



July 31 - August 2
Westford, MA

Airflow Research

Flex Duct vs Sheet Metal

**Building Science
Summer Camp**

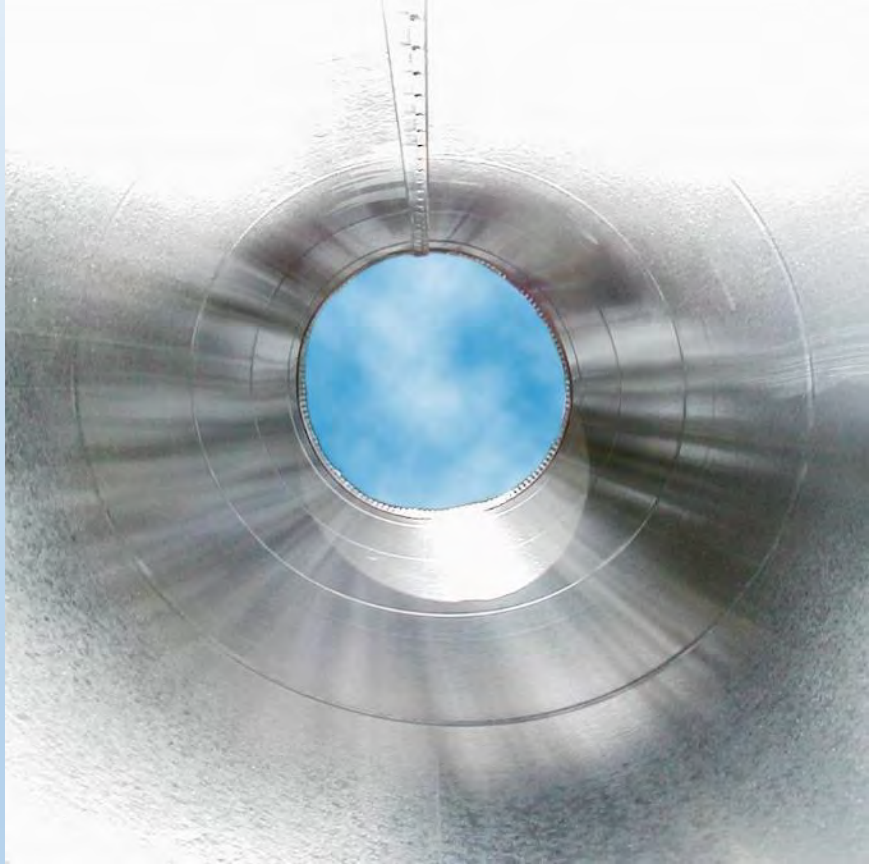
**Circa 2017
Bailes, Van Rite**



The
JOY OF FLEX



Sheet Metal Duct



Flex Duct



HVAC Duct Efficiency Study

2003

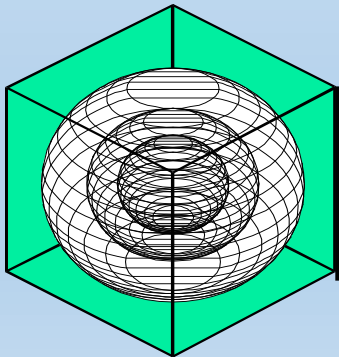
Air Distribution Institute

ASHRAE RP-1333

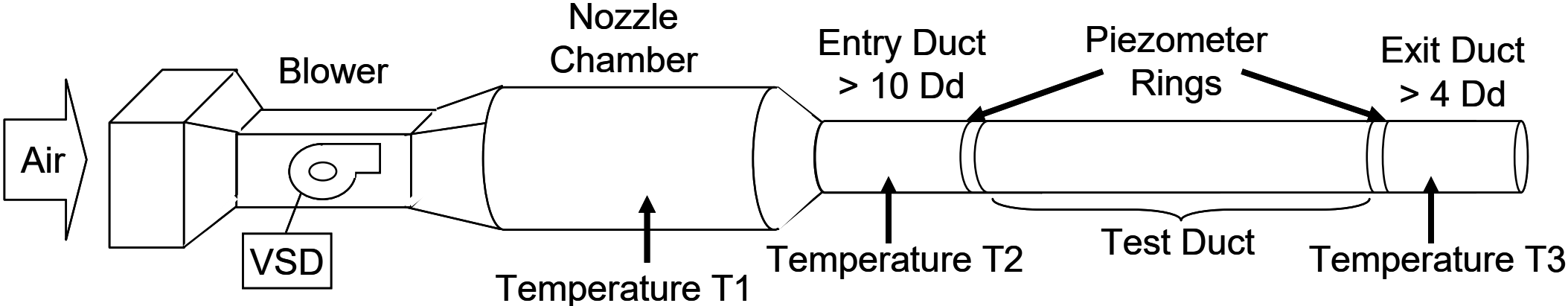
ONCOR - TXU Electric Delivery

Lennox Industries

Texas A&M University

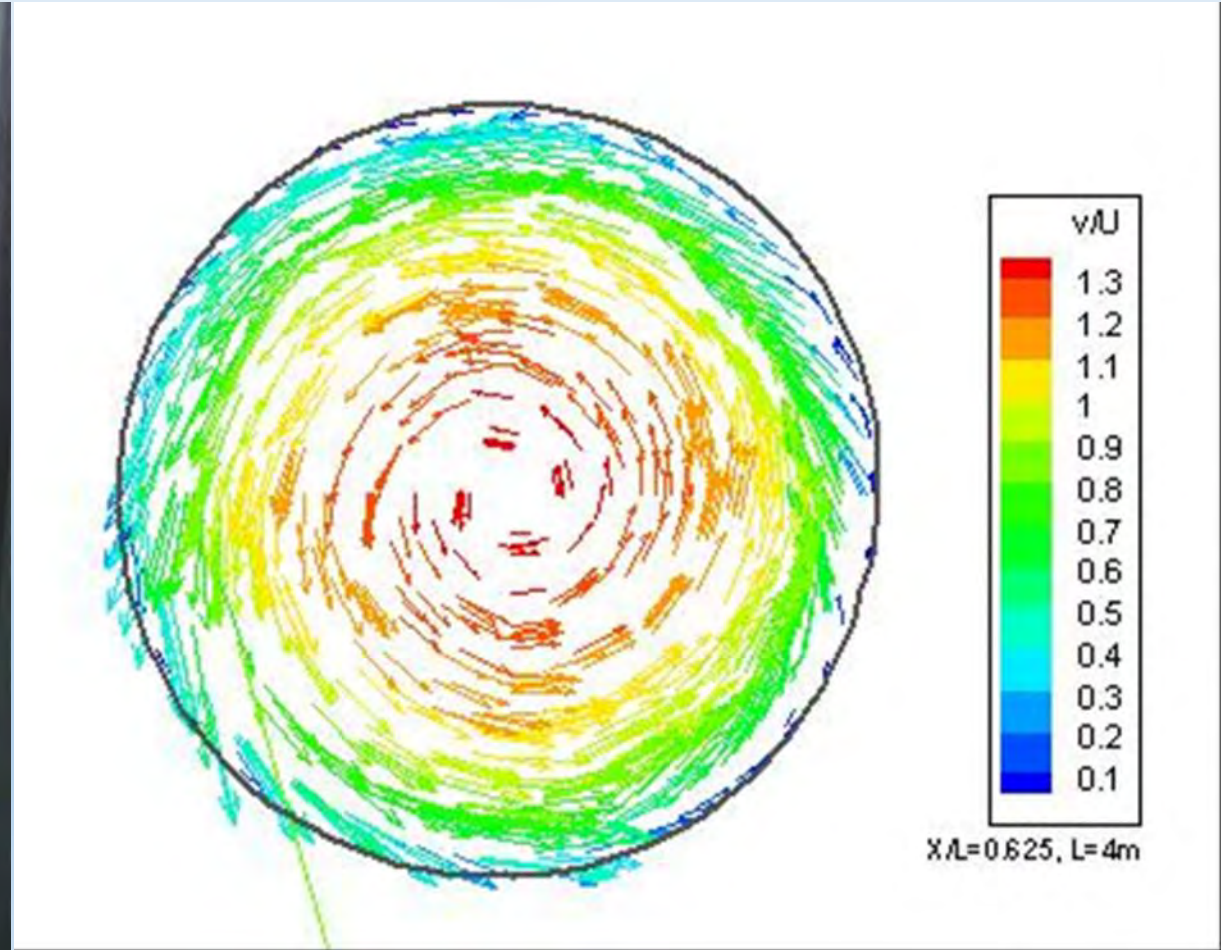


Data Acquisition (DAQ) setup



ASHRAE Standard 120

CFD Modelling



Darcy Equation

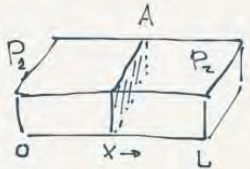
Darcy's Law

