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# **HVAC** for Low-load Buildings

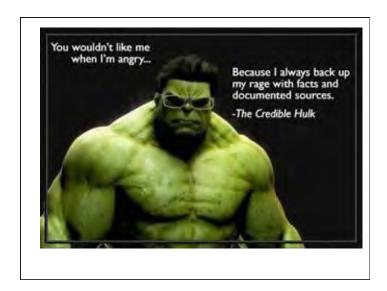
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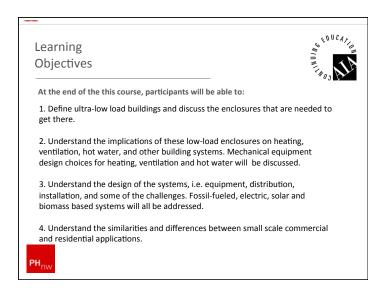


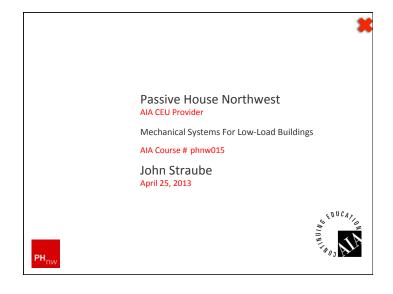
#### Course Description

Prof. Straube will define low and ultra-low load buildings and discuss the enclosures that are needed to get there. Next, the implications of these low load enclosures on heating, ventilation, hot water, and other building systems will be explored. Fossilfueled, electric, solar and biomass will be addressed. This session is intended for designers and builders of ultra-efficient residential (i.e. Passive House, DOE Challenge Home, and similar) and smaller commercial buildings.









#### The New World

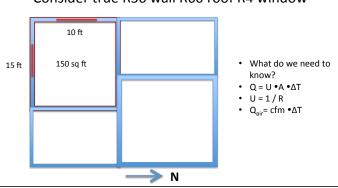
- Heating / cooling loads shrinking!
  - Better insulation, airtightness, windows
  - Smaller homes, townhomes, apartments, seniors
  - Multi-unit = small exterior enclosure area
  - New programs: NZE, PH, E-Star V3+
- DHW often larger energy demand than heat
  - Only efficient appliances can reduce DHW use

#### **Pacific Northwest**

- What's special about the PNW?
- Not very extreme cold or hot
  - Design temperatures?
- Moderate summer humidity
- Cheap electricity, cleaner ..

#### Example: Zones and rooms

Consider true R30 wall R60 roof R4 window



# Multi-unit Examples

- 20 x 30 ft = 600 sf 1 BDR interior apartment
  - 20\*9 ft height = 180 sq ft enclosure area
  - -40% windows = 72 sq ft
- R20 wall, R4 window, 20 F outdoor temp.
  - -(108/20+72/4)\*(70-20)=(23.4)\*50
  - 1170 Btu/hr total conduction losses (!)
- Achieve 0.40 cfm/sq ft @75 Pa airtightness
  - 18 cfm leakage natural = 950 Btu/hr air leakage loss
- Ventilation (New World needs it)
  - 30 cfm w/66%HRV = **1500/500 Btu/hr ventilation**

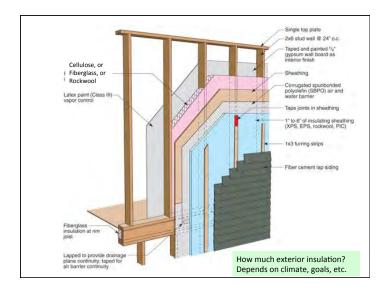
# Room/zone

- (15'+10') \* 9 ft high = 175 sf of wall
- If 5'x5' and 5x6 windows = 25+30 = 55 sf windows
- Therefore: 120 sf wall, 150 sf roof, 55 sf window
- Heat loss (ΔT=50 if 20°F outdoors)
- Wall 120 / 30 \* 50 = 200 Btu/hr
- Windows 55/4 \*50= 687 Btu/hr
- Roof 150 / 60 \*50 = 125 Btu/hr
- Total skin loss = 1012 Btu/hr
  - can be met by 17 cfm air @ 130F /54C

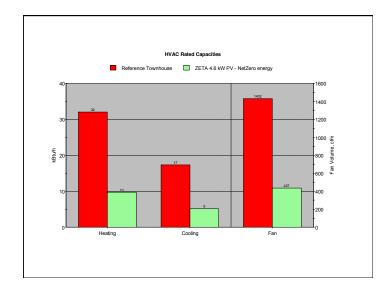
One therm = 29.3 kWh

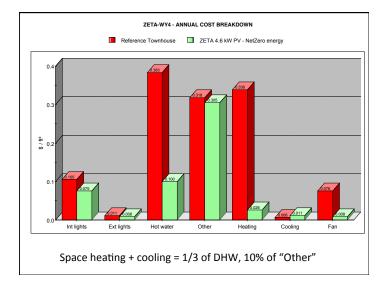
#### Simple Heating Analysis Apartment

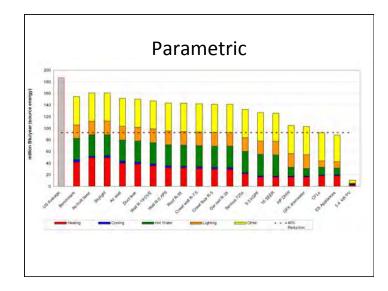
- Peak design load: 2.5-3.5 kBtu/hr (<1 kW)
  - Corner apartment up to 4-5 kBtu/hr (1.5 kW) ....
- Heat loss coefficient 50-70 Btu/F/hr
- If we use HDD65 = 5000
  - -(50 to 70)\*24\*5000 = 54-75 therms < \$100/yr
  - 1465-2200 kWh/yr <\$160/yr
- If we use HDD50=1500 .... Negligible
- If 2.5 kBtu/hr, airflow= 50 cfm @DT=130F











