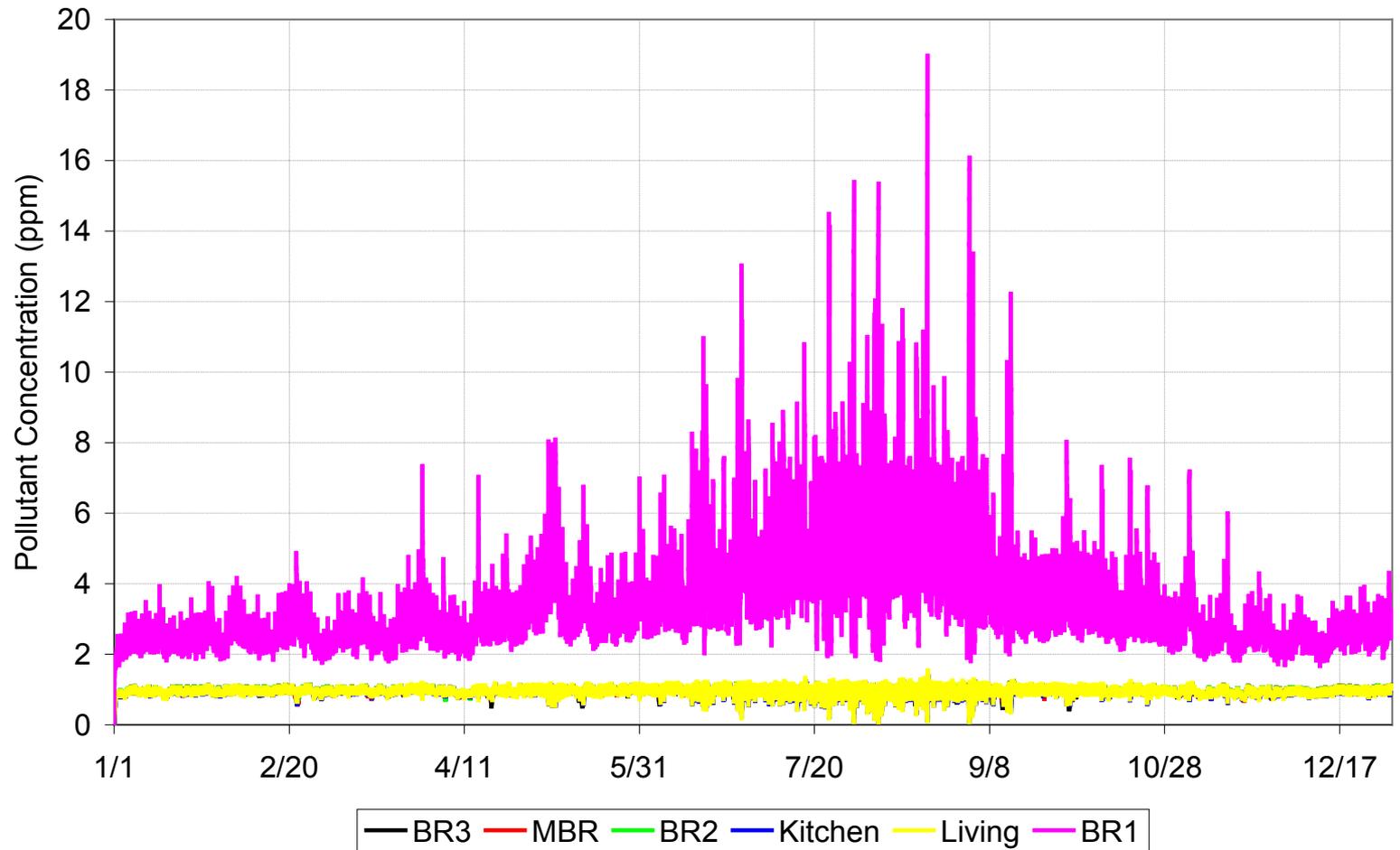


Example Results of Tuned CONTAM Model

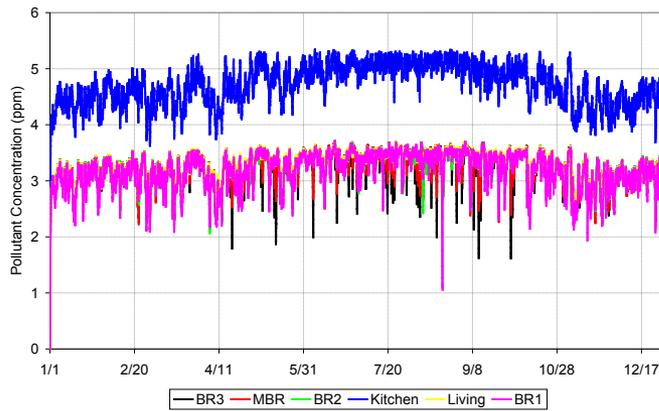
Laundry Exhaust, 100% of 62.2 Rate, Doors Closed, Transfer Grills Open, No Mixing