UNITS: cm sec d / cp atm SINGLE FLUID T = CONST

STEADY STATE C_f = CONST

$$\frac{q}{A} = -\frac{k}{\mu} \frac{dP}{dx}$$

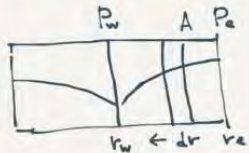
Linear



$$dP = -\frac{q}{A} \frac{\mu}{k} dx \quad \int_{P_1}^{P_2} dP = \int_0^L -\frac{q}{A} \frac{\mu}{k} dx$$

$$P_2 - P_1 = -\frac{q}{A} \frac{\mu}{k} L \quad q = \frac{k}{\mu} \frac{A}{L} (P_1 - P_2)$$

Radial



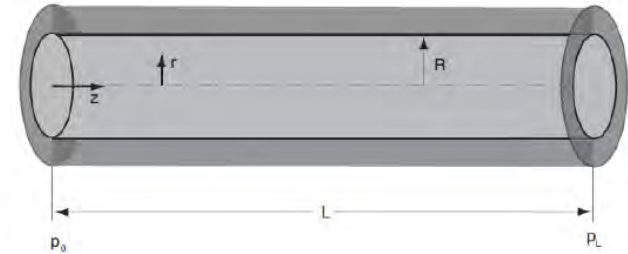
$$A = 2\pi r h \quad -\frac{q}{2\pi r h} = -\frac{k}{\mu} \frac{dP}{dr} \quad dP = \frac{q\mu}{2\pi k h} \frac{dr}{r}$$

$$\int_{P_w}^{P_e} dP = \int_{r_w}^{r_e} \frac{q\mu}{2\pi k h} \frac{dr}{r} \quad P_e - P_w = \frac{q\mu}{2\pi k h} \ln \frac{r_e}{r_w}$$