#### High-performance Enclosures for PNW

- Specs will change with climate, goals
- Walls?
  - Usually over R20, perhaps as high as R30
- Roofs?
  - Depends on construction. R40-100
- Windows?
  - R4 min, often R5-7
- Airtightness
  - Under 1.5 ACH@50

#### Low-Load Definition #1

- Peak design loads are smaller than smallest commodity central units
  - Eg less than 25-30 kBtu/hr furnace
  - 1.5/2 ton AC (18-24 kBtu/hr)
    - 2 ton is the smallest efficient affordable model

#### Low-load Definition #2

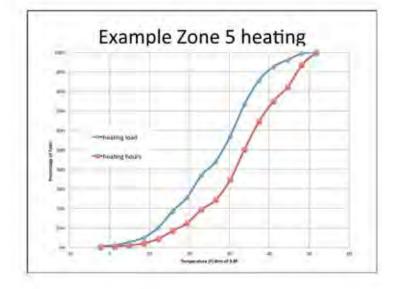
- Peak heating loads in the range of 10-20000 Btu/hr for PNW
- Or... peak heating power density of under 10 Btu/hr/ft<sup>2</sup>
  - Typ. house of 1000-2000 sf means ...
  - PH 3.15 Btu/hr/ft<sup>2</sup>
- DHW load often exceeds space heating load
- Mechanical ventilation almost always required due well-built airtightness

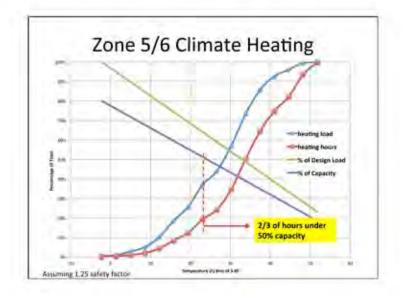
# Low Load - more implications

- Internal / solar gains have a BIG impact on space temperature (i.e., Passive gains)
  - Eg. Assume SHGC (g)=0.60
  - 6'x6'8" patio door with 80% glass, 20% frame
  - 6000 Btu/hr in bright sun! (1/2 ton AC in one room)
- · Different zoning may be needed
  - Room by room assessment
  - Mixing between rooms can be more effective
  - better enclosure for peaks? (e.g.lower SHGC glazing)

# So what's the problem?

- . Smallest condensing furnaces are 40 kBu/hr
- Two-stage furnaces allow for low stage fire at 30 kBtu/hr
- · But most hours are at fractions of peak design
- How does the system work with a hourly heat loss of 5 to 10 kBtu/hr?
  - Runs for 10 to 20 min/hour (two fires/hour?)
  - Short cycling (wear & tear, inefficiency)
  - But must provide ductwork for 30 kBtu/hr





### So what's the problem?

- If capacity >> demand
  - Overshoot temperatures, too hot in heating, too cold in cooling
  - Short-cycling kills AC durability and efficiency
  - Need modulation or thermal mass (or water)
- Cannot save money due to small size
  - Ductwork still largish (eg, say 800 cfm)
- Min. monthly charges of two utilities
  - Can dramatically increase cost

#### **Domestic Hot Water**

- DHW > Space heating in efficient apartments
- DHW exceeds space heat in efficient small house
- Typical US household (census data)
  - 4000 kWh demand (136 therm)
  - National consumption 5600 kWh (192 therm)
- Typical 5 unit + building. Per unit
  - 2500 kWh demand (86 therm)
  - 3575 kWh/yr estimated use (122 therm)

#### Some Goals Limit solutions

- Net Zero Energy houses: PV is hence preferred for on-site generation
  - Easy measurement and on-site generation drives solutions to all-electric
  - Solar thermal may be as expensive per Btu!
  - Small wind turbines often more expensive
- Passive House
  - Limiting heating demand not strongly correlated to low energy house
  - Calculation tool encourages high solar gain

### Summary

- High performance enclosures reduce heating and cooling loads
- · Low load means different solutions for HVAC
- DHW remains large
- Must balance
  - HVAC
  - Enclosure
  - Renewable

#### **HVAC** for Low-Load Houses

#### Introduction

- No one solution is perfect
- Depends on
  - building size, shape, etc.
  - New or retrofit?
  - Gas available or all-electric?
  - Trades and equipment availability
  - Money available
  - Comfort and IAQ goals

#### **HVAC**

- People want comfort
  - Surface temperatures, humidity
  - Heat, cool, humidity
- People assume health
  - Require fresh air = require ventilation
- Don't want to pay too much
- Don't want to do maintenance

#### **HVAC Functions**

Five Critical functions are needed

- 1. Ventilation
  - "fresh air"
  - Dilute / flush pollutants
- 2. Heating
- 3. Cooling
- 4. Humidity Control
- 5. Air filtration / Pollutant Removal
  - Remove particles from *inside* and outside air
  - Remove pollutants in special systems

2013-04-29

#### **HVAC Constraints**

- Safety
  - Combustion, explosion, scalding
- Health
- Comfort
  - Temperature, humidity, air speed, noise, light
- Reliability
  - Maintainable, long term performance,
- Efficiency
  - minimum of additional energy
- Economy
  - Builder/owner can afford

2013-04-29

34

# ASIDE: Ratings game

- EER
  - BTU/hr output to W input
  - Conditions: 95F outdoor, 80F return
- SEER
  - Seasonal EER
  - 82F outdoor, 80F return
- COP
  - Watts out to Watts in
  - HSPF (Watts in Btu out @ 47°F)

#### Interactions Interactions

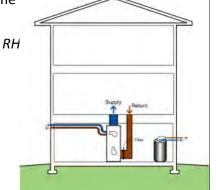
- BEWARE:
- "Perfect" solution for heating may not solve cooling
- "Perfect" cooling solution may not solve DHW supply
- Perfect heating+cooling+DHW may do nothing for ventilation!
- We need
  - heat+ cool + DHW + vent + filtration + humdity