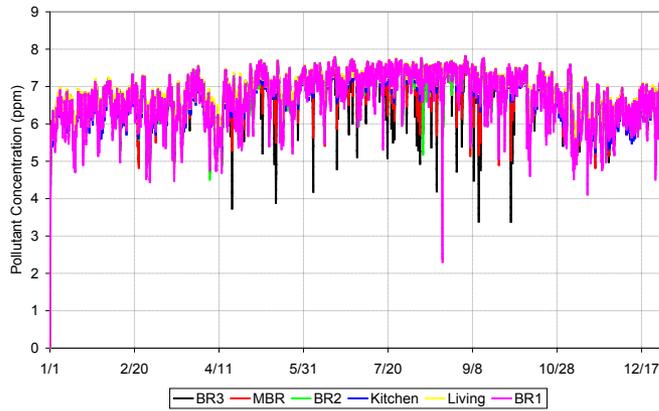
Bedroom 1 Pollutant



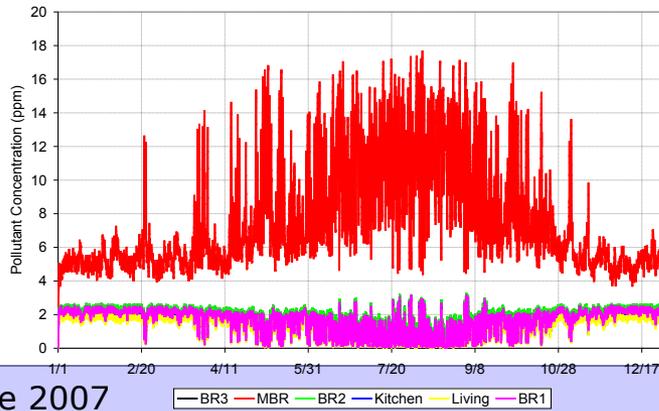
Kitchen Pollutant



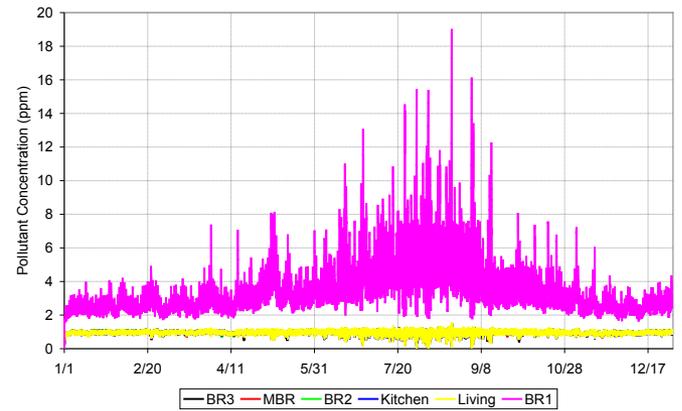
Living Room Pollutant



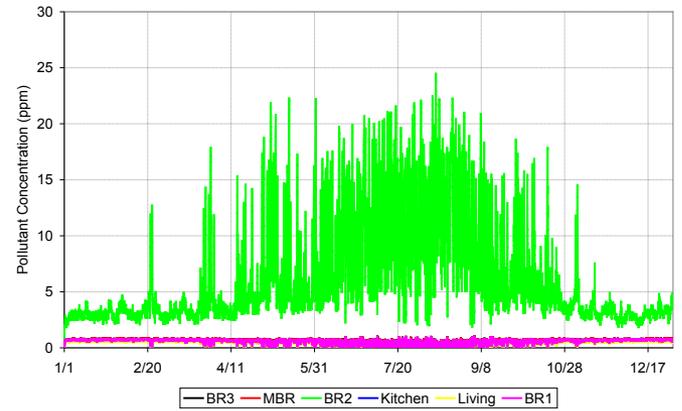
Master Bedroom Pollutant



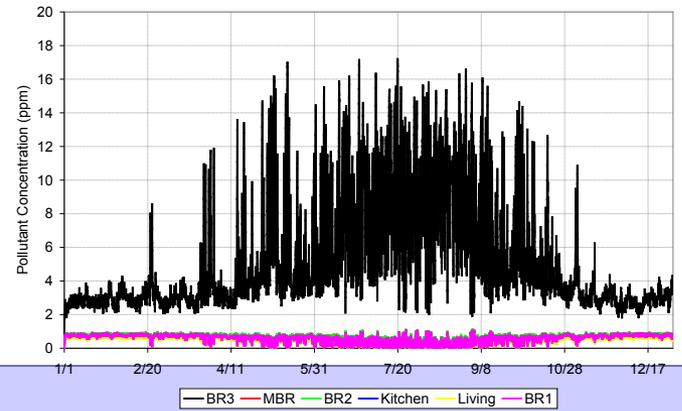
Bedroom 1 Pollutant



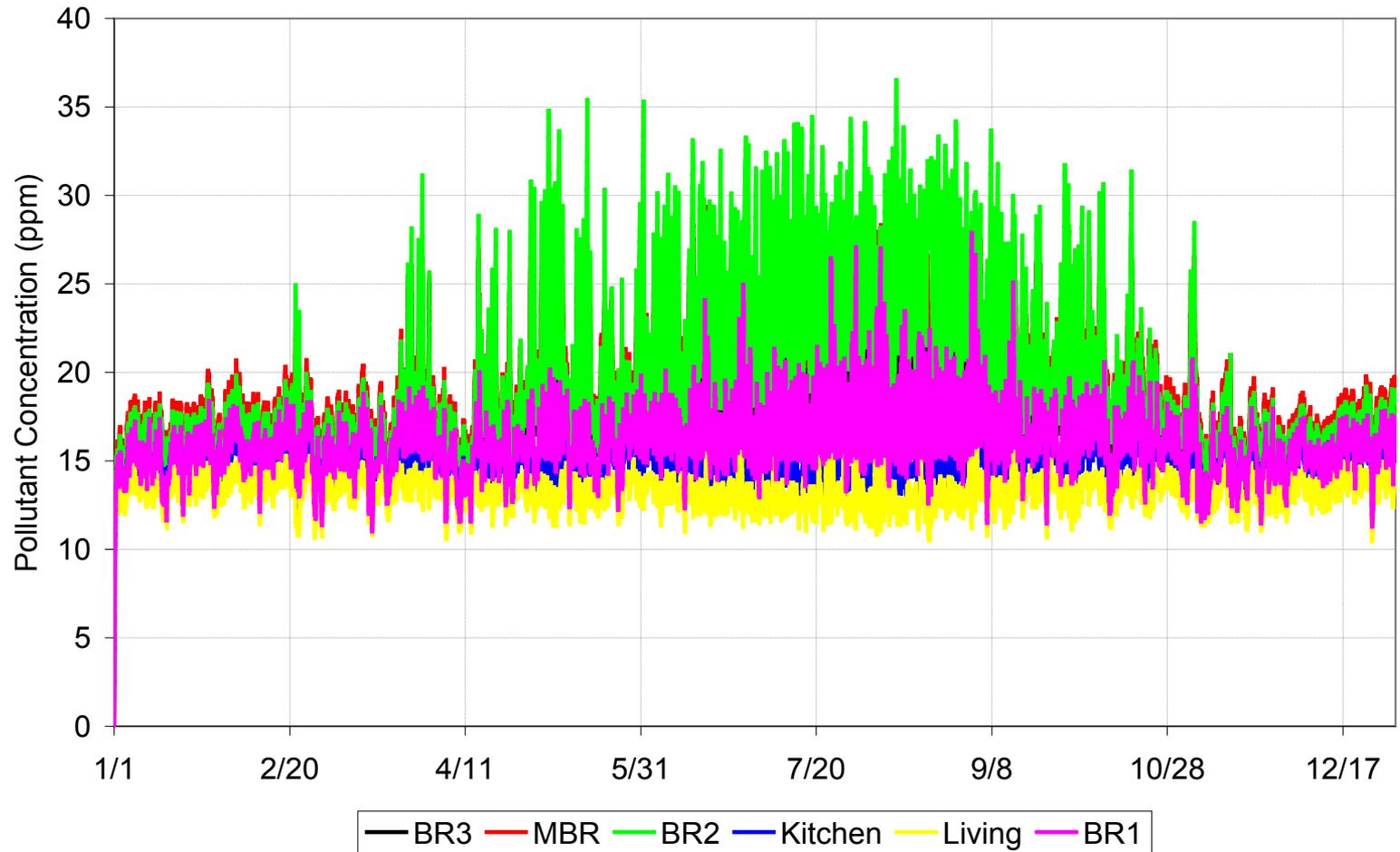
Bedroom 2 Pollutant



Bedroom 3 Pollutant



Total Pollutant Concentration by Room



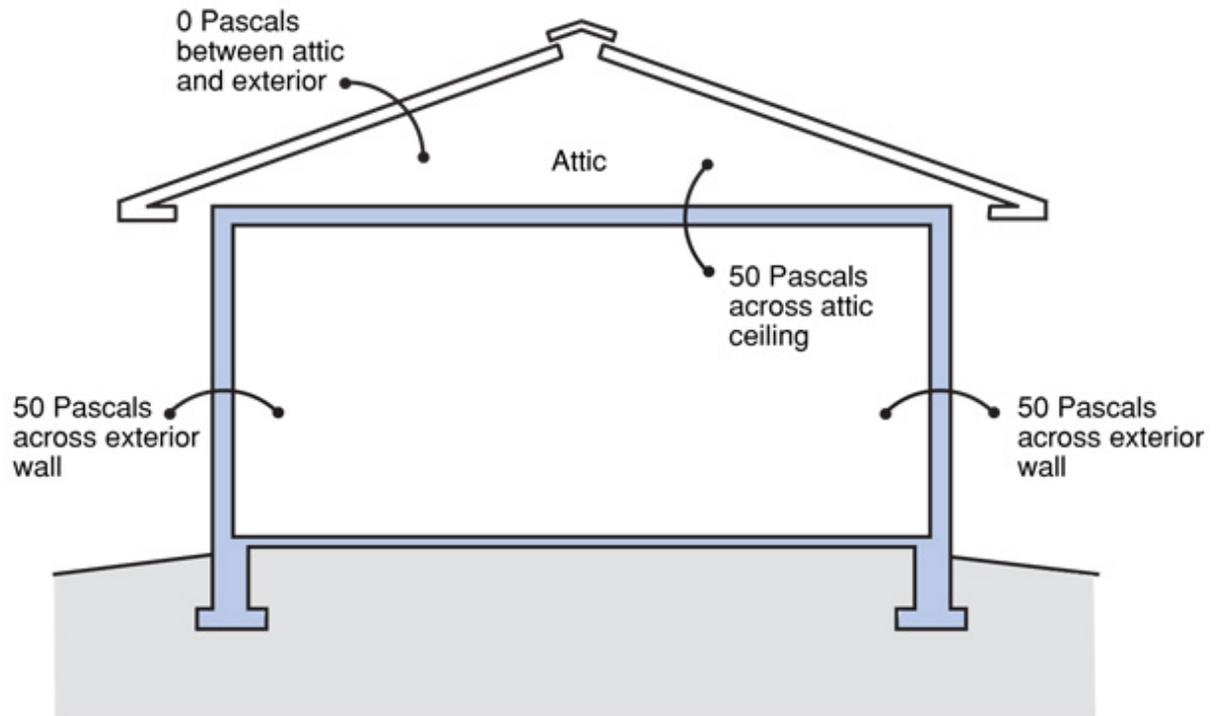