$$q = \frac{2\pi k h (P_e - P_w)}{\mu \ln \frac{r_e}{r_w}}$$

Colebrook Equation

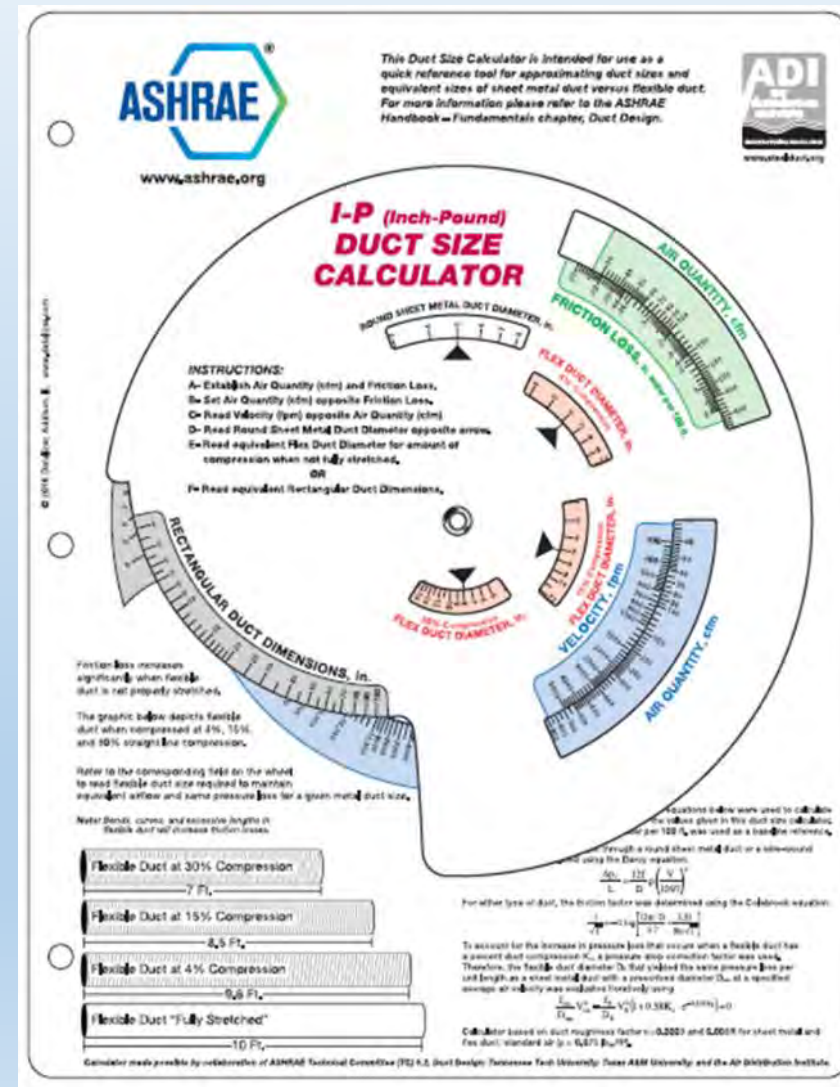
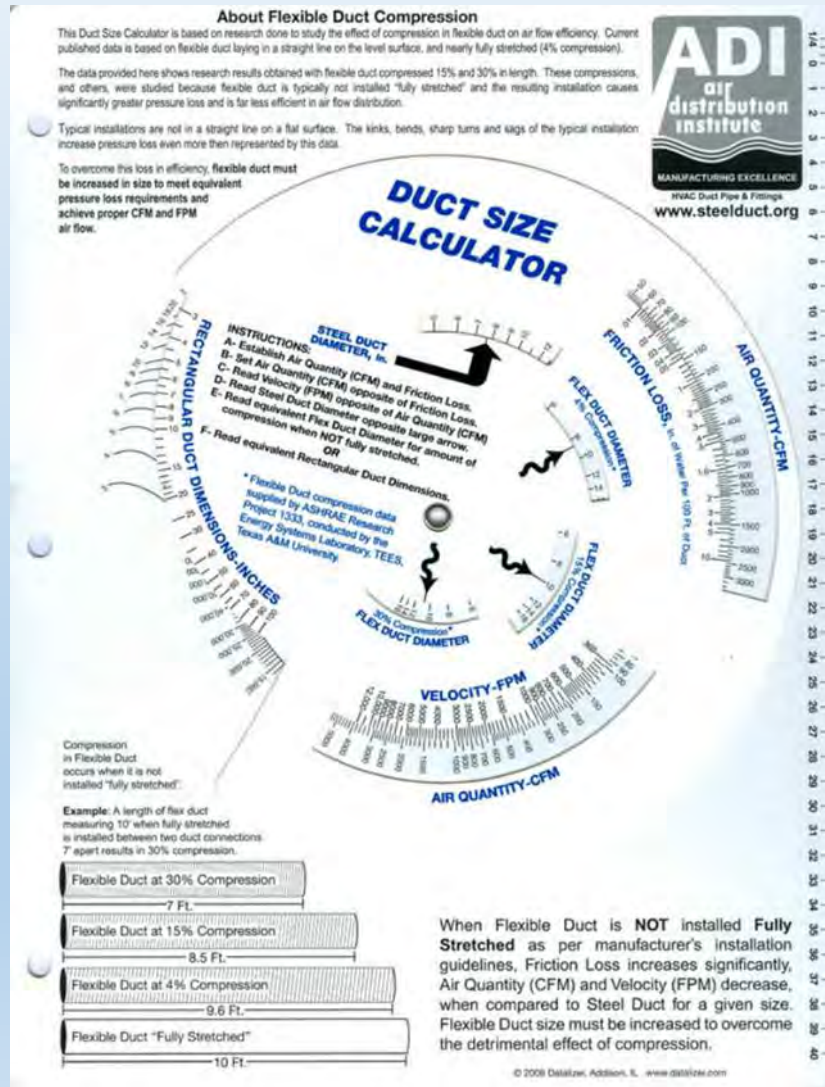
1. Tube flow is encountered in several material processes, such as in extrusion dies as well as sprue and runner systems, including those used in injection molds. Let's assume that the flow inside the tube is steady, fully developed, and is axis-symmetric. Furthermore, it has no entrance effects, the gravitational force is negligible, and the fluid is a Newtonian fluid. Based on the momentum equation in the z direction, simplify it and then solve for the velocity profile, $u_z(r)$ and the volumetric flow rate, Q . (10 points)