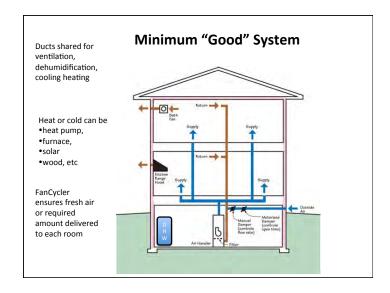
# • All rooms the same • No ventilation

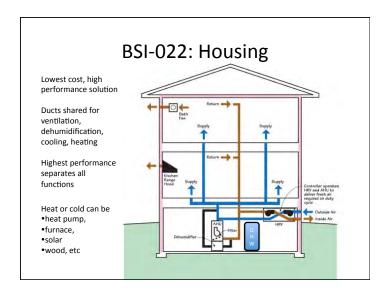
• In AC, accidental RH control

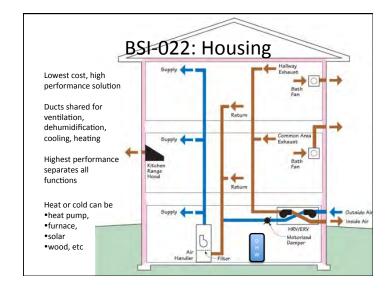
NOT acceptable for high-performance homes

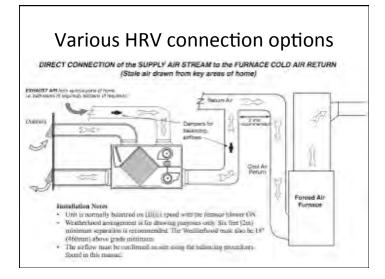
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#### **Small Residential HVAC**

- Cooling DOES NOT mean humidity control
- Energy removal for lowering temperature:
  - Sensible energy
- Energy removal to condense water vapor
  - Latent Energy
- Ratio of Sensible Heat Ratio =SHR
  - Normal cooling equipment 65% sensible
  - As enclosures become energy efficient the required SHR drops and latent
  - becomes more important!

Heat / Cool Production

#### **Heat Production**

- · Boilers: heat to water
  - Old types heated water to steam and distributed
  - Modern heat water to 35C (95F) to 85C (190 F) and pump water using small electric pumps
- Furnace: heat to air
  - Air is heated to min 40 C (110 F) and usually 50(130F)
  - Electric fan is used to move air
- · Both heat exchanger between flame to fluid
- Fuel sources
  - Nat gas, oil, propane, wood, electric, etc.

2013-04-29

44

# **Condensing Furnace**

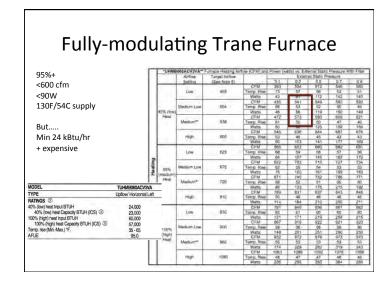
- Simple, reliable, lots of service available
- Cheap
- Usually works at near rating condition
- Eg 95% efficiency
- Specify efficient fans
- Use efficient ducts

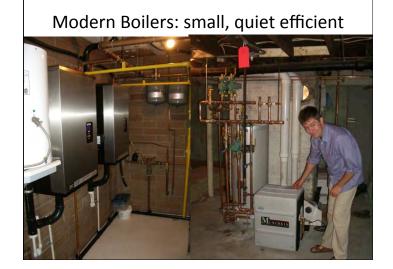




#### **Small furnaces**

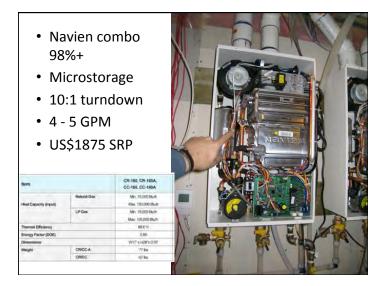
- Most products output 40 kBtu/hr or more
  - 40 kBtu= 750 cfm @ 50F temperature rise
- Some modulating products have lower outputs, e.g.
  - York YP9C (20KBtu) \$2500
  - Trane XC95M (23 kBtu) \$3000
  - Carrier 58MVC, Rheem RGGE, Lennox SLP98DFV
- Small two-stage can be better
  - Goodman GMH90-45 (30/44 kBtu) \$<1000
- Modulating furnaces can't "lock out" high output require duct sizing for 65-70 kBtu!















### **Condensing Boilers**

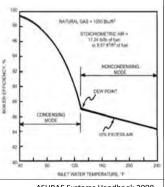
- Supply temperatures of max 140F (60C) under design conditions
  - ensures return temperature low enough to get condensing (=efficiency)
- Lower is better!
  - Outdoor reset
  - Variable speed pump + Delta T controller
  - Varibale speed pump +

# **Boiler Combustion Efficiency**

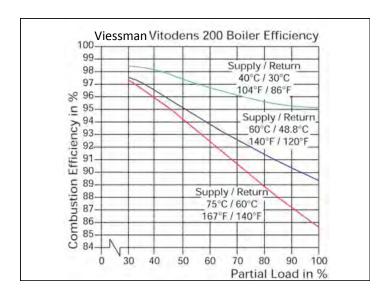
- Most combustion is >99.9% efficient
- Equipment varies on ability to extract useful heat from combustion via HX
- Heat exchanger size is important
- Temperature of entering fluid is also critical
  - Condensing furnace (72 F / 22 C)
  - Condensing boiler >90% (<110 F / 45 C)</p>
  - Normal boiler <85% (>130 F/ 55 C)

# Condensation % Efficiency

- Depends on return temperature
- Terminal equipment that can return low temps aid efficiency
- Target 95-110 F (35-43 C)