Figure 5.13

Well-Defined Pressure Boundary

- Pressure boundary defines effective building envelope environmental separator

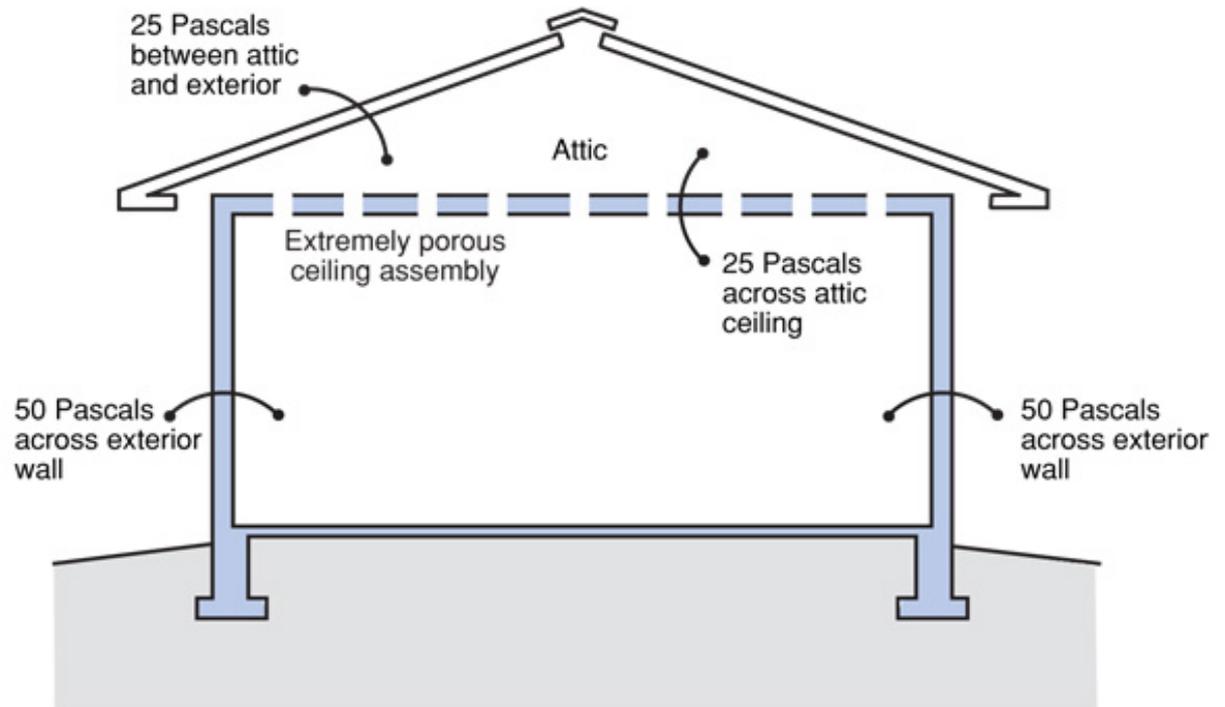


Figure 5.14

Poorly-Defined Pressure Boundary

- Pressure boundary poorly defined — ineffective at ceiling
- Pressure boundary not continuous at ceiling

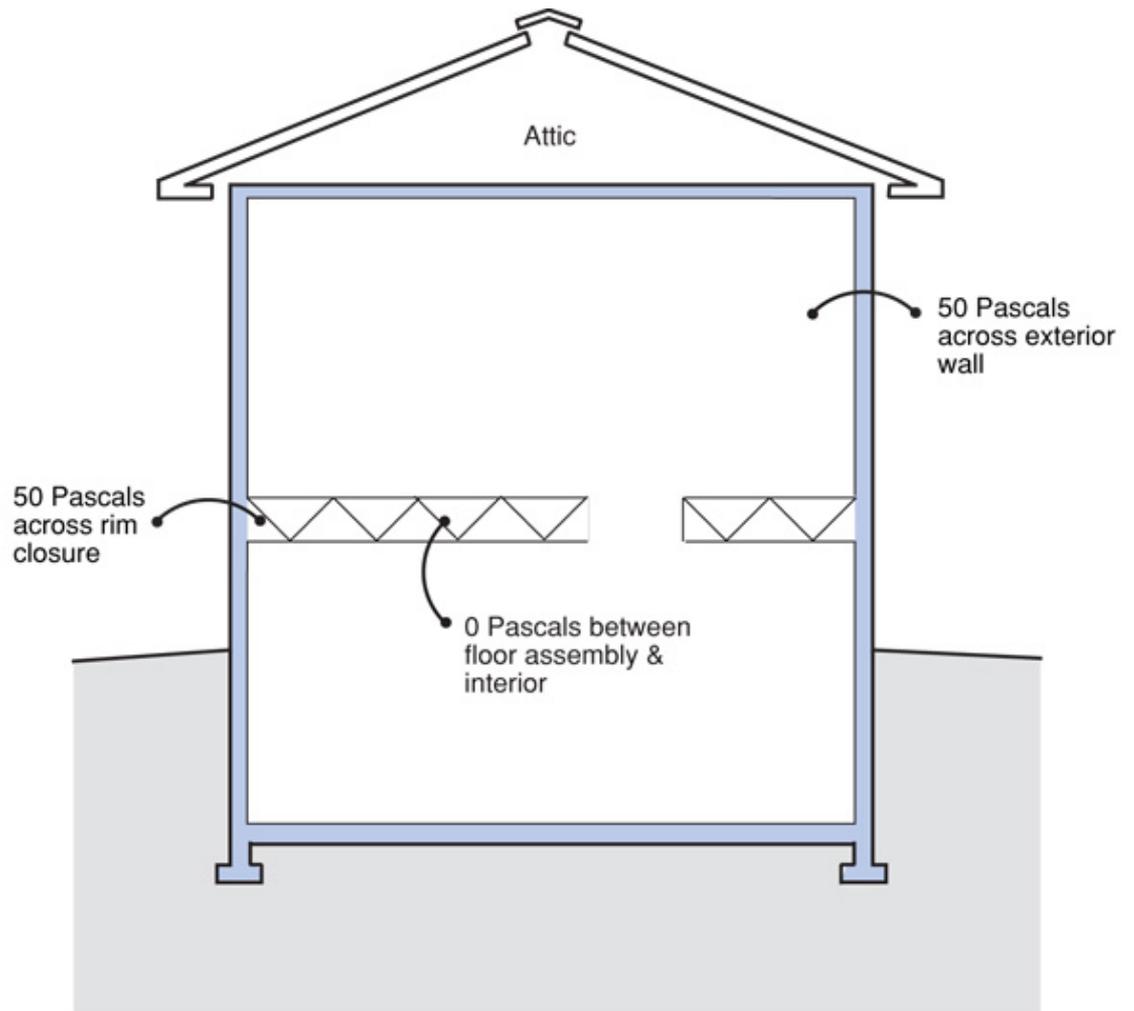


Figure 5.15

Tight Rim Closure

- Floor assembly “inside” well-defined pressure boundary
- Pressure boundary continuous at rim closure