$$\rho \left(\frac{\partial u_z}{\partial t} + u_r \frac{\partial u_z}{\partial r} + \frac{u_\theta}{r} \frac{\partial u_z}{\partial \theta} + u_z \frac{\partial u_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial u_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u_z}{\partial \theta^2} + \frac{\partial^2 u_z}{\partial z^2} \right] + \rho g_z$$

2. As a mechanical or manufacturing engineer or as a professional and a member of the global community, please list a few ways that we can reduce the overall consumption of plastics for products, as well as a few ways that we can reduce, if not eliminate, the post-consumer plastic waste. Please be as creative, thoughtful, or bold as you can. We love to hear from your ideas.

ADI and ASHRAE Duct Size Calculators



Big Whoop

Darcy Equation

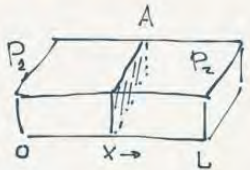
Darcy's Law

UNITS: cm sec d / cp atm SINGLE FLUID T = CONST

STEADY STATE C_f = CONST

$$\frac{q}{A} = -\frac{k}{\mu} \frac{dP}{dx}$$

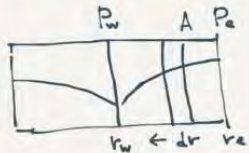
Linear



$$dP = -\frac{q}{A} \frac{\mu}{k} dx \quad \int_{P_1}^{P_2} dP = \int_0^L -\frac{q}{A} \frac{\mu}{k} dx$$

$$P_2 - P_1 = -\frac{q}{A} \frac{\mu}{k} L \quad q = \frac{k}{\mu} \frac{A}{L} (P_1 - P_2)$$

Radial



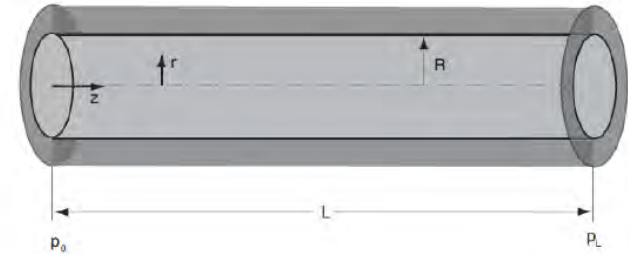
$$A = 2\pi r h \quad -\frac{q}{2\pi r h} = -\frac{k}{\mu} \frac{dP}{dr} \quad dP = \frac{q\mu}{2\pi k h} \frac{dr}{r}$$

$$\int_{P_w}^{P_e} dP = \int_{r_w}^{r_e} \frac{q\mu}{2\pi k h} \frac{dr}{r} \quad P_e - P_w = \frac{q\mu}{2\pi k h} \ln \frac{r_e}{r_w}$$

$$q = \frac{2\pi k h (P_e - P_w)}{\mu \ln \frac{r_e}{r_w}}$$

Colebrook Equation

1. Tube flow is encountered in several material processes, such as in extrusion dies as well as sprue and runner systems, including those used in injection molds. Let's assume that the flow inside the tube is steady, fully developed, and is axis-symmetric. Furthermore, it has no entrance effects, the gravitational force is negligible, and the fluid is a Newtonian fluid. Based on the momentum equation in the z direction, simplify it and then solve for the velocity profile, $u_z(r)$ and the volumetric flow rate, Q . (10 points)



$$\rho \left(\frac{\partial u_z}{\partial t} + u_r \frac{\partial u_z}{\partial r} + \frac{u_\theta}{r} \frac{\partial u_z}{\partial \theta} + u_z \frac{\partial u_z}{\partial z} \right) = -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial u_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 u_z}{\partial \theta^2} + \frac{\partial^2 u_z}{\partial z^2} \right] + \rho g_z$$

2. As a mechanical or manufacturing engineer or as a professional and a member of the global community, please list a few ways that we can reduce the overall consumption of plastics for products, as well as a few ways that we can reduce, if not eliminate, the post-consumer plastic waste. Please be as creative, thoughtful, or bold as you can. We love to hear from your ideas.