# Consequence

- Furnaces: return air temperatures = room temperature (70 F/21C)
  - Hence, condensing, 95%+ efficiency practical
- Boilers: depends on system design/operation
  - Radiant panels: 90-120 F / 32-48 C
  - Fan Coils: 100-180 F /40-80 C
    - Will not condense if T > 135F/55C
  - Baseboards: 120-180F+ supply

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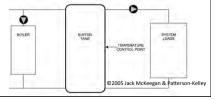
# Combo Systems

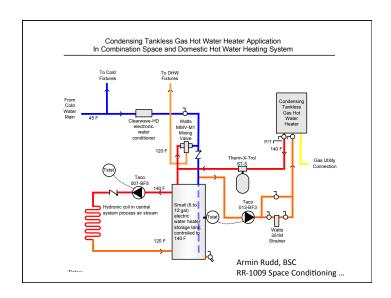
- Condensing Tankless heaters
  - Beware minimum output
  - Most units are 15 to 35 kBtu/hr minimum
  - Eg. no lower than a furnace
- Unless storage is provided, min output equals min output of heating system
  - This means duct sizes, coils, etc.



# **Combo System Warning**

- Provide buffer capacity, eg a storage tank
- Limits short-cycling when loads are small (eg 10-30% of min. boiler output)
  - Allows for very small demand systems
- Buffer tank avoids cold slug complaints





#### Combi

- small buffer tank
- Adds some standby losses





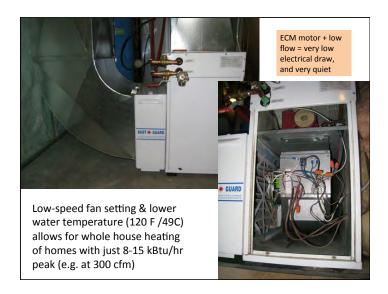




# Condensing tanked • several suppliers Rheem Rudd Advantage Plus 95% eff 100 kBtu/hr 45 gallons Stainless tank. Expensive. \*\*\*Interview of the process of

# Combo Fan coils / Air Handler

- Operate at over 100F (38C) air temperatures to avoid "cold blow" drafts
- Ensure low return (under 120F) to get condensation in condensing boilers
  - Small pump flow
- Lower speed jet (200 fpm), high supply location recommended
  - Higher supply temperatures if you don't do this

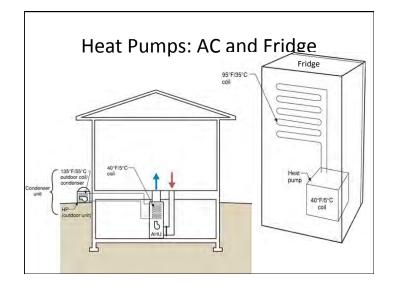


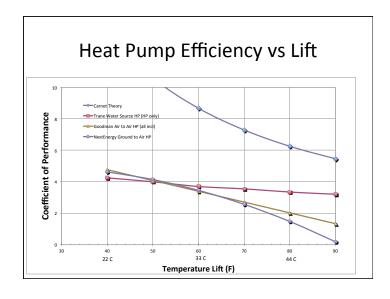
#### **Heat Pumps**

- Neither create or destroy heat, but move it around
- Require input energy just like any other pump
- Need
  - Source of thermal energy
  - Sink of thermal energy
- Sources (inside=cooling, outside=heating)
  - Air ("Air source")
  - Ground ("ground source")
    - Soil, Groundwater, or Surface water (eg lake)
  - Wasteheat in building via exhaust air or drain water

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# Heat Pumps Use compressors, and refrigerant ("Freon") All use internal heat exchangers to transfer hot or cold refrigerant to water or air Terminology "Air to air heat pump" = "air-source" "Water-to-water heat pump" "air conditioning" Water to air Ground source "Geothermal"





# Split System Heat Pump and Reject/ Collect in same box

• Compressor, and DX coils in one enclosure



# Cooling

- Most cooling equipment is a heat pump
  - uses the interior as a source (collection) and
  - Outside as the sink (rejection)
- Heat pumps do cooling and heating
- Challenge to get single speed units to be appropriate for both

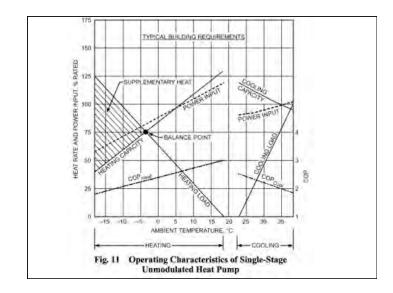
2013-04-29 76

#### Dehumidification

- Cooling will often require supplemental dehumidification
- This requires cold surface: eg fan coils, not radiant ceilings/floors!
- Separate dehumidifer is common
- Multi-speed AC may be sufficient in marginal cases (including mini-split)

# Heat pumps in heating mode

- Major reduction in heat output as outdoor temperature drops
- COP drops as outdoor temperature drops
- Typically designed for a "balance point" and then used electric "strip" heat
- Modern design avoids strip heat



April 25, 2013

Heat+cool: Ducts provides distribution, can add ventilation, no DHW

# **Split Heat Pumps**

- An option for Zone 3-4?
  - Eg Portland, Seattle, Tacoma 20 F design temp
- 2 ton HP produce about 16 kBtu/hr @20F