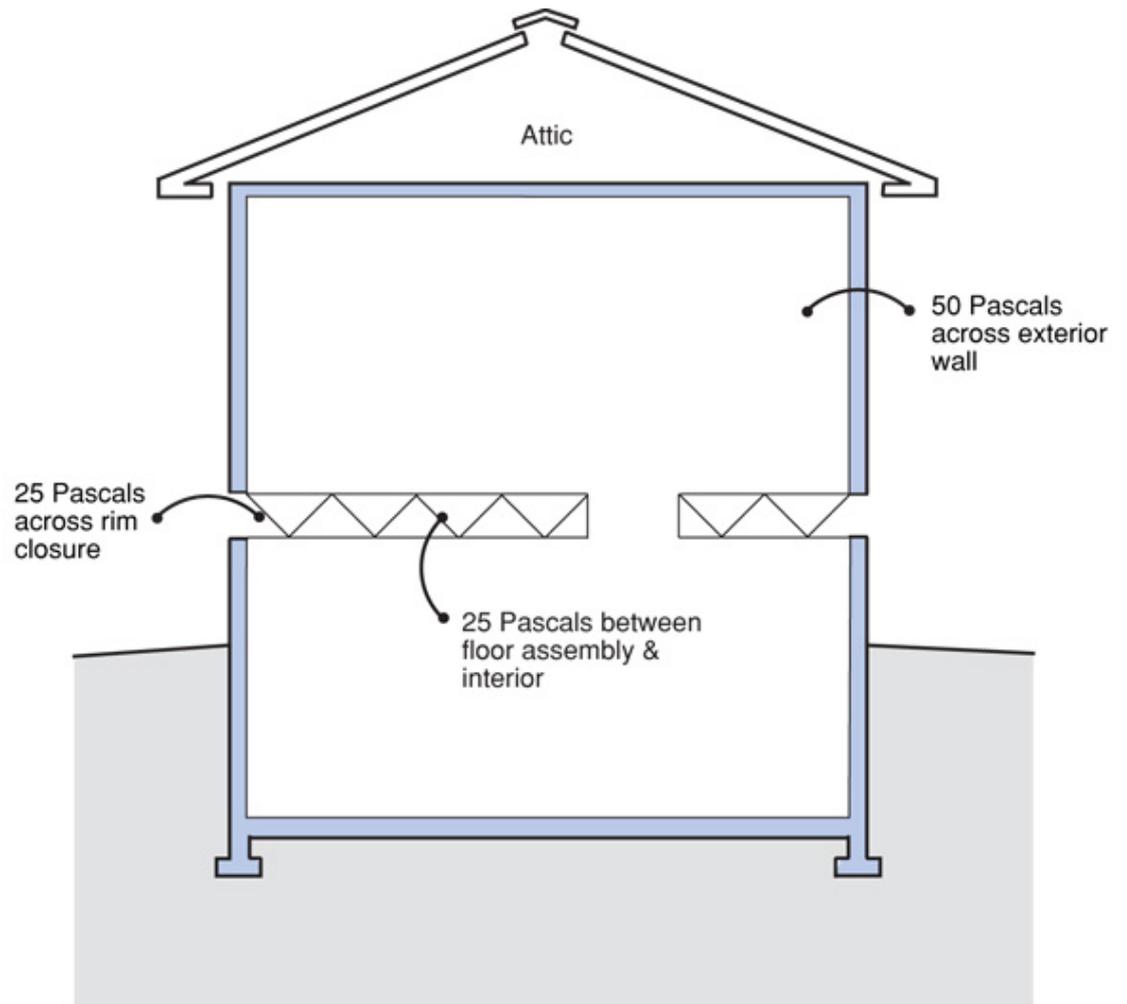


Figure 5.16

Leaky Rim Closure

- Floor assembly “outside” pressure boundary
- Pressure boundary not continuous at rim closure