Andy Ask





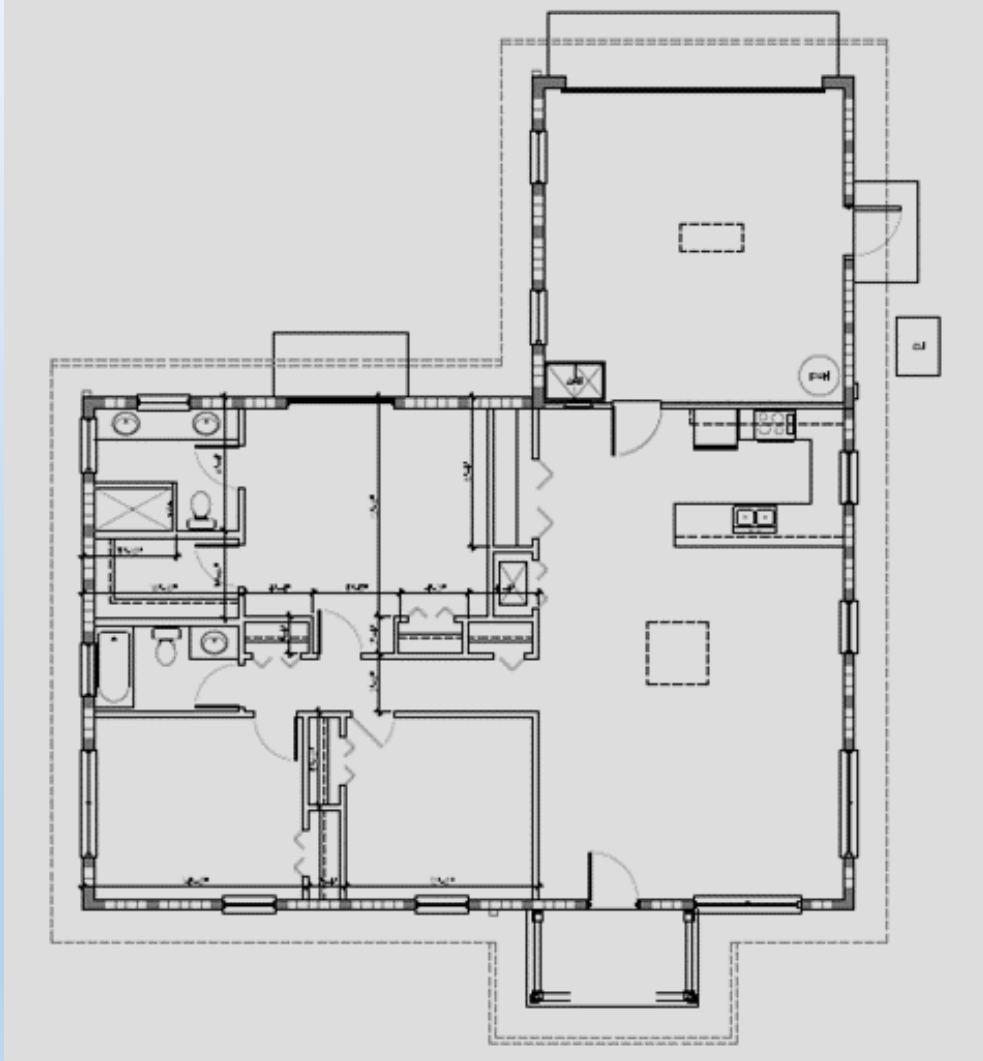
FLORIDA SOLAR ENERGY CENTER

FLEXIBLE RESIDENTIAL TEST STRUCTURE

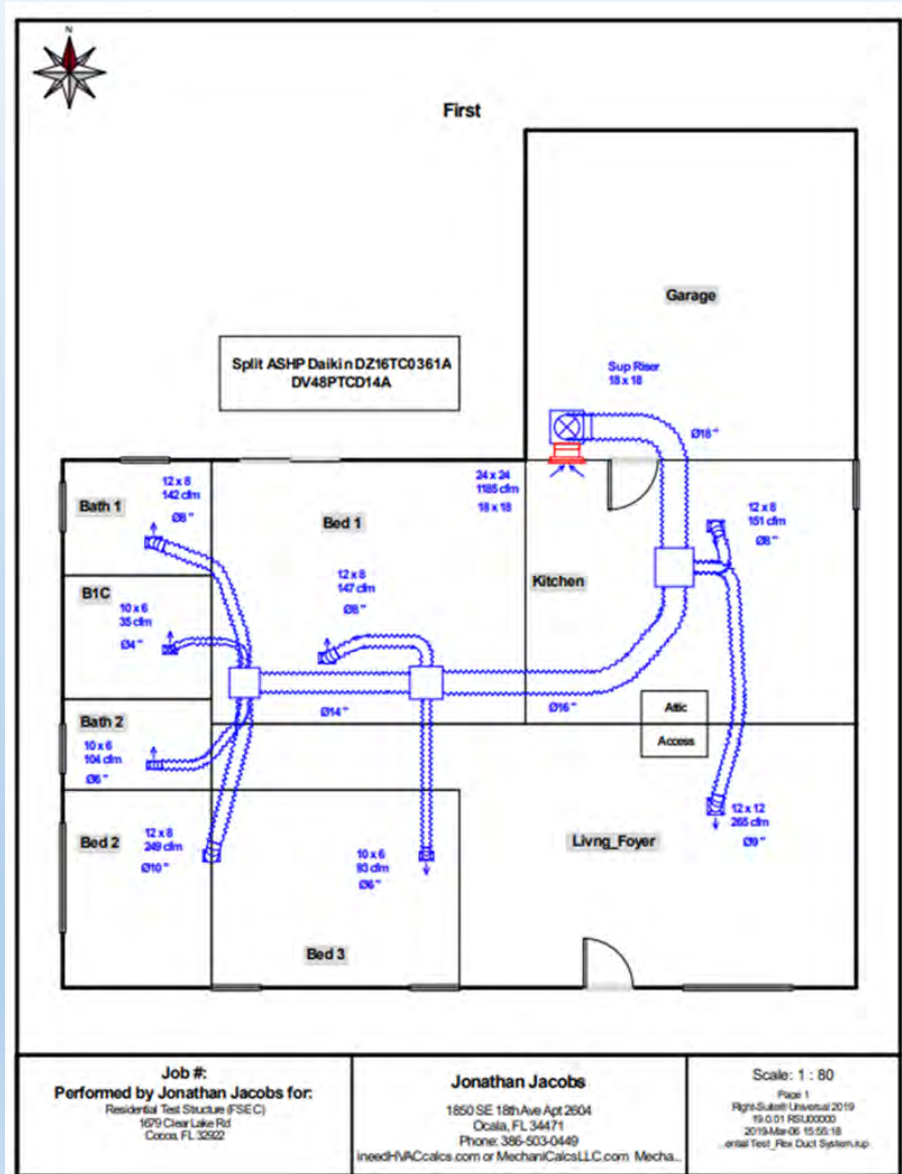




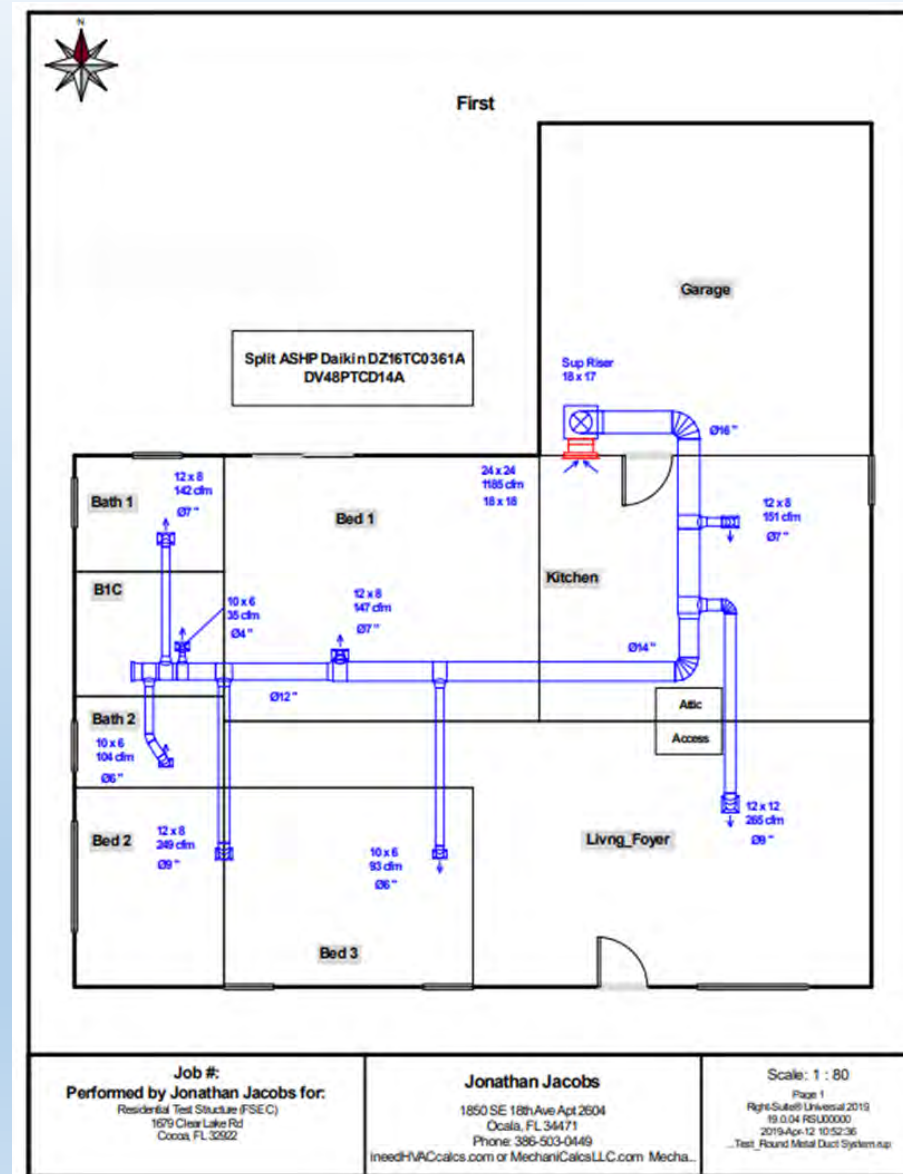
Single Zone



Flex Duct House



Metal Duct House



Sheet Metal Duct



Flexible Duct

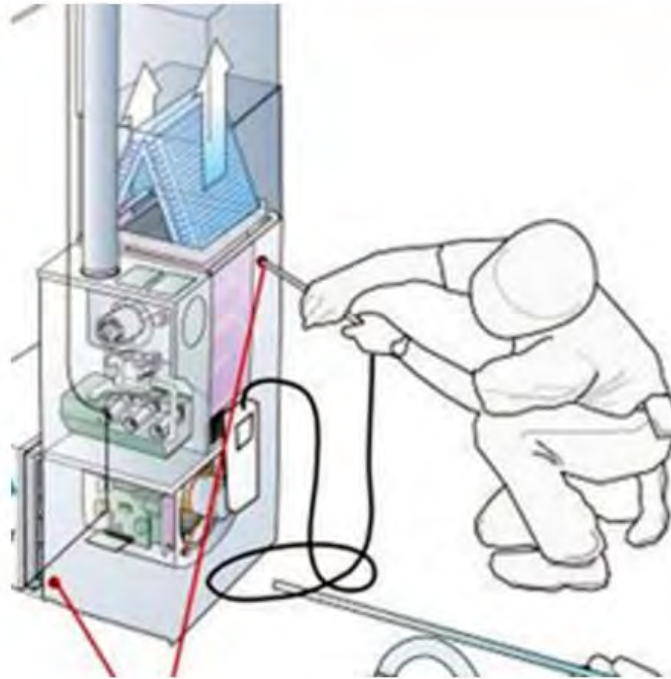


**5% LESS
OPERATING
COST**

TEST 1

- **BEST METAL** .34" wc TESP, High Fan Speed vs
- **BEST FLEX** .44" wc TESP, High Fan Speed
 - **SHEET METAL DUCT HOUSE USED 5% LESS HVAC ENERGY**
 - **75° Inside, 80° Outside**

Contracting Business – February 10, 2016



SERVICE

Measure Static Pressure in Six Simple Steps

Feb. 10, 2016

It typically should take less than five minutes to measure a residential system's static pressure.