SSZ150241A\* / CA\*F3636\*6A\* + TXV / MBE1600\*\*-1 e.g., Goodman SEER16 model

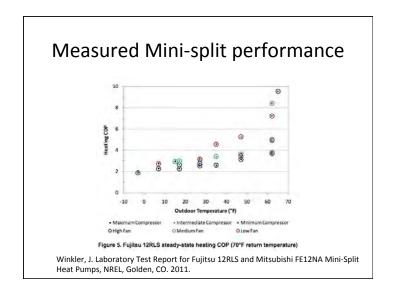
	Outdoor Ambient Temperature															
	85	60	.55	50	47	45	46	35	30	25	20	17	15	10	-5	0
MBh	30.2	28.6	26,9	25.1	24.0	23.3	21.6	19.9	18.7	17.1	15,9	15.0	14.5	13.0	11,5	10.0
ΔT	31.9	30.2	28.4	26.6	25.4	24.6	22.9	21.1	19.8	18.3	16.8	15.0	15.3	13.7	12.2	10.6
kW.	1.79	1,75	1.72	1.68	1.7	1.65	1.82	1.58	1,68	1.84	1,60	1.58	1.56	1.52	1.48	1,45
Агора	8.4	7.8	7.3	6.9	6.7	6.6	6.2	5,9	57	5.4	5.2	5.1	5.0	4.7	4.4	4.2
COP	4.93	4.78	4.57	4.37	4.22	4.13	3.91	3.69	3.26	3.08	2.91	2.79	271	2.49	2.27	2.03
EER	16.9	16.3	15.6	14.9	14.4	14.1	13.4	12.6	11.2	10.5	9.9	9.5	9.3	8.5	7.7	8.9
HI PR	349	334	322	307	300	295	283	272	280	249	239	233	229	220	212	203
Lo PR	144	133	125	115	108	104	196	85	77	89	80	.66	-54	46	40	25

Seasonal COP 3-3.5, cooling included, standard equipment, <<\$3000

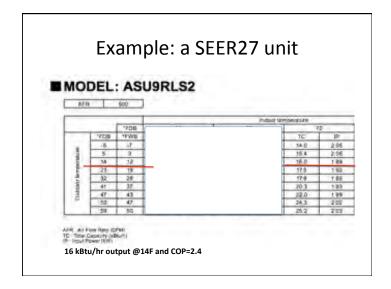
Some split-systems might work

| The control of the





# Modulating / staged heat pump • Loose less output as temperature drops • Always loose efficiency (COP drops) • Usually avoid electric heat, or supplement it



# Mini-split distribution

- Heat distribution from single head?
- Aesthetics of exposed heads
- Some hidden "slim duct" units exist but efficiency suffer
- Open doors between spaces really helps



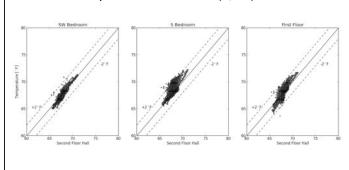
# Distribution from point sources

- Mini-split first floor only (heating)
- Installed 2<sup>nd</sup> floor for cooling
- Measured temperature distribution from bedrooms to hallway
- Work by Kohta Ueno / Dan Bergey
- Carter Scott NZEH
- unoccupied



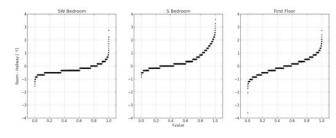
# **Temperature Distribution**

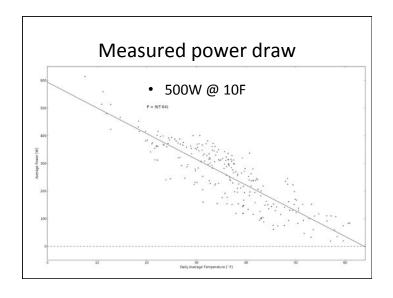
• little temperature variation (+/2F)



#### Distribution

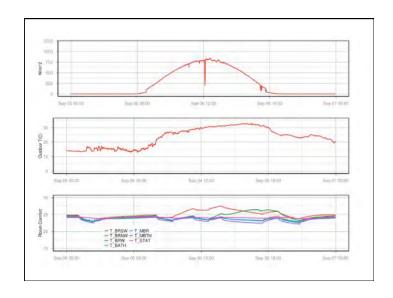
- SW/S bedroom was sometimes 2-3F warmer than hall
- Solar heating through SHGC=0.2 windows

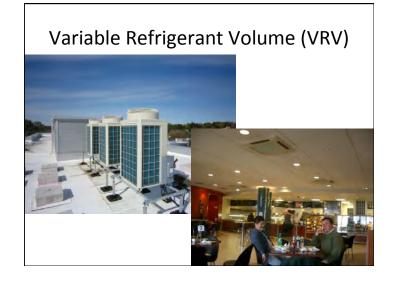




# More Distribution examples

- 2400 sf high performance home in Philly
- Sunny day, near peak cooling load
- 2 ton AC (about 2x what is needed)
- Temperature variations exceed 5F/3C from thermostat
- Solar gain in Southwest Bedroom results in peak load





# **Emerging alternate systems**

- Variable speed outdoor unit (VRV) (18 & 24)
- Two-speed indoor fancoil for ducts (ECM fan)
- 18 kBtu/hr model
  - Operates at 600/420 cfm
  - 12 kBtu/hr low speed
  - Up to 20 kBtu/hr heating

Air Handling Unit

FTQ-PA + RZQ-P9

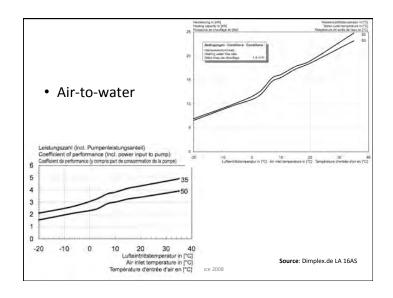
Split System Air Conditioners

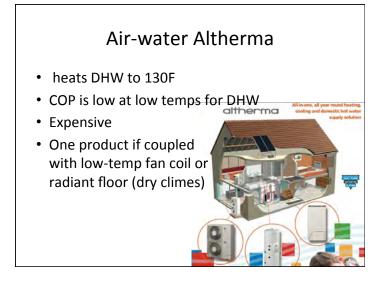
DAIKIN AC (AMERICAS), INC.