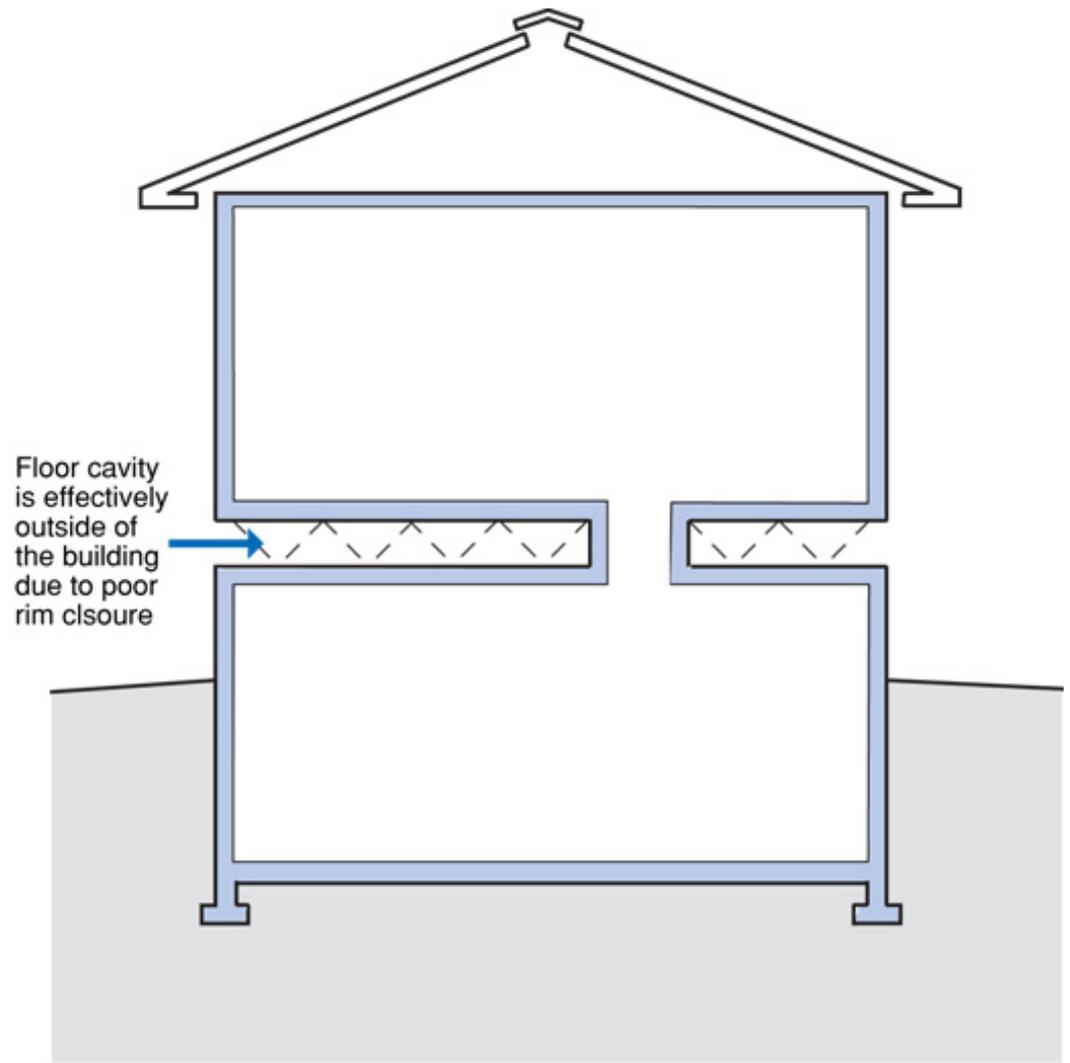


Figure 5.17

Pressure Boundary at Interior Floor

- Pressure boundary not contiguous with building envelope thermal boundary

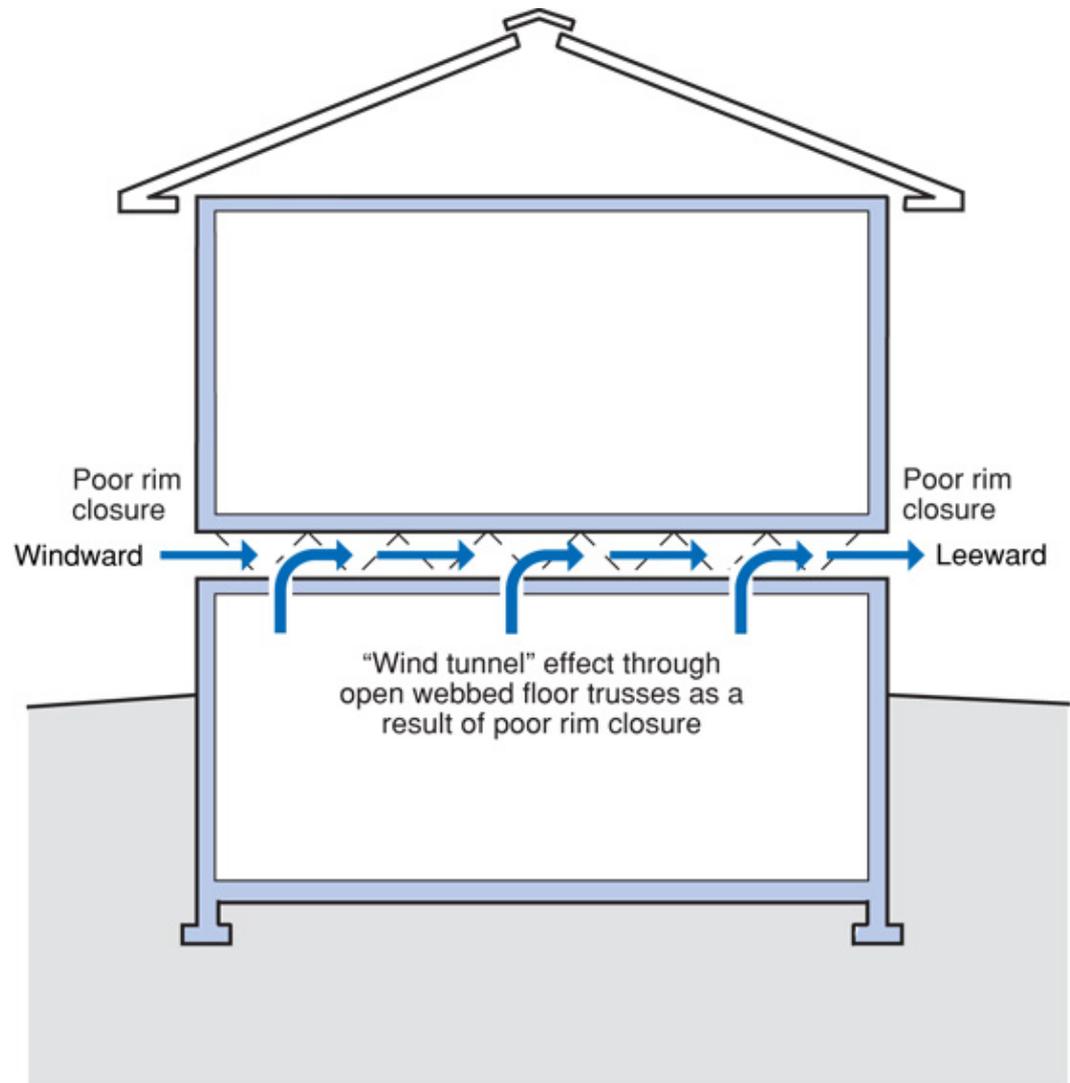


Figure 5.18
Wind Tunnel Effect

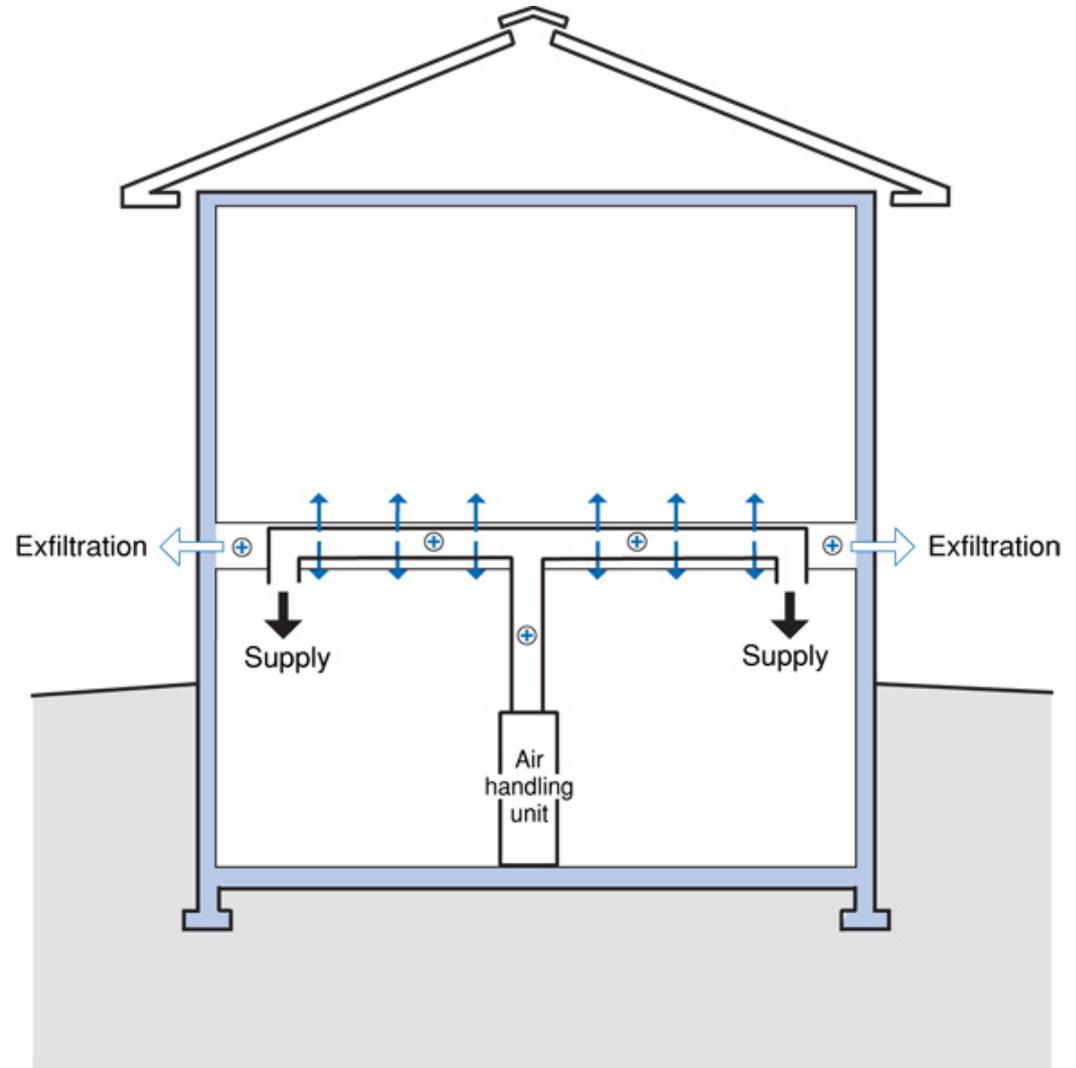


Figure 5.19

Supply Duct Leakage

- Leakage of supply ducts into floor space pressurizes floor space leading to exfiltration at rim closure