Rob 'Doc' Falke

2018 National Comfort Institute Total External Static Pressure Study

- 500 Homes across USA**
- Flex duct and metal duct**
- Average TESP was .82" wc**

**14% LESS
OPERATING
COST**

TEST 2

- **BEST METAL** .34" wc TESP, High Fan Speed vs
- **BAD FLEX** 1.0" wc TESP, High Fan Speed
 - **SHEET METAL DUCT HOUSE USED 14% LESS HVAC ENERGY**
 - **75° Inside, 80° Outside**

How do you screw up a good flex duct system ?

- Add more than minimum length required
- Add more turns and restrictions
- Create more sag and liner compression

Test 3 - Added Flex Duct to Produce .82" TESP

Test 3					
T & B	Flex Diameter	Test 1 length	Add for Test 3	Notes	Total Length
			Trunk + 30%		
T-1	18"	219"	66"	measured total at install	285"
T-2	16"	300"	90"	used full 25 ft length	390"
T-3	14"	124"	37"		161"
		643" / 53.6 ft	193" / 16 ft		836" / 69.7 ft
			Branch + 50%		
B-1	8"	173"	87"		260"
B-2	9"	221"	111"		332"
B-3	6"	131"	67"		198"
B-4	8"	121"	61"		182"
B-5	10"	164"	82"		246"
B-6	4"	80"	40"		120"
B-7	8"	192"	96"		288"
B-8	6"	159"	80"		239"
		1,241" / 103.4 ft	624" / 52 ft		1,865" / 155.4 ft
		<u>157 total feet</u>	<u>68 total feet</u>		225.1 total feet
				<u>Adding 68 feet of duct total (43% more)</u>	

**9% LESS
OPERATING
COST**

TEST 3

- **BEST METAL** .34" wc TESP, High Fan Speed vs
- **AVERAGE FLEX** .82" wc TESP, High Fan Speed
 - **SHEET METAL DUCT HOUSE USED 9% LESS HVAC ENERGY**
 - **ANUALIZED**

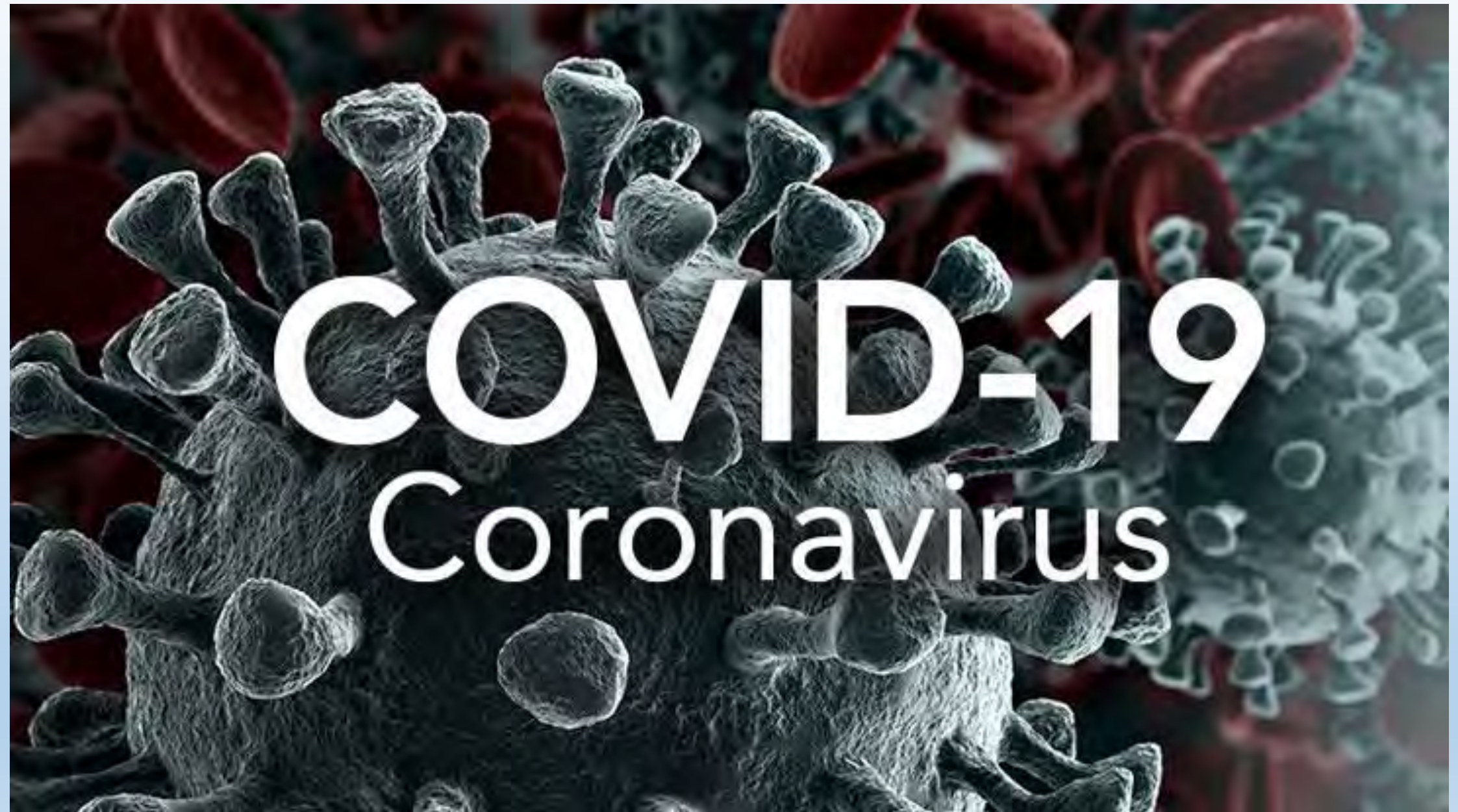
Peak Cooling Power Use During Hot Summer Period 4:15 pm – 5:45 pm