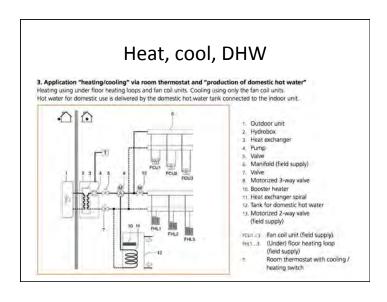
#### Chillers

- Air-water heat pumps used for cooling water
- Big units use cooling towers
- Usually large buildings
- "reverse-cycle chiller" is another name for a water-to-air heat pump









# Numerous systems available • but not in Canada/US ③ Therma V + Radiator + Underfloor Heating + Sanitary Tank

# CO2 Refrigerant air-to-water

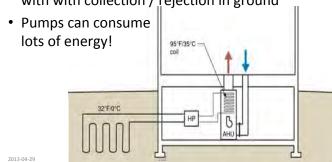
- allows true hot water (>140F/60C)
- Operates to low (-10F/-25C) temperatures
- Cant buy in north America 🕾





## **Ground Source Heat Pump**

 A water-to-air or water-to-water heat pump with with collection / rejection in ground



#### **GSHP** Geothermal

- Can buy small capacity systems (eg 1.5 2 tons)
- Many benefit from water storage tank
- Cost is challenging: just heat cool but often total system cost of \$20K+
- Desuperheaters don't help DHW much
  - Low load houses GSHP run little

#### **Electric Resistance**

- Electric heat
  - Cheap to buy, high operating cost, maybe hi GHG
- Baseboard / Cove
  - Impact on space design
- Radiant heat mats (heat does not rise)
  - Floor/ceiling
  - 10-15 W/sf capacity
  - Need 300-600W per room (30-60 sf)

#### **Pellet Boilers**

- Can be an option for heating and opt. DHW
- 8-50 kBtu/hr, modulating, some sealed combustion





#### **Domestic Hotwater**

Difficult to separate from design of HVAC in low-load *residential* buildings

# DHW – Health & Safety

- Require water temps over 120 F (50C)
  - 66 °C (151 °F): Legionellae die within 2 minutes
  - 60 °C (140 °F): Legionellae die within 30 minutes
  - 55 °C (131 °F): Legionellae die within 5 to 6 hours
  - 50 °C (122 °F): They can survive but do not multiply
- Showers are primary indoor residential vector
- Scalding 130F
  - 10 / 30 seconds for child/adult 3<sup>rd</sup> deg burns

#### **DHW**

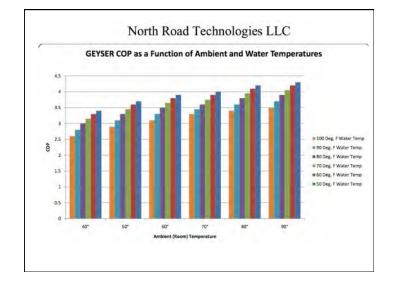
- Heat pumps
  - difficult to achieve >120F efficiently
  - Need to use R134a/R507 to get hot
- Gas combustion
  - High capacity and >130F easy
- Electric
  - Expensive to operate
  - Point-of-use requires large kW service





# Performance: "depends" • Work well in warm spaces - Eg boiler rooms • Dehumidify basements in summer • Cool basements in winter • Steal heat from house - Is free heat available?

# Add-on Heat pump • eg Geyser. Allows unit location near heat source (fireplace? Sunroom?)



# DHW efficiency

- Gas Condensing can only happen with low entering temperature
- · Long pipe runs can eat up energy
  - Small pipes help
- Heat Pump Water Heaters
  - Depend on where you are

## Distribution of Thermal Energy

# Air-based Energy Delivery

- Heat Capacity: Energy required to raise the temperature or released when a material is cooled
  - Air heat capacity: 0.240 Btu/lb/F.
  - Air density: 0.074 lbs/cf @ room temp = 0.018 Btu/cf/F
  - 1 cfm = 60 cubic feet per hour
  - So... heat delivered per cfm
  - =  $60 \times 0.018 \approx 1.1$  Btuh/cfm/F (1.2 W/lps/C)
  - Usually use 1.05 for cool air, 1.08 for warm air

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#### Air-based 2

- Cooling air supply 55 F, and room air 75 F
  - 1.1 (75-55) = 22 Btu/hr/cfm
  - Need more flow for cooling than heating
- Heating return 70 F

- Furnace 130 F: 1.0\*50= 50 Btu /hr/cfm

- Heat pump 100 F: 1.0\*30 = 30 Btu/hr/cfm

- Therefore need 5/3 more airflow for low temp air

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#### Fans

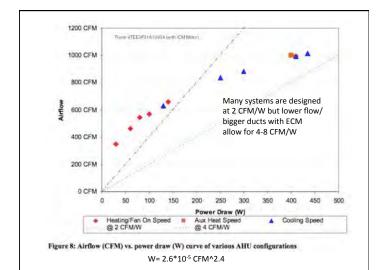
- Efficiency
  - Rating: Watt per cfm (or cfm per Watt)
  - Higher pressure = higher power requirement
  - Power (W) = Flow rate \* Δpressure / efficiency
  - -HP = cfm \* Inch Water / (6356 \* eff)
  - Efficiency: 0.4 (good) to 0.65 (best)
- Energy: 0.25 to 1.5 W/cfm for ducted systems
- Reduce pressure or flow required = direct energy savings

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#### Fan Laws

- 1. Increase RPM = direct CFM increase
- 2. Static Pressure increases RPM<sup>2</sup>
- 3. Horsepower increases with RPM<sup>3</sup>
- Double pressure means 1.41 times RPM
- Requires 2.8 times horsepower
- Energy saving designs use low CFM and/or Low ΔP