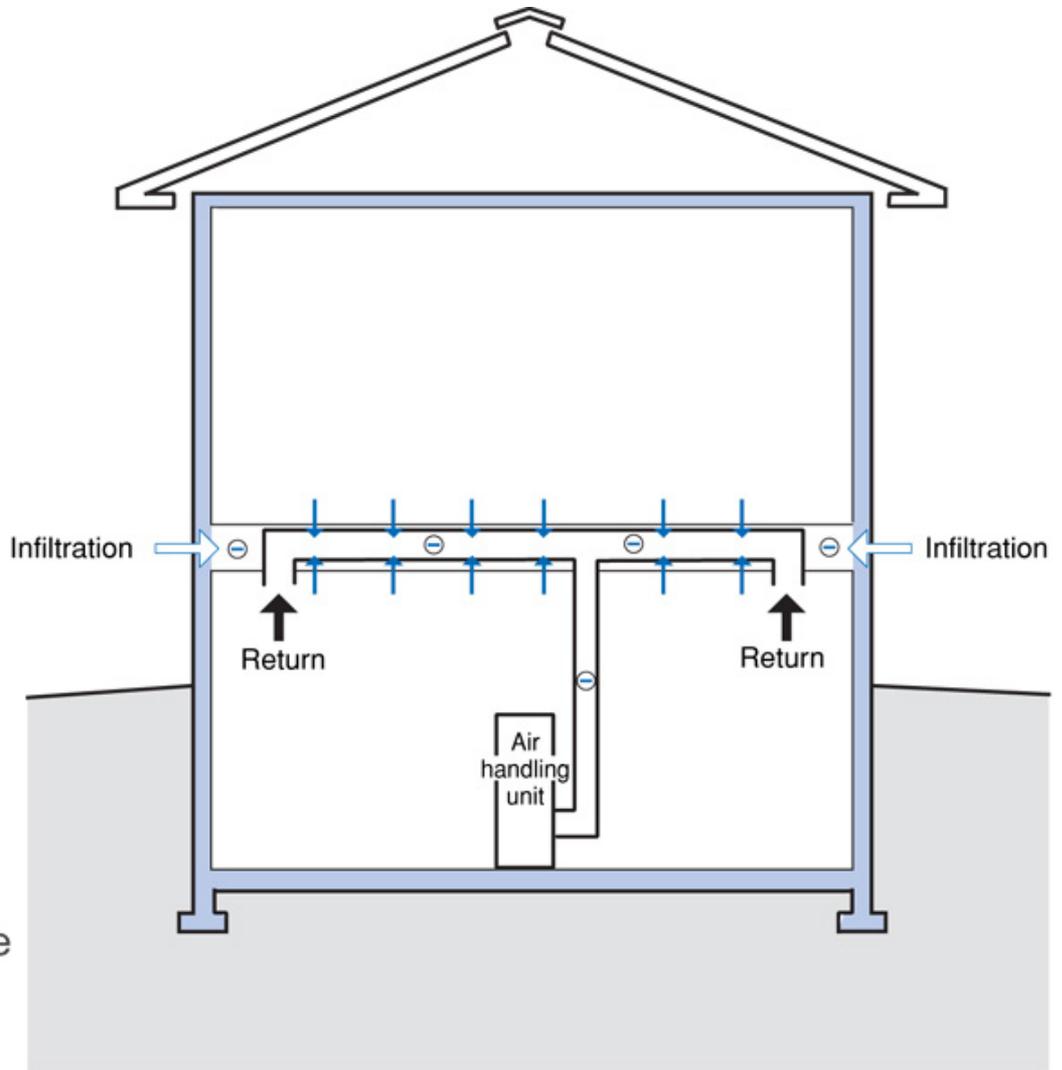


Figure 5.20

Return Duct Leakage

- Leakage of return ducts into floor space depressurizes floor space leading to infiltration at rim closure

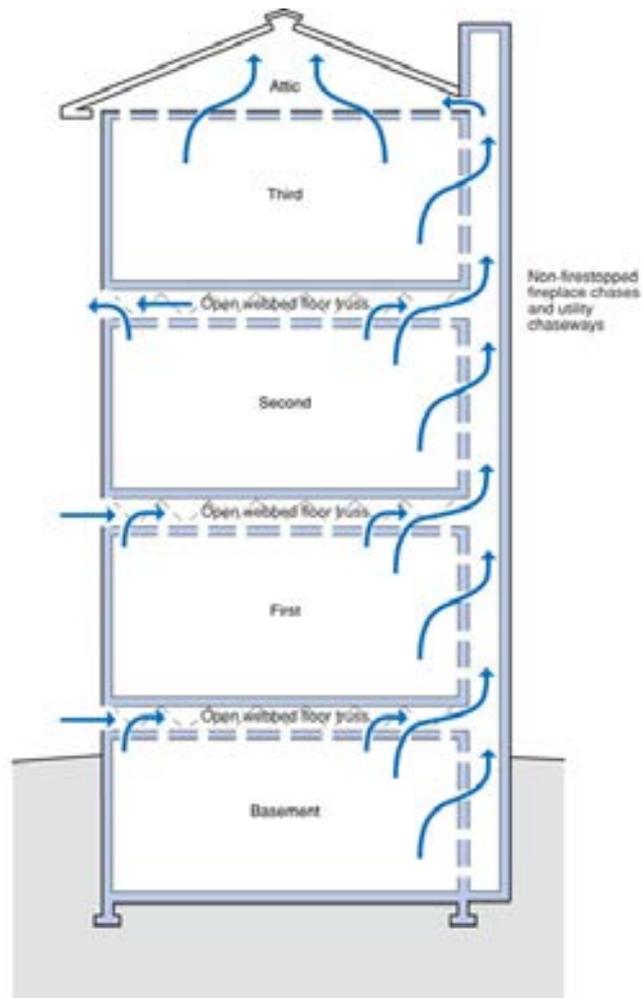


Figure 5.21

Combined Floor Paths and Pressure Drivers

- Vertical and horizontal communication of open webbed floor trusses through fireplace and utility chaseways
- Pressure drivers are wind, the stack effect and the operation of the HVAC system