	Metal AHU	Metal Condenser	Flex AHU	Flex Condenser	Metal Total	Flex Total
Power (kW)	0.265	2.335	0.399	2.445	2.600	2.843
Delta flex- metal (kW)			0.134	0.110		0.244
Delta flex- metal (%)			50.5%	4.7%		9.4%

Big Whoop

Does Anyone Care ?





COVID-19

Coronavirus

“Don’t Do Stupid Stuff”



R-6 Duct Wrap Installed Thickness = 1 5/8"

Product Data Sheet



Availability and Installed R-Values

Standard roll width: 48" (1.2m), 60" (1.5m)

Installed R (RSI) values: When installed in accordance with recommended installation procedures, SOFTR® Duct Wrap FRK will provide installed R (RSI) values as follows:

Nominal Thickness		Out-of-Package R (RSI) Value ¹		Installed Thickness ²		Installed R (RSI) Value ^{1,2}	
in.	mm			in.	mm		
Type 75 – 0.75 pcf (12 kg/m ³)							
1½	(38)	5.1	(0.90)	1½	(29)	4.2	(0.74)
2.2	(56)	7.4	(1.30)	1¾	(42)	6.0	(1.06)
3	(76)	10.0	(1.76)	2¼	(57)	8.3	(1.46)

Turns Out, This Stuff Really Matters

What's the worst thing that can happen ?



**17% LESS
OPERATING
COST**

CALCULATED

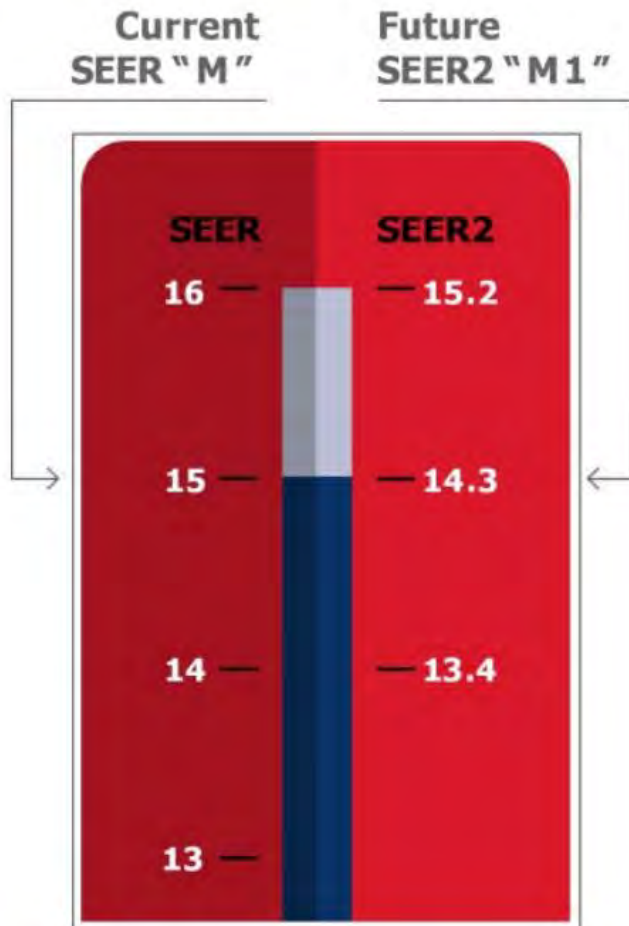
- **BEST METAL** .34" wc TESP, High Fan Speed vs
- **AVERAGE FLEX** .82" wc TESP, High Fan Speed
- **SHEET METAL DUCT HOUSE MAY HAVE USED 17% LESS HVAC ENERGY IF THE DUCTS WERE INSULATED WITH R-6 INSTEAD OF R2.5 (RESULT OF OVER-COMPRESSED WRAP)**

Conclusions

- Higher static pressure across the constant flow ECM fan resulted in higher cooling power and energy use.
- Test 1 Good Practice for metal and flex ducts
 - Flex duct system 13.4% (5.4%+8%) greater than metal duct.
- Test 2 Poor Practice
 - Flex duct system 21.8% (13.8%+8%) greater than metal duct.
- Test 3 Likely Practice
 - Flex duct system 16.8% (8.8%+8%) greater than metal duct.

SEER2

New Measurement



New Testing Procedure



Questions ?

Thank you