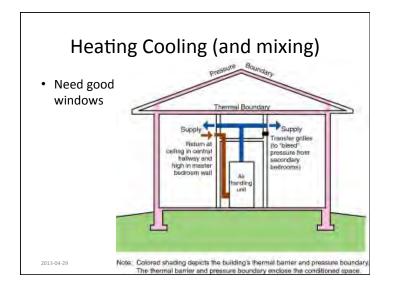
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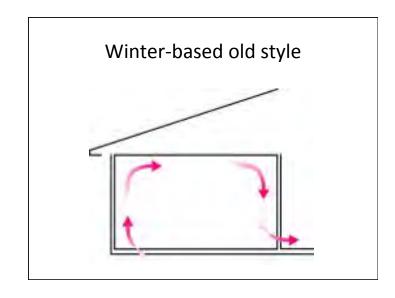


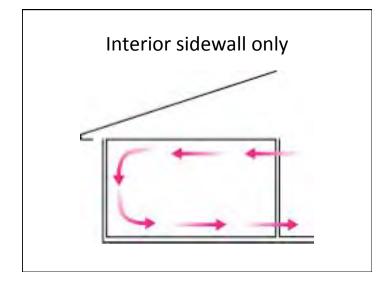
# Reducing duct friction

- Reduce velocity
  - Increase duct area!
- Fittings are major source of friction
  - Larger radius bend
- Simplify duct runs if possible

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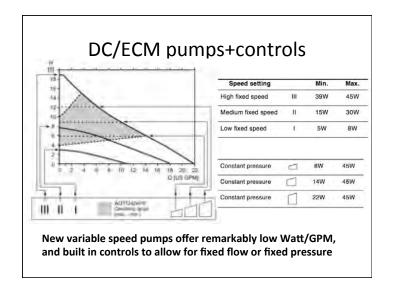






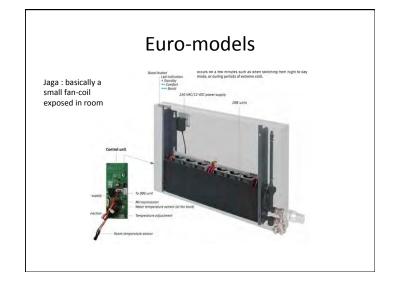
# Water-Based Systems

- Water moves...
  - 500 Btu/hr/GPM/F
  - 375 Btu/hr/GPM/F (Glycol)
  - Radiant floor
    - 100 F supply 90 return <= 5000 Btu/hr/GPM
- Example: 30 000 Btu/hr
  - Furnace: @ 50 Btu/hr/cfm  $\rightarrow$  600 cfm (300W)
  - Heat pump @ 30 Btu/hr/cfm → 1000 cfm (500W)
  - Radiant 5000 Btu/hr/GPM → 6 GPM (40W)
- But, good design/spec can deliver 600 cfm@150W



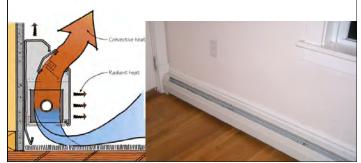
# **Energy of distribution**

- Furnace: 1000 cfm 60 000 Btu/hr
  - Fan 300-800W (=1000-2700 Btu/hr)
  - 1.5 to % of energy delivered
- Heat Pump 1000 cfm 30 000 Btu/hr
  - Fan 300-800 W (4 to 9%)
- Radiant floor
  - Pump 85W 10 GPM 50 000 Btu/hr (0.6%)
- Distribution energy can vary by 5X to 15X



# Convector / Radiant

- Usually only for heating
  - larger Delta T need to drive buoyancy



# Convector / Radiator

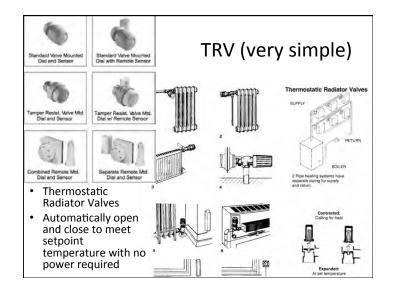
- Hydronic terminal units
  - no energy required at unit

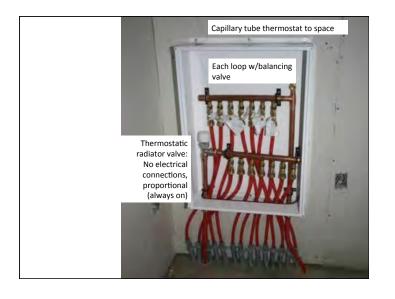


# Low-temperature baseboard

- Typical convectors
  - rated at 180F / 80C (mean or supply)
  - return temperatures >160F /70C
- Want to supply with lower temperature!
  - Condensing boilers and heat pumps only work with supply temperatures under 140F/60C
- Must increase SIZE of convector to reflect lower supply water temperature







#### Radiant floors in low-load houses

- Radiant floors wont heat up enough to be noticed
  - This is not barefoot friendly
- Still, zero-noise, no maintenance



# Emission plates under wood

• Not as effective as topping. Requires higher water temperatures.



# **Heat Exchange from Surfaces**

- Example: 80F (27C) floor, 72F (22C) room air
  - 15.2 Btu/hr/ft<sup>2</sup> heating
- Example: 60F (15.5C) ceiling, 74F (23C) room air
  - 26.6 Btu/hr/ft<sup>2</sup> cooling (500 sf/ton)
- Example: 68F floor, 74F air (1500 sf/ton)

	heating		cooling	
	Btu/hr/ft²/F	W/m <sup>2</sup> K	Btu/hr/ft²/F	W/m <sup>2</sup> K
floor	1.9	11	1.2	7
wall	1.4	8	1.4	8
ceiling	1.1	6	1.9	11

#### Radiant Floor "Self-control"

- Low temperature radiant has some self control
- Huge practical control and comfort benefit in <u>low</u> heat flux radiant floor and ceilings
- If room rises 1F @ low load, heat output drops 38%!