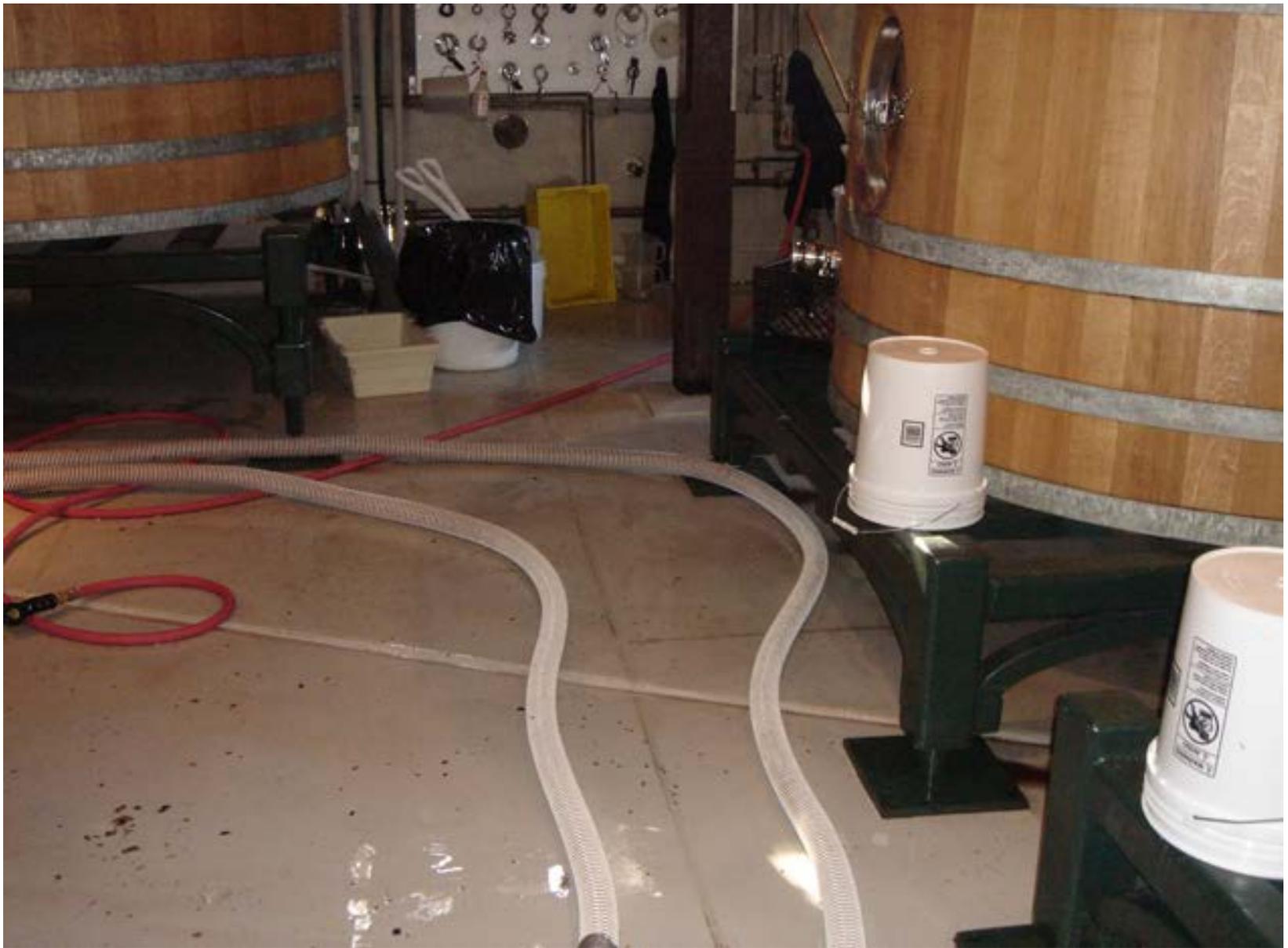












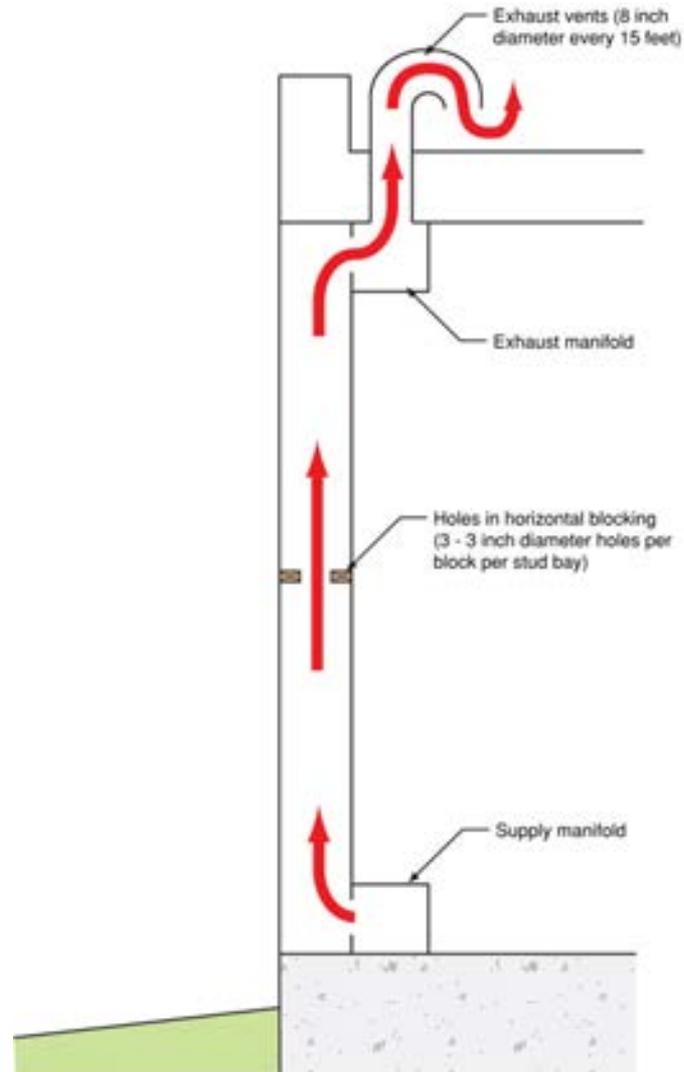


Figure 1: Ventilation of Exterior Walls

- Outside air supplied into supply manifold
- Exhaust manifold vented through roof

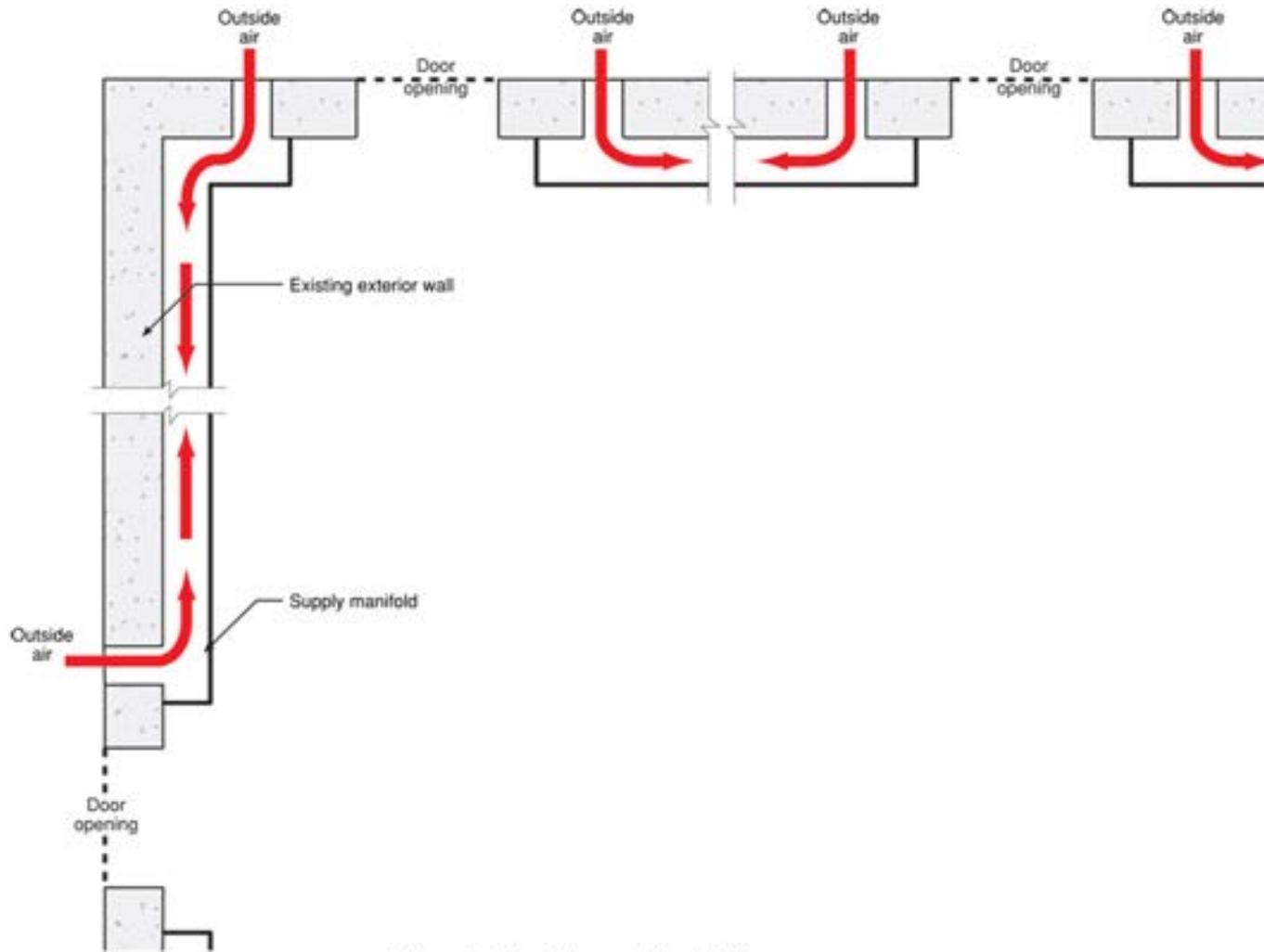


Figure 2: Plan View — Ventilation

- Outside air introduced into supply manifold at bottom of walls through vents penetrating exterior wall and stone
- Vents minimum 14 inches x 14 inches

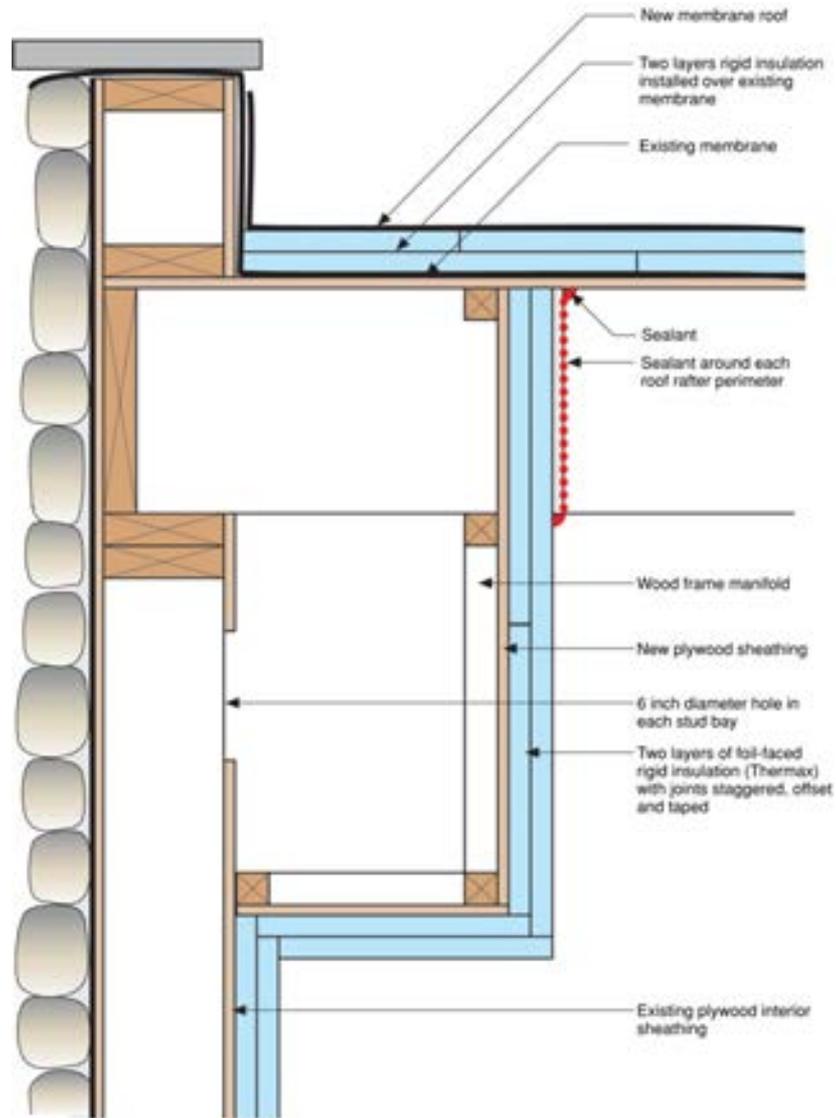


Figure 4: Exhaust Manifold

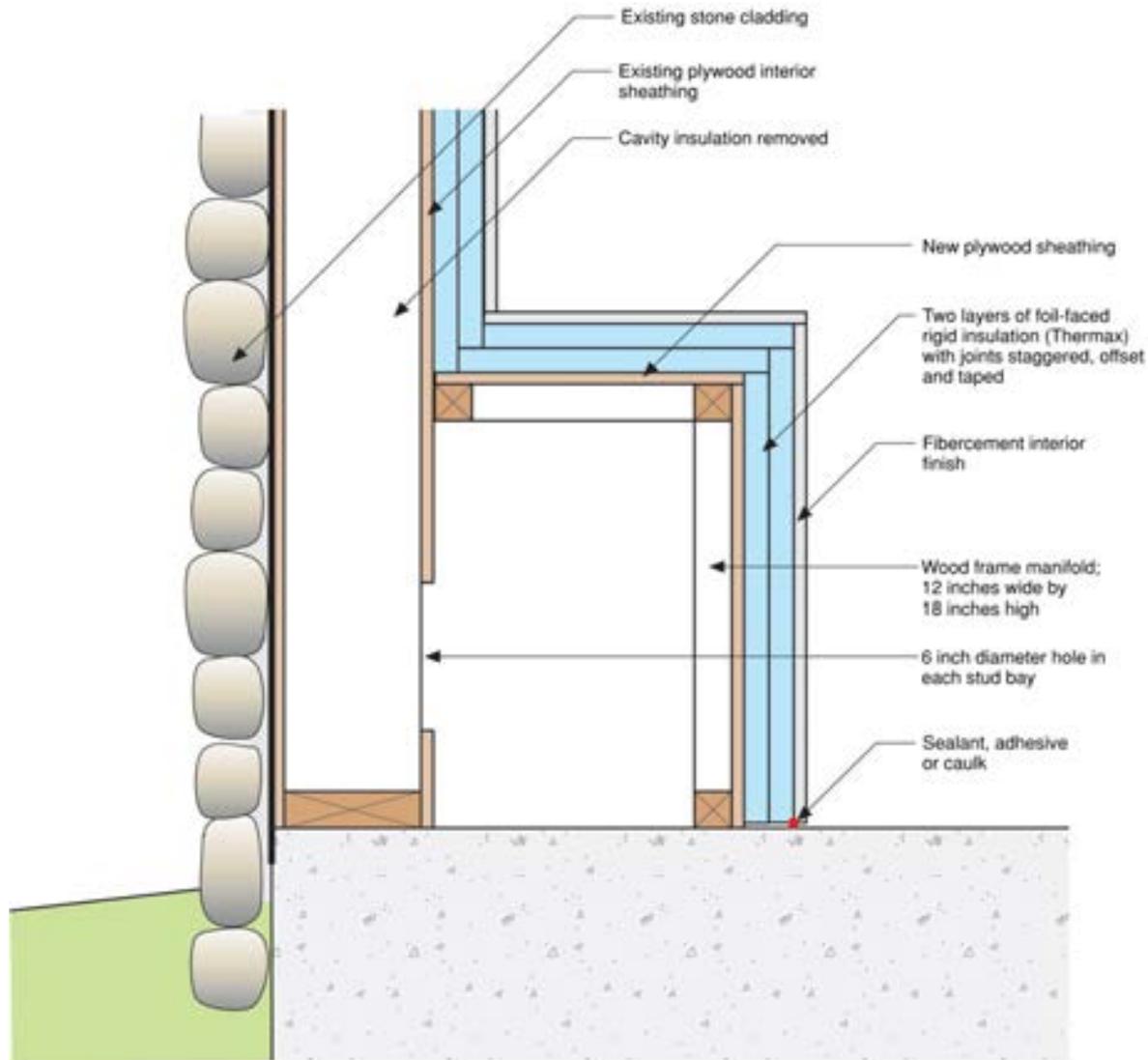


Figure 3: Supply Manifold