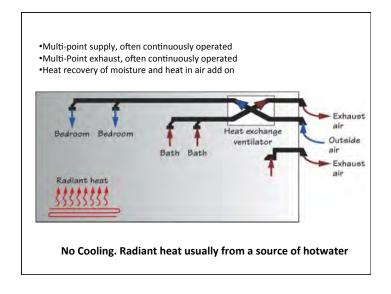
Heating Power		Room 70F Floor Temp	21.1C erature	Power outp 1F/C room	Percentage change	
Btu/hr/ft2	W/m2	(F)	Celsius	Btu/hr/ft2	W/m2	Output
5	15.8	72.6	22.5	1.9	11	38%
10	31.5	75.3	24.0	1.9	11	19%
15	47.3	77.9	25.4	1.9	11	13%

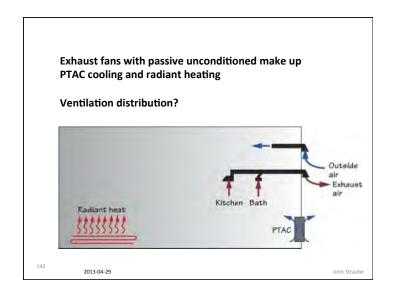
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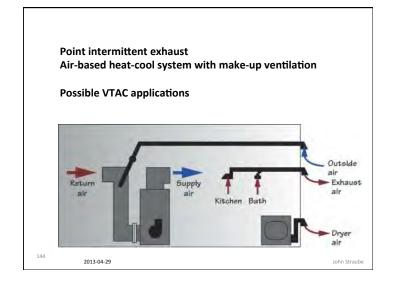
# Ventilation

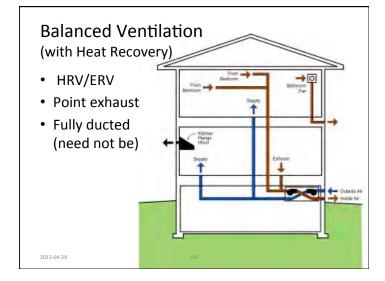
#### Intro

- Require fresh air for health and humidity
- ASHRAE 62.2 latest
  - -7.5 cfm/person + 0.03 / sf
- Therefore
  - -3 BDR / 2000 sf = 90 cfm
  - Was 50 cfm until recently

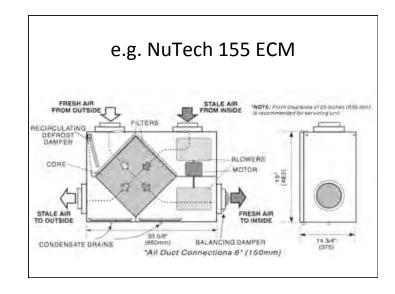












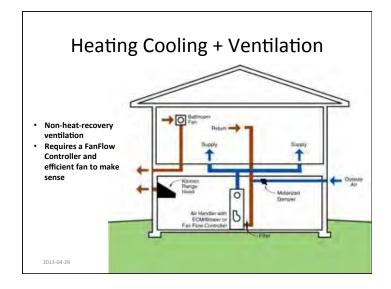


- Mid-scale HRV
- Emerging tech
- 200-600 cfm
- Need to watch fan energy!



# HRV/ERV

- Heat Recovery Ventilator
  - This is a ventilation system that recovers heat from the exhaust air and transfers to incoming air
- Enthalpy/Energy Recovery Ventilator
  - Transfer heat and humidity from incoming to exhaust
- Both, beware poor electric motor efficiency
  - Aim for less than 1 W/cfm



#### Multi-unit Issues

- Metering: per suite or per building
- Fuel-Source: Gas or all-electric
  - Carbon? Dollars? Energy?
- DHW or just space heat?
- Is Cooling necessary?
- Grouping: Central, unit, or mix?
- Equipment owned per suite or per building?
- Perceived access to apt issues?

#### Central vs. Distributed

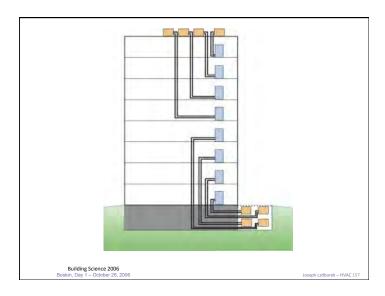
- · Central systems often
  - reduce capital cost per unit output of *plant*
  - Increase distribution costs dramatically
  - Increase distribution energy losses
  - Decrease redundancy
  - Increase complexity
  - Make sub-metering expensive/difficult
  - Take advantage of load diversity



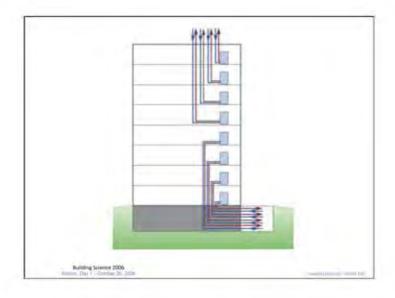


# **DHW** Distribution

- Distribution losses
  - Can be significant for long runs
  - Recirculation pumps increase loss unless controlled
  - Large pipe diameters store lots of water
    - Use smaller pipes









# Schools • E.g Double-loaded corridor or exterior corridor • One wall + roof exposed/class • Small systems work well per class - mini-split + HRV - Ventilation control / class - Individual control of temperature! - Lots of redundancy, easy to maintain

#### **Conclusions**

- This is still complex
- No simple or easy solutions

# Cooling

- Need variable speed / staged small units
  - Ductless mini-split on upper floor only?
- Separate dehumidifier required in hot-humid weather
  - Could be DHW heat pump!

#### Choices

- Furnace is still a good choice if you have natural gas and loads over 20 kBtu/hr
  - Choose smallest condensing unit, lock out high fire
- Combo Systems
  - Use high-efficiency DHW system to provide heating
  - Space heat can be fan coil, radiator, floor
  - Can be integrated into ventilation, filtration
  - Add cooling coil
- Size of duct/coil often fixed by cooling system